

VIBRATION ANALYSIS OF 3-D PIPING
VIA TRANSFER MATRICES

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THESIS

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by

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ABSTRACT

This thesis develops the theory for and presents a digital computer program capable of determining the natural frequencies of a three-dimensional piping system having arbitrary configuration. The analysis uses the method of transfer matrices. Piping hangers, loops, and complex branches (branches emanating from branches) have not been included in the analysis. A distributed mass model is used for straight pipe sections, while mass is lumped for curved sections. Inclusion of shear deflection and rotary inertia is optional.

Several piping configurations are analyzed using the program; the results are compared with analytical solutions or values from the literature to demonstrate the accuracy and integrity of the program.

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I. INTRODUCTION

A. THE PROBLEM

From an engineering standpoint it is important to be able to analyze and predict the vibration characteristics of structural elements and assemblies. The mathematical aspects of several approaches to such analysis have been known for a long time, but the volume of arithmetic operations dictated a gross simplification of complicated systems in order to reduce the arithmetic to manageable limits. The development of the electronic digital computer within the last two decades, however, has provided a means for the practical and accurate analysis of quite complicated vibrating systems without great simplifications which cast doubt on the integrity of results. In turn, this has provided a stimulus for further developments in theory which take advantage of computer capabilities.

Most methods of analysis of actual systems in which both mass and elastic compliance are continuously distributed, involve a discretization of one sort or another; usually the total mass of the system is considered to be lumped at a finite number of distinct points, the masses being connected by massless spring elements. It is almost universally assumed that the elastic characteristics of the system are linear and that the excursions of points in the system from their equilibrium positions are sufficiently small that no

geometric nonlinearities are involved. Such an idealization leads to n second order ordinary differential equations. Provided vibrations are assumed to occur isochronously and no damping is present, these equations reduce to a system of n algebraic equations in the n amplitudes and frequency squared.

The most convenient and expedient way to cast this problem is in matrix form. Thus engineers using these methods deal with such quantities as the mass-matrix, the stiffness matrix, the modal matrix, etc.

On the other hand, for certain simple cases of distributed mass and elasticity, it is not necessary to lump the mass into a finite number of concentrated masses. Instead the vibration problem can be formulated as a partial differential equation which can be solved directly. Recently, this point of view has been extended for the analysis of more complicated cases than can be treated by the conventional use of partial differential equations. Namely, the system being analyzed is divided into a number of sub-systems, a satisfactory analysis existing for each one; then, by a procedure which will be described in greater detail later, these sub-solutions are combined to obtain the solution applicable to the original, complicated configuration. This procedure, known as the transfer matrix method, evolved from a method described by H. Holzer fifty years ago for the analysis of torsional vibrations. Applications of Holzer's viewpoint to the analysis

of flexural systems was made independently by M. A. Prohl and N. O. Myklestad about thirty years ago. The essentials of these methods were abstracted by a number of engineers and mathematicians who developed and formalized them into what is now known as the transfer matrix method. The names of S. Falk, K. Marguerre, E. C. Pestel, and F. A. Leckie, among others, are associated with this development. A book by Pestel and Leckie [Ref. 9] represents the fullest and most accessible treatment of the subject available today.

The transfer matrix method is most naturally applicable to the analysis of "chain-like" structures composed of elements which are adjoined at distinct points and for each of which a satisfactory vibration analysis exists. The simplest application is to the case where such a chain of elements has but two ends. However, it can be applied to structures having slightly greater topological complexity, but it simply is not appropriate for the analysis of structures which are highly reticulated or in which major elements are joined along lines or surfaces rather than at distinct points.

Piping systems are substantially chain-like and topologically simple. Accordingly, it was natural that the transfer matrix method be applied to the analysis of vibration in piping. In this country the first such effort which was reported was that done by G. E. Fink in his Naval Postgraduate School thesis of 1964. At about the same time similar efforts were being made in Japan under the sponsorship of

the Japan Society of Mechanical Engineers. It is known that the Japanese have produced useful digital computer programs for the analysis of piping using the transfer matrix method, but it has not been possible to learn any of the details since these programs remain under the proprietary control of the Japanese government.

Fink developed a program which has convenient generality and gives good results, but which is limited to systems lying entirely in a single plane. His program is capable of making separate determinations of "in-plane" and "out-of-plane" vibrations, which, for the case of uniplanar configurations, are uncoupled. Further work on the theory was done by W. S. Baird, Jr. and J. L. Simmons in their Naval Postgraduate School theses; this work mainly related to systems having appreciable topological complexity. In his Naval Postgraduate School thesis, Y. S. Kim returned to the development of Fink's program, adding several useful features, including the incorporation of several alternative mathematical techniques for accurate solution. Some of these techniques reduced solution time below that required in Fink's original program; however, Kim's program is still limited to uniplanar systems.

Accordingly, the writer undertook to develop a program capable of treating truly three-dimensional configurations. The remainder of this thesis presents the analytical background of this development, lists the program which includes

rules for its operation, and gives evidence concerning the high accuracy of the results.

It should be made clear that no claims are made concerning the relative usefulness of the transfer matrix method as compared to lumped-mass methods which are presently in widespread use for analysis of piping vibrations. In general, the writer believes that the results which are obtained by the transfer matrix procedure are substantially more accurate than can be obtained in most cases using lumped-mass analysis. However, the delicacy of the calculations involved in the transfer matrix method seems to imply that only a limited number of the lowest frequencies (and their associated modes) can be obtained using present programming methods and computer hardware. Greater word-length in the computer, or availability of fast multiple precision capability, would permit obtaining a larger number of frequencies and modes via the transfer matrix method. Briefly, the motivation for the development reported herein has been simply that the transfer matrix method is clearly appropriate for the analysis of piping vibrations and that the application should be made. It seems likely that in some cases and for some purposes the transfer matrix method may be more convenient and economical while in other cases lumped-mass methods may be preferable.

B. THE TRANSFER MATRIX METHOD

Specific details of the application of transfer matrix methodology to the analysis of piping vibrations will be given in subsequent parts of this thesis. Background material and a general exposition of the method is most readily available in the book by Pestel and Leckie [Ref. 9] and it would be pointless to include herein a paraphrase of such material. However, for the reader who is not familiar with the ideas behind the transfer matrix method, a very brief exposition is given in this section.

An appropriate "state vector" is defined which presents force-type and deflection-type information capable of specifying the "state" at each of several points in the configuration at which sub-elements are joined together. These quantities specify the configuration and internal force system in the structure at its extreme isochronous, deflected position. For example, in a simple torsional system, there are only two quantities in the state vector, the angle of rotation from the equilibrium position and the torque transmitted from one element to the next. The elastic compliance and the inertial characteristics of an element permit obtaining equations relating the quantities in the state vector at the left end of the element to those in the state vector at the right end of the element. When these relations are put in matrix form it is found that the state vector at the right can be written as the product of the state vector at the left premultiplied by a square

matrix, called a transfer matrix, the elements of which represent the elastic and inertial properties of the element.

In applying the method, a state vector is chosen for the left end of the assembly of elements in such a way as to satisfy conditions of restraint at the left. Then, by successive multiplication by transfer matrices representing individual elements encountered in order, progressing from the left end of the assembly toward the right, one arrives at a representation for the state vector at the right end. The individual transfer matrices are functions of isochronous frequency; accordingly the right end matrix is a function of frequency. When frequency is adjusted, any particular frequency which permits satisfaction of restraint at the right end is a natural frequency of the system. The several state vectors at the junction points give the deflected configuration, or mode shape, corresponding to this frequency, when this value is substituted in the expressions for the state vectors.

For simple cases it is possible to carry out the multiplications in literal form, keeping the frequency as an unknown parameter. Then the right end restraint conditions provide a polynomial equation in the frequency which can be solved by customary methods. However, in relatively complicated cases it is not feasible to retain the frequency as a literal parameter. In these cases a definite numerical value is assumed for frequency and this leads to a numerical measure of the failure to satisfy right end restraint

conditions. By assuming different values of the frequency and by constructing a curve of measure-of-failure versus frequency, the correct frequencies can be obtained.

The transfer matrices which describe the individual elements may be a theoretically complete and full representation, or they may themselves be approximate representations. In the application to piping systems it is convenient to represent a straight length of pipe by a transfer matrix which is theoretically complete and exact. For systems composed of only straight lengths of pipe there is no approximation whatsoever in the transfer matrix method (other than that implied by internal round-off in the computer); this may be compared with the fact that all lumped-mass procedures definitely imply an approximation and thus result in some theoretical error.

Curved pipe elements (elbows and bends) could be treated exactly since adequate theory exists for the construction of theoretically exact transfer matrices with which to represent them. However, there are severe computational difficulties involved with such exact representation. Accordingly, with the awareness that the total length of curved pipe is but a small fraction of the total length of both straight and curved pipe in the average configuration, it was decided to represent curved pipe by a succession of massless curved sections having proper elastic properties, alternating with appropriately located point masses. For each of these idealized elements, it is

possible to obtain the corresponding transfer matrices with no computational difficulties. Thus, elbows and bends are represented in much the same way that is done with the lumped-mass methods mentioned previously. It is believed that the approximation introduced by this point-mass representation of a small part of a configuration is quite tolerable for practical engineering purposes. However, it is certainly possible to modify the program reported herein so as to provide for exact representation for curved elements. This would not involve major changes but only the development of a subroutine to replace those given herein for curved elements.

II. PROGRAM DEVELOPMENT

A. THE STATE VECTOR, TRANSFER MATRIX, AND FREQUENCY DETERMINANT

1. State Vector

The state vector z_i at point i of an elastic system is a column vector whose components are the generalized displacements, D_i , and the generalized forces, F_i , at that point. In matrix notation, we have

$$z_i = \begin{Bmatrix} D \\ F \end{Bmatrix}_i$$

For a beam free to move in three dimensions with longitudinal, torsional and flexural elasticity, the state vector takes the form

$$z_i = \begin{Bmatrix} u & - \text{deflection in x-direction} \\ v & - \text{deflection in y-direction} \\ w & - \text{deflection in z-direction} \\ \phi & - \text{rotation about x-axis} \\ \psi & - \text{rotation about y-axis} \\ \theta & - \text{rotation about z-axis} \\ N & - \text{axial force in x-direction} \\ V_y & - \text{shear force in y-direction} \\ V_z & - \text{shear force in z-direction} \\ T & - \text{Torque about x-axis} \\ M_y & - \text{Moment about y-axis} \\ M_z & - \text{Moment about z-axis} \end{Bmatrix}$$

The order in which the individual displacements and forces occur in the state vector is of no particular importance so long as consistency is maintained. Program VIBREL

(for VIBRation of ELastic piping), the program developed herein, employs the arrangement given above.

2. Sign Convention and Coordinate System

A right-handed cartesian coordinate system is used to describe the local generalized forces and displacements, the x-axis coinciding with the centroidal axis of the pipe as shown in Fig. 2.1.

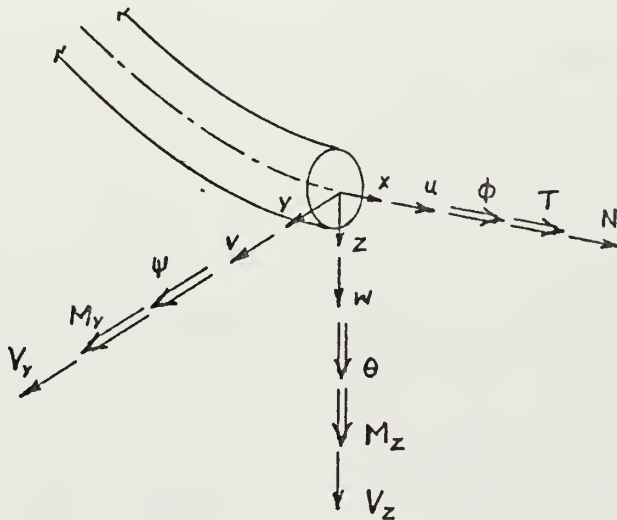


Figure 2.1: Vectorial Representation of State Vector Displacement and Forces

A cut made at any location of the pipe will expose two faces; the face with its outward normal in the positive x-axis direction is considered positive. The state vector parameters are positive if their vector representations coincide with the directions prescribed by the cartesian coordinate system at the positive face as in Fig. 2-1. Rotation and moment vectors are portrayed according to the right-hand-screw rule by a double line with an arrowhead.

3. Transfer Matrices

The transfer matrix is defined as the matrix which relates two state vectors at successive selected points of

division in the elastic system.

$$z_i = U_i z_{i-1}$$

where U_i is the transfer matrix. The transfer matrix must be square, and in the case of a spatial system must have 12 rows and 12 columns. When the transfer matrix relates the flexibility between the points i and $i-1$ of a continuous system it will hereafter be referred to as a field matrix, V . When a discontinuity arises in either force or displacement, as with a point mass in a lumped system, the transfer matrix relating the state vectors on either side of the discontinuity is called a point matrix, P . The term "transfer matrix" will henceforth be used with regard to the matrix, U , which includes mass and flexibility in the relationship between the state vectors at two distinct points. A complete discussion of transfer matrices, with examples and derivations, can be found in Ref. 9.

4. The Elimination Process and Frequency Determinant

The process of combining the system transfer matrices to find natural frequencies can be demonstrated by a simple planar example using the Myklestad method of replacing a distributed mass beam with lumped masses connected by flexible elements (Fig. 2.2). Assuming that the field and point

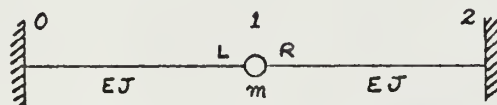


Figure 2.2: Lumped Mass Representation of a Planar Beam

matrices for a massless flexible beam and a point mass are already known, then

$$z_1^L = V_1 z_0, \quad z_1^R = P z_1^L, \quad \text{and} \quad z_2 = V_2 z_1^R$$

where $z_i = \begin{Bmatrix} w \\ \psi \\ V_z \\ M_y \end{Bmatrix}_i$ and superscripts L and R indicate left side

or right side, respectively. Combining the above yields the relationship

$$z_2 = V_2 P V_1 z_0 = U z_0 \quad (2-1)$$

where U is the system transfer matrix. Written in full this relationship is

$$\begin{Bmatrix} w \\ \psi \\ V_z \\ M_z \end{Bmatrix}_2 \begin{bmatrix} a_{11} & a_{12} & a_{13} & a_{14} \\ a_{21} & a_{22} & a_{23} & a_{24} \\ a_{31} & a_{32} & a_{33} & a_{34} \\ a_{41} & a_{42} & a_{43} & a_{44} \end{bmatrix} \begin{Bmatrix} w \\ \psi \\ V_z \\ M_z \end{Bmatrix}_0$$

the a_{ij} being vibration functions of mass, stiffness, and frequency. Application of the boundary conditions $w_0 = w_2 = \psi_0 = \psi_2 = 0$ reduces the matrix equation to the following

$$\begin{bmatrix} a_{13} & a_{14} \\ a_{23} & a_{24} \end{bmatrix} \begin{Bmatrix} V_z \\ M_z \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad (2-2)$$

Because the elements of this square matrix are functions of frequency, its determinant is called the frequency determinant of the system. Note that this 2 X 2 matrix is a

submatrix of the 4 X 4 matrix appearing above. A non-trivial solution of the equations 2-2 requires that the value of the frequency determinant (Δ) equal zero. A value of the frequency, ω , which causes Δ to vanish is called a natural frequency of the system.

The natural frequencies could be solved for directly provided the expressions for the a_{ij} were kept in terms of the circular frequency, ω , but in practice these expressions are forms much too unwieldy to be handled explicitly. Program VIBREL evaluates the frequency determinant of order 6 X 6 for successive numerical values of frequency, zero values of the frequency determinant corresponding to natural frequencies of the system.

Let the state vector be composed of $2r$ quantities. Then to compute U in Eq. 2-1, two multiplications of $2r \times 2r$ matrices are required. Noticing that application of the boundary conditions to the state vector at point 0 will cause r of its elements to be zero, we find that r columns of U are multiplied by zero, and thus play no part in the calculation. Kim [Ref. 6] introduced the concept of the "state matrix," S , such that the left end state vector, z_0 , of $2r$ quantities can be written in terms of a compressed state vector, z_0^* , having only r quantities, viz.,

$$z_0 = Sz_0^*$$

where S is the state matrix at the left end, having r columns and $2r$ rows, its elements consisting only of zeros

and ones. All elements of the state matrix are zero except that in each column, unity appears in the same row position as the corresponding non-zero term in the state vector, boundary conditions having been applied. If the state matrix is used in the multiplication scheme, then we can multiply $2r \times r$ matrices by $2r \times 2r$ matrices and reduce the calculation considerably. The resulting $2r \times r$ system transfer matrix requires only the application of right end boundary conditions to yield the frequency condition.

To illustrate this state matrix concept most simply, consider again a two-dimensional case in which the state vector consists of four quantities. The three dimensional case is an obvious generalization of the following discussion. Taking strictly the state vector approach we can

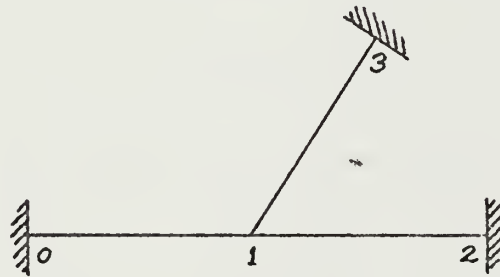


Figure 2.3: Single Branch Planar System

express the relation between the state vector at 2 and that at 0 by

$$z_2 = U_2 P_1 U_1 z_0 \quad (2-3)$$

where U_2 and U_1 are straight section transfer matrices and the point matrix P_1 accounts for the effects of the branch. Carrying out the matrix multiplication and exhibiting the

system transfer matrix and state vectors which consider bending only, we have

$$\begin{Bmatrix} w \\ \psi \\ V_z \\ M_z \end{Bmatrix}_2 = \begin{bmatrix} a_{11} & a_{12} & a_{13} & a_{14} \\ a_{21} & a_{22} & a_{23} & a_{24} \\ a_{31} & a_{32} & a_{33} & a_{34} \\ a_{41} & a_{42} & a_{43} & a_{44} \end{bmatrix} \begin{Bmatrix} 0 \\ 0 \\ A_1 \\ A_2 \end{Bmatrix} \quad (2-4)$$

The boundary conditions have been applied to the left end state vector; the quantities A_1 and A_2 represent the unknown quantities of V_z and M_z respectively. Application of the right end boundary conditions yields a frequency condition of the same form as Eq. 2-2.

Using the state matrix approach instead, Eq. 2-3 becomes

$$z_2 = U_2 P_1 U_1 S z_o^* \quad (2-5)$$

where S is the state matrix $\begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 1 & 0 \\ 0 & 1 \end{bmatrix}$. z_o^* contains the non-

zero quantities of the left end state vector. After multiplication of the matrices of Eq. 2-5, the number of individual multiplications being considerably less than before, the following relation is obtained:

$$\begin{Bmatrix} w \\ \psi \\ V_z \\ M_z \end{Bmatrix}_2 = \begin{bmatrix} a_{13} & a_{14} \\ a_{23} & a_{24} \\ a_{33} & a_{34} \\ a_{43} & a_{44} \end{bmatrix} \begin{Bmatrix} A_1 \\ A_2 \end{Bmatrix} \quad (2-6)$$

We see from this result that inclusion of the state matrix compresses the left end state vector to r quantities while providing the same useful information as the matrix of Eq. 2-4, inasmuch as the same frequency condition is obtained after the right end boundary conditions are applied. Also, if the relative magnitudes of the unknowns A_1 and A_2 have been determined, the mode deflections and forces at a point can be calculated simply by multiplication of the compressed left end state vector, consisting of r quantities, by the $2r \times r$ system transfer matrix up to that point. The appropriate mode frequency is used in the computation. This will be discussed in greater detail in Chapter 3.

B. 3-D PIPING GEOMETRY

1. Working Points and Piping Nomenclature

The geometry of the piping system as it is used in program VIBREL is based on working points. A working point of the system occurs where cross section or properties change, where branches join the main member, where the piping changes direction, and at the ends of the main member and branches. These points are measured in cartesian coordinates from an origin located arbitrarily but fixed in space. For purposes of this discussion we shall distinguish between the two types of working points at which piping direction changes. At a corner point (Fig. 2.4) the piping direction changes with no discernible radius of curvature. At a bend point (Fig. 2.5), which is defined as the intersection

of the tangents to the two ends of a curved pipe, the radius of curvature is finite.

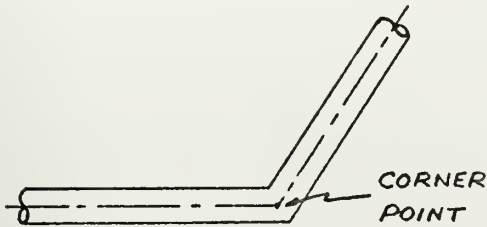


Figure 2.4: Piping Corner

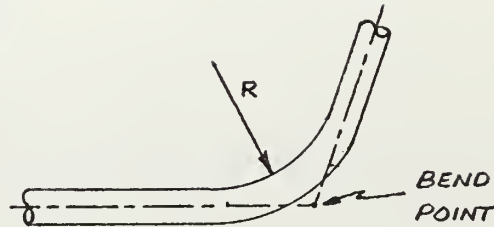


Figure 2.5: Piping Bend

The main member is designated as the longest continuous run of pipe from which all branches emanate. Program VIBREL cannot handle branches emanating from branches.

2. Unit Vectors and Coordinate Transformations

If a set of mutually orthogonal x , y , and z unit vectors are formed at the main member and branch starting points, and prior to and following every directional change, we can determine, by angular differences in successive sets of unit vectors, the appropriate coordinate transformation angles. The local coordinate systems described by these unit vectors are formed with the piping system in its quiescent configuration.

Let us take, for example, the two sections of pipe pictured in Fig. 2.6 determined by the three working points E , F , G , where F is a corner point. Also let the two sections of pipe, EF and FG , be represented by vectors \vec{A} and \vec{B} , the

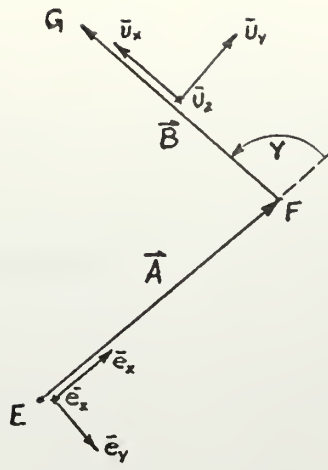


Figure 2.6: Vector Representation of Directional Change

components of which have been determined from the coordinates of E, F, and G. Then the angle γ can be determined by the following relation:

$$|\gamma| = \cos^{-1} \left[\frac{\bar{\mathbf{B}} \cdot \bar{\mathbf{A}}}{|\bar{\mathbf{B}}| |\bar{\mathbf{A}}|} \right]$$

The sign will be determined later. Since the vectorial representation of state vector quantities in Fig. 2.1 requires that the x-axis coincide with the centroidal axis of the pipe, let

$$\bar{\mathbf{e}}_x = \frac{\bar{\mathbf{A}}}{|\bar{\mathbf{A}}|}$$

For the purposes of succeeding derivations we desire that $\bar{\mathbf{e}}_y$ be in the plane of $\bar{\mathbf{A}}$ and $\bar{\mathbf{B}}$ and point outward from the directional change. Hence

$$\bar{\mathbf{e}}_z = \frac{\bar{\mathbf{B}} \times \bar{\mathbf{A}}}{|\bar{\mathbf{B}} \times \bar{\mathbf{A}}|} \quad \text{and} \quad \bar{\mathbf{e}}_y = \bar{\mathbf{e}}_z \times \bar{\mathbf{e}}_x$$

By the right hand screw convention the rotation of amount γ will always be in the negative $\bar{\mathbf{e}}_z$ sense. The set of unit

vectors, \bar{U}_x , \bar{U}_y , \bar{U}_z are found simply by rotating the initial unit vectors about the local z-axis by the angle γ .

Now consider the difference in orientation between the \bar{U}_x , \bar{U}_y , \bar{U}_z vectors and those prior to the next directional change farther down the pipe, $\bar{U}_{x'}$, $\bar{U}_{y'}$, and $\bar{U}_{z'}$. These have been constructed in a manner similar to \bar{e}_x , \bar{e}_y , and \bar{e}_z . Referring to Fig. 2.7, the angular difference between the

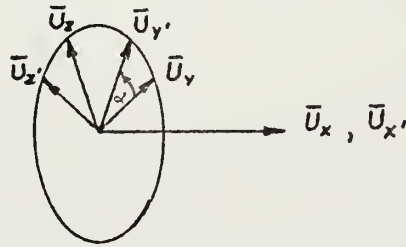


Figure 2.7: Unit Vector Rotation about x-axis

two adjacent sets of unit vectors is merely the rotation α about the local x-axis. Alpha may be determined by the expression

$$|\alpha| = \text{Cos}^{-1} (\bar{U}_y \cdot \bar{U}_{y'})$$

This rotation is in the positive (negative) \bar{U}_x sense according to whether the triple product $(\bar{U}_y \times \bar{U}_{y'}) \cdot \bar{U}_x$ is positive (negative).

In this way, the state vector is always expressed in the appropriate local coordinate system. For the simple system of Fig. 2.8

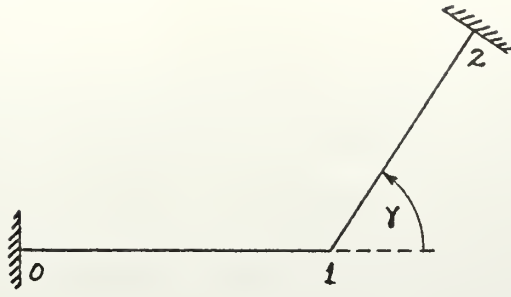


Figure 2.8: Planar System with Directional Change

$$z_1^L = U_1 z_0 \quad z_1^R = U_\gamma z_1^L \quad z_2 = U_2 z_1^R$$

which combining gives

$$z_2 = U_2 U_\gamma U_1 z_0$$

where U_γ is the transformation matrix corresponding to a rotation γ about the z-axis. For a 3-D system with the state vector defined as in section 2.1 this matrix is

$$U_\gamma = \text{DIAG} [T_\gamma, T_\gamma, T_\gamma, T_\gamma] \quad , \quad \text{where } T_\gamma = \begin{bmatrix} \cos \gamma & \sin \gamma & 0 \\ -\sin \gamma & \cos \gamma & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

Similarly, for the x-axis rotations

$$U_\alpha = \text{DIAG} [T_\alpha, T_\alpha, T_\alpha, T_\alpha] \quad , \quad \text{where } T_\alpha = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \alpha & \sin \alpha \\ 0 & -\sin \alpha & \cos \alpha \end{bmatrix}$$

C. 3-D TRANSFER MATRICES

1. The Straight Pipe Transfer Matrix

In order to maximize accuracy, a distributed mass model was decided upon for straight pieces of pipe. The program uses the straight bar transfer matrix cited in the appendix of Ref. 9, but revised to conform to the ordering

of the state vector described herein. The revised matrix, which appears in Appendix B, combines the vibration functions for longitudinal, torsional, and flexural modes and can include or neglect rotary inertia and shear deflection.

2. Development of the Curved Pipe Field Matrix

Due to the complexity and increased machine time involved in consideration of a distributed mass model for curved sections of pipe, a lumped mass treatment was chosen. This, coupled with the distributed mass approach for straight pipe, yields satisfactory accuracy combined with minimal computer time.

Given the compliance of a piece of massless curved pipe at its center of curvature, we seek the field matrix V such that $z_R = Vz_L$ (see Fig. 2.9) where $z_i = \begin{Bmatrix} D_i \\ F_i \end{Bmatrix}$. The

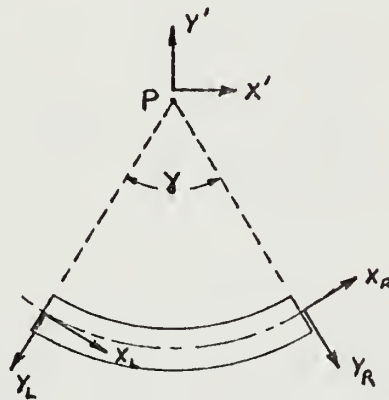


Figure 2.9: Massless Curved Pipe

compliance matrix C_p is defined by the relationship

$D_p = C_p F_p$ where D_p is a vector of generalized displacements observed at P and F_p is some vector of generalized forces applied at P. Point P is assumed to be attached by a rigid

bar to one end of the pipe while the other end of the pipe remains fixed. The compliance matrix for a pipe bend and its derivation are presented in a paper by J. E. Brock [Ref. 2].

The analysis requires that we find the compliance matrix at point L which is obtained by the appropriate rotation and translation of coordinates from point P such that

$$C_L = B_{PL}^T L_{PL}^T C_P L_{PL} B_{PL}$$

$$\text{where } B_{PL} = \begin{bmatrix} I & 0 \\ b_{PL} & I \end{bmatrix}, \quad b_{PL} = \begin{bmatrix} 0 & z_P - z_L & y_L - y_P \\ z_L - z_P & 0 & x_P - x_L \\ y_P - y_L & x_L - x_P & 0 \end{bmatrix}$$

for translation and

$$L_{PL} = \begin{bmatrix} K_{PL} & 0 \\ 0 & K_{PL} \end{bmatrix}, \quad K_{PL} = \begin{bmatrix} \cos(x_P, x_L) & \cos(x_P, y_L) & \cos(x_P, z_L) \\ \cos(y_P, x_L) & \cos(y_P, y_L) & \cos(y_P, z_L) \\ \cos(z_P, x_L) & \cos(z_P, y_L) & \cos(z_P, z_L) \end{bmatrix}$$

for rotation.

$$L_{PL} = L_{LP}^T \quad \text{and} \quad B_{PL} = B_{LP}^{-1}$$

The matrix equation for the generalized displacements at point L can now be written.

$$D_L = C_L F_L + B_{RL}^T L_{LR} D_R$$

where the rotation and translation matrices use the above notation. Solving for D_R we arrive at

$$D_R = L_{LR}^T B_{LR}^T D_L - L_{LR}^T B_{LR}^T C_L F_L \quad (2-7)$$

The equation for the generalized forces at the right end of the pipe has the form

$$F_R = -L_{RL} B_{RL} F_L \quad (2-8)$$

If equations 2-7 and 2-8 are combined in matrix form, we have the result

$$\begin{Bmatrix} D_R \\ F_R \end{Bmatrix} = \begin{bmatrix} L_{LR}^T B_{LR}^T & -L_{LR}^T B_{LR}^T C_L \\ 0 & -L_{RL} B_{RL} \end{bmatrix} \begin{Bmatrix} D_L \\ F_L \end{Bmatrix}$$

By the sign convention discussed in section 2.1, point L is located at a negative face, so after appropriate sign changes are taken into account, the field matrix for the bend becomes

$$V = \begin{bmatrix} L_{LR}^T B_{LR}^T & L_{LR}^T B_{LR}^T C_L \\ 0 & -L_{RL} B_{RL} \end{bmatrix}$$

This matrix in its explicit form, as used in program VIBREL, appears in Appendix B.

3. Development of Curved Pipe Point Matrix

The mass of a pipe bend or elbow is taken into account through the transfer matrix procedure by lumping it at the center of mass and including its effect on the system at point 0 (Fig. 2.10), the left end of the elbow section. The resulting point matrix relates only the forces across point 0 since the deflections are continuous.

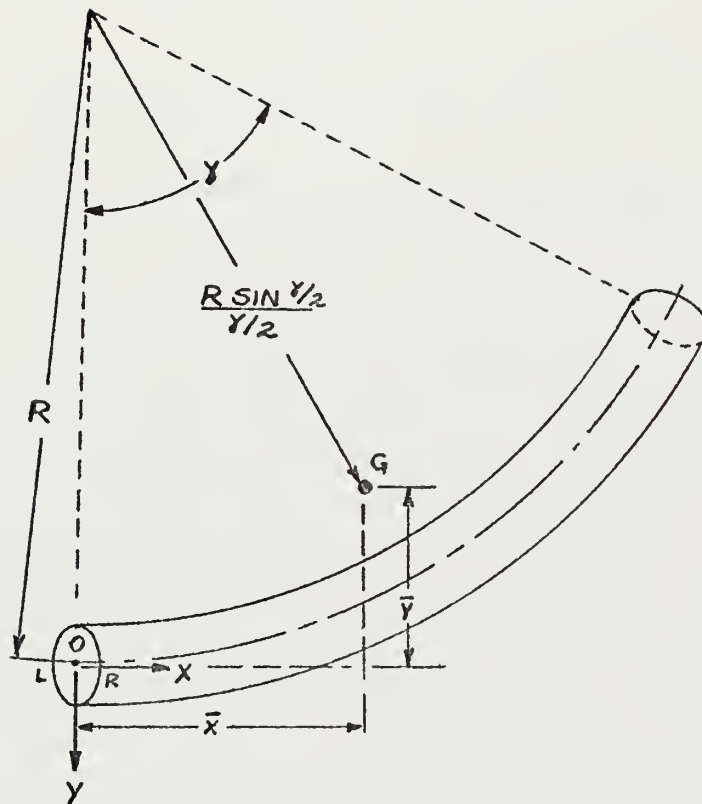


Figure 2-10: Curved Pipe Showing Location of Center of Mass

A force summation at point O yields the following scalar equations (the quantities appearing in this development are defined in Appendix A, subscripts L and R refer to the left and right of point O, respectively),

$$N_R = N_L - m\omega^2(u - \bar{y}\theta) \quad (2-9)$$

$$V_{yR} = V_{yL} - m\omega^2(v + \bar{x}\theta) \quad (2-10)$$

$$V_{zR} = V_{zL} - m\omega^2(w - \bar{y}\phi - \bar{x}\psi) \quad (2-11)$$

Similarly, a moment summation at the same point produces

$$T_R = T_L - I_{xO} \omega^2 \phi \quad (2-12)$$

$$M_{yR} = M_{yL} - I_{yO} \omega^2 \psi \quad (2-13)$$

$$M_{zR} = M_{zL} - I_{zO} \omega^2 \theta \quad (2-14)$$

Combining equations 2-9 through 2-14

$$F_R = TF_L$$

where

$$T = \begin{bmatrix} -m\omega^2 & 0 & 0 & 0 & 0 & -m\omega^2\bar{y} \\ 0 & -m\omega^2 & 0 & 0 & 0 & -m\omega^2\bar{y} \\ 0 & 0 & -m\omega^2 & m\omega^2\bar{y} & m\omega^2\bar{x} & 0 \\ 0 & 0 & 0 & -I_{xO}\omega^2 & 0 & 0 \\ 0 & 0 & 0 & 0 & -I_{yO}\omega^2 & 0 \\ 0 & 0 & 0 & 0 & 0 & -I_{zO}\omega^2 \end{bmatrix}$$

and the resulting point matrix equation is:

$$\begin{Bmatrix} D_R \\ F_R \end{Bmatrix} = \begin{bmatrix} I_6 & 0_6 \\ T & I_6 \end{bmatrix} \begin{Bmatrix} D_L \\ F_L \end{Bmatrix}$$

An inertia tensor analysis was used in the determination of the mass moments of inertia about point O. It can readily be shown that the inertia tensor at point O is

$$\begin{aligned} \bar{\bar{I}}_O = & \frac{\rho A R \bar{r}^2}{2} \left[\gamma \bar{u} + \left(\frac{\gamma}{2} + \frac{\sin 2\gamma}{4} \right) \bar{i}\bar{i} - \sin^2 \gamma \left(\frac{\bar{i}\bar{j} + \bar{j}\bar{i}}{2} \right) + \left(\frac{\gamma}{2} - \frac{\sin 2\gamma}{4} \right) \bar{j}\bar{j} \right] \\ & + 2\rho A R^3 [\gamma - \sin \gamma] \bar{u} + \rho A R^3 \left[\left(-\frac{\gamma}{2} + \sin \frac{2\gamma}{4} \right) \bar{i}\bar{i} \right. \\ & \left. + (1 - \cos \gamma - \frac{\sin^2 \gamma}{2}) (\bar{i}\bar{j} + \bar{j}\bar{i}) - \left(\frac{3\gamma}{2} - 2\sin \gamma + \frac{\sin 2\gamma}{4} \right) \bar{j}\bar{j} \right] \end{aligned}$$

where it is assumed that pipe wall thickness is small compared to pipe radius. (The symbols used here are defined in Appendix A.) Then $I_{xO} = \bar{i} \cdot \bar{\bar{I}}_O \cdot \bar{i}$, $I_{yO} = \bar{j} \cdot \bar{\bar{I}}_O \cdot \bar{j}$, and $I_{zO} = \bar{k} \cdot \bar{\bar{I}}_O \cdot \bar{k}$, where \bar{i} , \bar{j} , and \bar{k} are unit vectors along the local coordinate axes at point O. The ensuing moments of inertia and point matrix are presented in their entirety in Appendix B.

D. BRANCHED SYSTEMS

Any branch joining a piping system has a significant effect on its dynamic behavior. While the deflections across the junction point are continuous, there is a discontinuity in the forces, the magnitude of which is dependent on the displacement at the junction point and the nature of the branch.

Consider the simple branch system of Fig. 2.11.

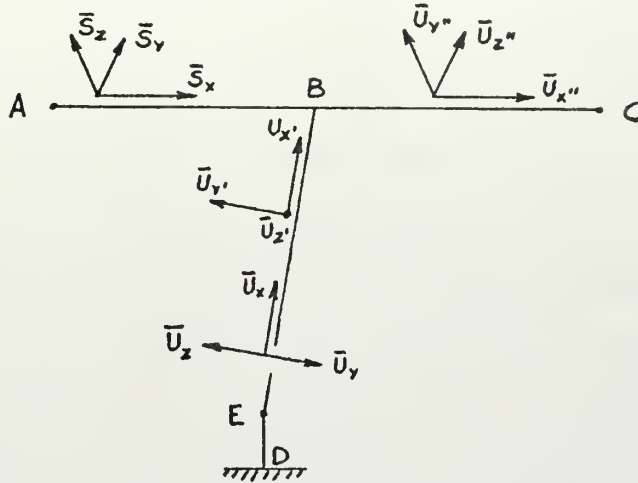


Figure 2.11: Unit Vector Orientation at a Branch Junction

Using the state matrix concept discussed in section 2.1 we have the following relationship between the state vector at point D and the state vector at point B (the bar denotes branch coordinate scheme).

$$\begin{Bmatrix} \bar{D}_B \\ \bar{F}_B \end{Bmatrix}^{12 \times 1} = \begin{bmatrix} R_1 & 6 \times 6 \\ -\frac{1}{6} & -X \frac{1}{6} \\ R_2 & \end{bmatrix} \{\bar{z}_D\}^{6 \times 1} \quad (2-15)$$

The zero quantities of the state vector at D have been eliminated by application of the boundary conditions and introduction of the state matrix, and the branch transfer matrix has been partitioned into two submatrices R_1 and P_2 . The local coordinate system of the branch at point B is represented by the unprimed set of unit vectors. From the relationship 2-15 it can be seen that $\bar{D}_B = R_1 \bar{z}_D$ and $\bar{F}_B = R_2 \bar{z}_D$, so that when \bar{z}_D is eliminated, the equation between the forces and displacements at point B in the unprimed coordinate system becomes $\bar{F}_B = R_2 R_1^{-1} \bar{D}_B$ (2-16).

The vectors of the generalized forces and displacements in the branch system must now be transferred to the main member system represented by the unit vectors \bar{s} in Fig. 2.11. This can be achieved by one z-axis and two x-axis transformations of the unprimed coordinates. We first form the primed set of unit vectors in a manner identical to that for a corner point (section 2.2), the three working points describing the corner being E, B and C. An x-axis rotation α_1 will align the unprimed with the primed coordinates. Next, a rotation γ about the z' -axis will allow u_x , to coincide with the centroidal axis of section BC resulting in the double-primed coordinate scheme. Finally, rotation through the angle α about the x'' -axis will align the branch system with the main member coordinates, s . The total transformation of the displacements can be expressed by

$$D_B = G_\alpha G_\gamma G_{\alpha_1} \bar{D}_B = G \bar{D}_B \quad (2-17)$$

$$\text{where } G_\alpha = \text{DIAG} \left\{ \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \alpha & \sin \alpha \\ 0 & -\sin \alpha & \cos \alpha \end{bmatrix}, \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \alpha & \sin \alpha \\ 0 & -\sin \alpha & \cos \alpha \end{bmatrix} \right\},$$

similarly for G_{α_1} , and

$$G_\gamma = \text{DIAG} \left\{ \begin{bmatrix} \cos \alpha & \sin \alpha & 0 \\ -\sin \alpha & \cos \alpha & 0 \\ 0 & 0 & 1 \end{bmatrix}, \begin{bmatrix} \cos \alpha & \sin \alpha & 0 \\ -\sin \alpha & \cos \alpha & 0 \\ 0 & 0 & 1 \end{bmatrix} \right\}.$$

Likewise for the forces, $F_B = G \bar{F}_B$ (2-18). Rearranging relations 2-17 and 2-18 and substituting in 2-16 yields

$$G^{-1}F_B = R_2 R_1^{-1} G^{-1} D_B .$$

Noting that $G^{-1} = G^T$ we now form the force displacement equation

$$F_B = G R_2 R_1^{-1} G_{\alpha 1}^T G_{\gamma}^T G_{\alpha}^T D_B .$$

Referring to the free body diagram in Fig. 2.12 we find

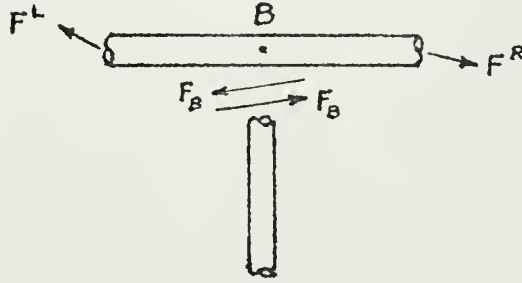


Figure 2.12: Generalized Forces at a Branch Junction

that the main member force equation at point B can be written

$$\begin{aligned} F^R &= F^L + F_B \\ &= F^L + G R_2 R_1^{-1} G_{\alpha 1}^T G_{\gamma}^T G_{\alpha}^T D_B . \end{aligned}$$

Thus, since $D_B = D^L = D^R$, the point matrix relating the state vectors to the left and right of a branch junction point of a 3-D system is given by

$$\begin{Bmatrix} D \\ F \end{Bmatrix}^R = \left[\begin{array}{c|c} I & 0 \\ \hline G R_2 R_1^{-1} G_{\alpha 1}^T G_{\gamma}^T G_{\alpha}^T & I \end{array} \right] \begin{Bmatrix} D \\ F \end{Bmatrix}^L$$

The foregoing development assumes that there is no curvature of the main member or branch at point B and that no abrupt change of direction of the main member occurs at that immediate location.

For two or more branches joining the main member at a single point, the system transfer matrix just prior to the branch point is simply multiplied in turn by the point matrix for each of the branches.

E. COMPUTER IMPLEMENTATION OF TRANSFER MATRICES

It would be an extremely tedious job to attempt explicit assemblage of the elements of the system transfer matrix in terms of the circular frequency. Subsequent solution for the roots of the expanded frequency determinant would also be impractical for all but the simplest of systems. It is for these reasons that digital computation becomes a necessity.

Program VIBREL, which incorporates the theory of preceding sections, was developed to utilize the speed with which the complete system transfer matrix can be formed time after time for various values of frequency. A typical graph of frequency determinant $\Delta(\omega)$ versus circular frequency ω is shown in Fig. 2.13. The values of ω at which the graph

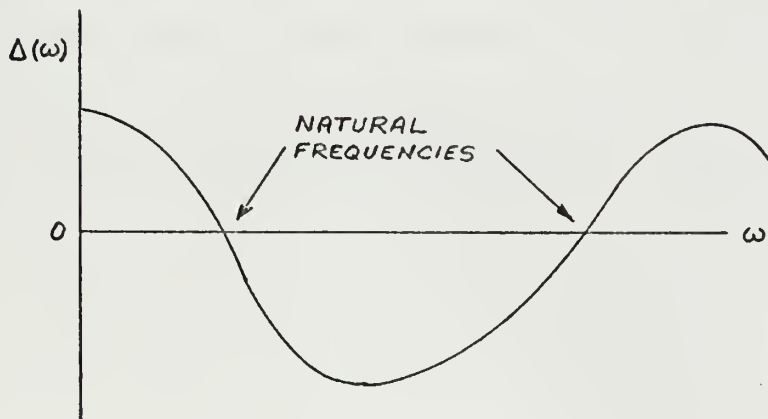


Figure 2.13: Graph of Frequency Determinant vs. Frequency

crosses the horizontal axis, namely those values of ω for which $\Delta(\omega) = 0$, are the natural frequencies of the system.

The program, presented in Appendices C through E, was coded using Fortran language [Ref. 7] in double precision to minimize the effects of round-off and truncation errors. Since the prime objective was to produce a working program of demonstrable accuracy and reliability, limited attention was focused on the niceties of programming intended to reduce execution time. However, Kim's state matrix concept is included and this effects a substantial saving of time as compared to other procedures. There is no reason to believe that the actual programming is particularly inefficient; however, it is not unlikely that close scrutiny of program details might point out some places where slight revisions would effect economies.

The transfer matrices used in computation of system frequencies are those presented in Appendix B.

Input formats were designed to provide satisfactory simplicity for the user in preparing data decks, while output formats were chosen to check for correct input as well as to provide a detailed summary of results. The I/O formats are discussed more thoroughly in Appendix E.

III. DISCUSSION

A. SCOPE OF THE SOLUTION

In its present form, program VIBREL is capable of analyzing a piping system having few restrictions regarding topology. The system must, however, be composed of a main member from which can emanate as many as fifty branches. The program as listed in Appendix E can accommodate no more than two branches joining the main member at a particular point. In addition, there can be no branches emanating from other branches, nor can there be any curvature of the main member or branch at a branch junction point.

The total number of sections capable of being analyzed, presently set at one hundred, can be increased by following the procedure described in Appendix C. Likewise the number of branches can be augmented from its present limit of fifty. For each increase of one section, eighty additional bytes of computer storage are needed above the 140,000 bytes now required for the entire program. For most computer installations this allows considerable leeway in the size of the piping system which can be handled by the program.

To provide an idea of the computer time involved in the frequency analysis, the twelve-section piping system of the sample problem in Appendix G required about four minutes for computation of four modes and almost fifteen minutes for eighteen modes. The IBM 360 computer compiles the program in thirty-nine seconds.

The usual assumptions of linearity as well as homogeneity and isotropy of the structural material have been incorporated in the analysis. All sections of the system are assumed to vibrate isochronously with negligible damping.

B. NATURE OF ERRORS AND INACCURACIES

1. Round-Off Errors

As piping system complexity increases the number of multiplications required to form the system transfer matrix grows proportionately. Each time a multiplication occurs, there is some round-off and loss of accuracy; this is a function of the significant figure capacity of the computer.

FORTTRAN, in conjunction with the IBM 360 and double precision arithmetic, has the capability of carrying numbers of 16 significant digits with an exponent range of 10^{-78} to 10^{78} . Simpler systems can be checked for accuracy and it is evident that this significant digit capacity keeps round-off errors negligible. Since with more complex systems other forms of errors creep into the results, we have no way of predicting the exact contribution of round-off errors to total inaccuracy of frequency.

It is reasonable to assume that round-off plays a more significant role in the 3-D piping system accuracy than in a planar case with equivalent sections because of the greater number of coordinate transformations that are necessary for system description.

2. Lumped Mass Idealization

For sections of curved pipe, lumped mass connected by massless springs is used as a model for the real system. This idealization leads to inherent inaccuracy in the analysis of systems containing bends.

3. Zeros of the Frequency Determinant

Once it has been established that the frequency determinant has crossed the zero axis, Program VIBREL uses Mueller's method of successive bisection and inverse parabolic interpolation to determine the zero. Care should be exercised in considering as significant only the same number of places as in the acceptability criterion which was specified for the solution. (See Appendix C.)

4. Other Sources of Inaccuracies

The imperfect nature of the real system and inevitable errors in the measurement of working points will render the computer solution only a good engineering approximation of what can be expected in actuality. In the discussion of accuracy in succeeding sections, inaccuracies caused by real system imperfections will not be considered.

C. ACCURACY AND INTEGRITY OF SOLUTION

1. Solution Accuracy

Confidence in the accuracy of the program solution was established by three comparison tests, the results of which were tabulated in Appendix F.

a. Analytical Comparison

For single straight sections of pipe a closed form solution was obtained from the governing fourth order differential equation with the appropriate fixed, free, or ball joint end conditions. The largest variation between the natural frequencies calculated analytically and those computed by program VIBREL was 0.009%.

b. Reference Value Comparison

Where the exact closed form solution was not available, recourse to the literature provided some comparison frequencies. For the simple curved pipe shown in Appendix F, Refs. 5, 6, and 8 contained a range of frequency values for the first two modes. VIBREL results for the same curved pipe fell within this range in each of the two modes examined, with an average variation from the reference values of about 4%. Because, for a curved pipe, program VIBREL includes bend flexibility factor and an accurate value for shear distribution factor taken from Ref 2, the values computed are considered to be at least as accurate as those contained in the literature.

For a single branch system composed of straight pipes only, comparison frequencies for three modes were available from Ref. 6. The largest difference between reference and VIBREL in this case was 0.06%.

c. Dual Analysis Comparison

Dual analysis is based on the fact that the system transfer matrix is directionally dependent. One end of the

system main member is designated as the starting point; then the component transfer matrices are multiplied together in sequence proceeding toward the other end. If an error were to occur in the structure of one of the transfer matrices or in the method of their combination to form the system transfer matrix, the results from starting at opposing ends could differ significantly.

An improperly constructed state matrix would also be evident from a dual analysis.

The straight pipe systems and branched systems showed no significant differences when subjected to dual analysis. The maximum difference observed in eighteen modes of the complex system dual analysis was 2.67%, while the average difference was 0.58%.

2. Solution Integrity

Solution integrity was established by observing that the mode frequencies detected by the program were, in fact, those of the system with no omissions. This was accomplished by comparison with analytical and reference results. Dual analysis comparison was also used in determining that the same modes had been detected in either direction. No discrepancies were noted in any of the systems checked on the basis of these comparisons.

Program VIBREL includes in its output a graph of frequency determinant versus frequency in order to check whether the frequency increment is such that a mode or modes have been skipped in the scanning process. In the

first analysis of a branched system which appears in Appendix F, the trend of the output plot showed that two natural frequencies in close proximity had been overlooked. When the program was rerun with the starting frequency increment decreased by a factor of 2, these two frequencies were detected.

The maximum number of natural frequencies which can be found using VIBREL is related to the significant figure capacity of the computer and the physical configuration of the piping system. As pointed out by Kim [Ref. 6], at the higher frequencies the columns of the frequency determinant approach the point of being parallel; hence the numerical value of the determinant includes differences of large numbers. Systems with few components generally exhibit this parallelism at a lower frequency than the more complex ones; however, there is no way of determining beforehand where numerical difficulties will be encountered. When parallelism is approached, a scattering of the values of frequency determinant will be noticed in the output plot.

Although the exponent limit on the IBM 360 computer used for testing the program is +78, program VIBREL has been coded to cease computation for a particular problem and print a message when the exponent of frequency determinant exceeds +60.

D. CONCLUSIONS AND RECOMMENDATIONS

Based on the results of the accuracy and integrity analyses previously discussed and through comparison with similar programs developed for planar cases, namely those of Fink [Ref 5] and Kim [Ref. 6], program VIBREL could be employed in its present form as a working tool for engineers in the dynamic analysis of spatial piping systems. The fact that eighteen mode frequencies were obtained for the complex system exhibited in Appendix F implies that at least the first few and usually the most useful frequencies can be determined for practical piping configuration with an accuracy consistent with engineering design.

Minor modifications for adapting the program to a particular situation can be made using the guidelines of Appendix C. With regard to major revisions, program VIBREL was developed with solution accuracy and integrity of first importance. It is recommended for future users that changes in the coding be made for a minimum computer execution time if that becomes a prime requisite. Avenues of investigation regarding numerical difficulties at higher frequencies are also open to the future user. A similar analysis for the planar case was undertaken by Kim [Ref. 6], who utilized alternatives to the frequency determinant approach.

Although the calculation of mode shapes of the piping system has not been included in the program because of time limitations, an outline of how to incorporate this feature is discussed in the following paragraphs. This modification

to the program is recommended as a further important step in the dynamic analysis of 3-D piping systems.

Let us consider again, for simplicity, the two-dimensional system of Fig. 2.3 which is repeated below. We found that the total system frequency condition could be expressed as:

$$\begin{bmatrix} a_{13} & a_{24} \\ a_{23} & a_{24} \end{bmatrix} \begin{Bmatrix} A_1 \\ A_2 \end{Bmatrix} = 0 \quad (3-1)$$

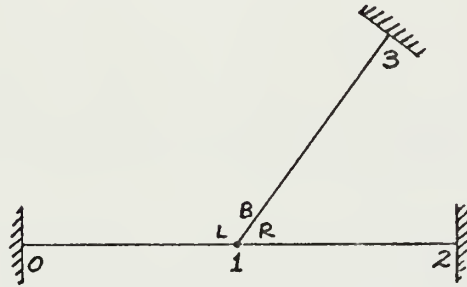


Figure 2.3: Single Branch Planar System

A vanishing determinant of the coefficient matrix yields a natural frequency. At this frequency there is a non-trivial solution to Eqs. 3-1 and A_1/A_2 can be found. Normalizing A_1 and A_2 gives the modal values of the non-zero quantities of the left end state vector, A_1^* and A_2^* . A suitable normalizing condition would be $A_1^2 + A_2^2 = 1$. With these values known we proceed along the main member calculating the modal deflections and forces at each divisional point. This is done simply by multiplying the normalized and compressed left end state vector by the system transfer matrix ($2r \times r$). In Fig. 2.3 the modal forces and deflections to the left of point 1 would be

$$\left\{ \begin{matrix} w \\ \psi \\ v_z \\ M_z \end{matrix} \right\}_{1b} = \begin{bmatrix} b_{13} & b_{14} \\ b_{22} & b_{24} \\ b_{33} & b_{34} \\ b_{43} & b_{44} \end{bmatrix} \begin{Bmatrix} A_1^* \\ A_2^* \end{Bmatrix} \quad (3-2)$$

In order to permit unequivocal interpretation of program output, it would be advisable to incorporate a point numbering scheme for each point at which modal forces and deflections are desired. In addition, these forces and deflections will be expressed in the local coordinate systems and a method should be incorporated for transforming them to the global system.

When a branch point, such as 1 in Fig. 2.3, is encountered, the branch transfer matrix up to point 1 is formed. Thus

$$\left\{ \begin{matrix} w \\ \psi \\ v_z \\ M_z \end{matrix} \right\}_{1B} = \begin{bmatrix} c_{13} & c_{14} \\ c_{23} & c_{24} \\ c_{33} & c_{34} \\ c_{43} & c_{44} \end{bmatrix} \begin{Bmatrix} B_1 \\ B_2 \end{Bmatrix} \quad (3-3)$$

where B_1 and B_2 are the unknown quantities of the state vector at the remote end of the branch. Since we know that the deflections are continuous across the junction point

$$\left\{ \begin{matrix} w \\ \psi \end{matrix} \right\}_{1B} = \left\{ \begin{matrix} w \\ \psi \end{matrix} \right\}_{1L} \quad (3-4)$$

The quantities at 1L are given in terms of the known values A_1^* and A_2^* so that the values of B_1 and B_2 which are intrinsic to the 1B deflections may be solved for through the relations 3-4. The known values of B_1 and B_2 may now be used to find modal forces and deflections in a manner similar to that used for the main member.

Just to the right of the junction point, 1R, the modal deflections are the same as at the left; however, the forces must be calculated by adding the 1L forces to the 1B forces in the global coordinate system.

APPENDIX A

NOTATION AND NOMENCLATURE

The following matrix notation is used in the text:

$[]$ - matrix

$\{ \}$ - $n \times 1$ column vector

DIAG $\{ \}$ - square matrix having the given elements
in its principal diagonal and zero
elsewhere

T - superscript denoting the transpose of a given
matrix

The following vector notation is used in the text:

\bar{A} - vector A

$|\bar{A}|$ - length of vector A

$\bar{A} \cdot \bar{B}$ - dot product of vectors A and B

$\bar{A} \times \bar{B}$ - cross product of vectors A and B

Table A.1 first lists the symbols used in either the test or Appendix B, with the corresponding computer program variable identifiers. Following these appear the program variable names which have no counterpart in the text.

TABLE A.1

TEXT SYMBOL	PROGRAM VARIABLE	DESCRIPTION
A	AREA	cross sectional area of pipe
\bar{A}	A	first of two vectors defining a piping directional change

TABLE A.1 (Continued)

TEXT SYMBOL	PROGRAM VARIABLE	DESCRIPTION
B		translation matrix
\bar{B}	B	second of two vectors defining a piping directional change
C		compliance matrix
D		vector of generalized displacements
\bar{e}_x	UX	unit vector in local x-direction
E	EY	Young's modulus
F		vector of generalized forces
G	G	shear modulus
G_α	G2	branch coordinate transformation matrix
G_γ	G3	branch coordinate transformation matrix
$G_{\alpha 1}$	G1	branch coordinate transformation matrix
\bar{i}		unit vector in local x-direction
I_x	DIX	mass moment of inertia of a curved pipe about x-axis
\bar{I}_0		inertia tensor about point 0
I_6		6 x 6 identity matrix
i		radius of gyration of pipe cross section
\bar{j}		unit vector in local y-direction
J	DJ	moment of inertia of pipe section
\bar{k}		unit vector in local z-direction
k	BFF	bend flexibility factor
ℓ	DL	length of a straight section of pipe
L		rotation matrix
m	DM	mass of a section of pipe

TABLE A.1 (Continued)

TEXT SYMBOL	PROGRAM VARIABLE	DESCRIPTION
M_y		moment about local y-axis
M_z		moment about local z-axis
N		axial force in local x-direction
O_6		6 x 6 null matrix
P		point matrix
R	R	radius of curvature of pipe bend
\bar{r}	$RBAR$	mean radius of pipe section
R_1	$R1$	submatrix of branch transfer matrix
R_2	$R2$	submatrix of branch transfer matrix
r_G	RG	radius of center of mass of a curved section of pipe
\bar{s}_x	SX	main member unit vector just prior to branch junction
t	T	pipe wall thickness
T		torque about local x-axis
u		deflection in x-direction
\bar{u}		unit dyadic
\bar{U}_x	UX	unit vector in local x-direction
$\bar{U}_{x'}$	UXP	unit vector in local x'-direction
$\bar{U}_{x''}$	$UXDP$	unit vector in local x''-direction
U	$U, U1$	transfer matrix
v		deflection in local y-direction
V		field matrix
V_y		shear force in local y-direction
V_z		shear force in local z-direction

TABLE A.1 (Continued)

TEST SYMBOL	PROGRAM VARIABLE	DESCRIPTION
w		deflection in local z-direction
z		state vector
α	AL	x-axis coordinate transformation angle
α_1	AL1	x-axis coordinate transformation angle
γ	GAM	z-axis coordinate transformation angle
Δ		frequency determinant
θ		angular deflection about z-axis
μ	UU	mass per unit length of pipe
δ	PR	Poisson's ratio
ξ	SDF	shear distribution factor
ρ	RHOM	mass density
ϕ		angular deflection about x-axis
ψ		angular deflection about y-axis
ω	W1,W2	circular frequency
	AA	array of section properties
	ARCL	arc length of curved pipe
	BL	distance from bend point to tangent point
	CTX12	12 x 12 x-axis coordinate transformation matrix
	CTZ12	12 x 12 z-axis coordinate transformation matrix
	D	value of frequency determinant
	D1	outside pipe diameter
	D2	inside pipe diameter

TABLE A.1 (Continued)

TEXT SYMBOL	PROGRAM VARIABLE	DESCRIPTION
	DD	frequency determinant array
	FN	array of natural frequencies
	GO	gravitational constant
	ISEC	section identifier code
	JBC	branch boundary condition code
	K	boundary condition sequence number
	KB	branch repeat discriminant
	LBC	left boundary condition code-main member
	LBR	branch point identifier discriminant
	M	dimension of array of frequency determinant values
	MBC	right boundary condition code-main member
	MCS	curved subsection override discriminant
	MSR	shear deflection/rotary inertia discriminant
	MST	straight section test discriminant
	N	section number
	NB	vector of boundary condition codes
	NIB	initial unit vector discriminant for a branch
	NID	point identifier code
	NIT	number of iterations required for Mueller's method
	NIV	initial unit vector discriminant for main member
	NK	dual branch discriminant
	NMO	number of modes requested in analysis

TABLE A.1 (Continued)

TEXT SYMBOL	PROGRAM VARIABLE	DESCRIPTION
	NN	iteration section number
	NPROB	number of systems to be analyzed by program
	NSEC	section number
	NSS	number of curved pipe subsections
	PX	X-coordinate of a working point
	PY	Y-coordinate of a working point
	PZ	Z-coordinate of a working point
	RHO	weight density
	UA	point matrix for a curved pipe
	UB,UCY	field matrix for a curved pipe
	UST	straight pipe transfer matrix
	WINCR	frequency increment
	WN	natural frequency
	X	array of frequencies for output plot
	Y	array of frequency determinant values for output plot

CATALOG OF TRANSFER MATRICES USED IN THE PROGRAM

B.1 STRAIGHT PIPE TRANSFER MATRIX

$\cos \alpha_0$	0	0	0	0	0	0	$\frac{l}{EA}$	$\frac{\sin \alpha_0}{\alpha_0}$	0	0	0	0	0	0	0
0	$C_0 - \sigma C_2$	0	0	0	0	$l[C_1 - (\sigma + r)C_3]$	0	$-\frac{\alpha_0}{l}[-\sigma C_1 + (\sigma^2 + r^2)C_3]$	0	0	0	0	0	0	aC_2
0	0	$C_0 - \sigma C_2$	0	$l[C_1 - (\sigma + r)C_3]$	0	0	0	0	$-\frac{\alpha_0}{l}[-\sigma C_1 + (\sigma^2 + r^2)C_3]$	0	0	0	$-aC_2$	0	0
0	0	0	$\cos \gamma_0$	0	0	0	0	0	0	$\frac{l}{GA} \frac{\sin \gamma_0}{\gamma_0}$	0	0	0	0	0
0	0	$-\frac{\theta^4}{l} C_3$	0	$C_0 - rC_2$	0	0	0	0	0	aC_2	0	0	$\frac{a}{l}(C_1 - rC_3)$	0	0
0	$\frac{\theta^4}{l} C_3$	0	0	0	0	$C_0 - rC_2$	0	$-aC_2$	0	0	0	0	0	$\frac{a}{l}(C_1 - rC_3)$	0
$-\mu l \omega^2 \frac{\sin \alpha_0}{\alpha_0}$	0	0	0	0	0	0	$\cos \alpha_0$	0	0	0	0	0	0	0	0
0	$-\frac{\theta^4}{a l}(C_1 - \sigma C_3)$	0	0	0	$-\frac{\theta^4}{a} C_2$	0	0	0	$C_0 - \sigma C_2$	0	0	0	0	0	0
0	0	$-\frac{\theta^4}{a l}(C_1 - \sigma C_3)$	0	$\frac{\theta^4}{a} C_2$	0	0	0	0	0	$C_0 - \sigma C_2$	0	0	0	0	0
0	0	0	$-\mu l \omega^2 \frac{\sin \gamma_0}{\gamma_0}$	0	0	0	0	0	0	0	$\cos \gamma_0$	0	0	0	0
0	0	$-\frac{\theta^4}{a l} C_2$	0	$\frac{a}{l} l[r C_1 + (\theta^2 + r^2)C_3]$	0	0	0	0	0	$l[C_1 - (\sigma + r)C_3]$	0	0	$C_0 - rC_2$	0	0
0	$\frac{\theta^4}{a} C_2$	0	0	0	0	$\frac{a}{l} l[r C_1 + (\theta^2 + r^2)C_3]$	0	$-l[C_1 - (\sigma + r)C_3]$	0	0	0	0	0	$C_0 - rC_2$	0

$$\alpha_0 = l \omega \sqrt{\frac{\mu}{AE}}$$

$$\gamma_0 = l \omega \sqrt{\frac{\mu}{AG}}$$

$$a = l^2/EJ$$

$$\theta^4 = \mu \omega^2 l^4/EJ$$

$$\sigma = \mu \omega^2 l^2/GA$$

$$r = \frac{l^2}{EJ} (\mu l^2 \omega^4)$$

$$\Lambda = \frac{1}{\Lambda_1^2 + \Lambda_2^2}$$

$$\Lambda_1 = \sqrt{\beta^2 + \frac{1}{2}(\sigma - r)^2} - \frac{1}{2}(\sigma + r)$$

$$\Lambda_2 = \sqrt{\beta^2 + \frac{1}{2}(\sigma - r)^2} + \frac{1}{2}(\sigma + r)$$

$$C_0 = \Lambda (\lambda_1^2 \cosh \lambda_1 + \lambda_2^2 \cos \lambda_2)$$

$$C_1 = \Lambda (\lambda_1^2/\lambda_1 \sinh \lambda_1 + \lambda_2^2/\lambda_2 \sin \lambda_2)$$

$$C_2 = \Lambda (\cosh \lambda_1 - \cos \lambda_2)$$

$$C_3 = \Lambda (1/\lambda_1 \sinh \lambda_1 - 1/\lambda_2 \sin \lambda_2)$$

B.2 CURVED PIPE FIELD MATRIX

$\cos \gamma$	$-\sin \gamma$	0	0	0	$A_1 \cos \gamma + A_2 R(\cos \gamma - 1) - A_2 \sin \gamma$	$A_1 \cos \gamma + \sin^2 \gamma \cdot R \cdot (\cos \gamma - 1) - A_2 \sin \gamma$	0	0	$A_0 \cos \gamma + A_1 R(\cos \gamma - 1) - A_2 \sin \gamma \sin \gamma_2$
$\sin \gamma$	$\cos \gamma$	0	0	0	$A_1 \sin \gamma + A_2 R \sin \gamma + A_3 \cos \gamma$	$A_1 \sin \gamma + A_2 R \sin \gamma + A_3 \cos \gamma$	0	0	$A_0 \sin \gamma + A_1 R \sin \gamma + A_2 \cos \gamma \sin \gamma_2$
0	0	1	$R(\cos \gamma - 1)$	$-\sin \gamma$	0	0	$C_1 + D_1 R(\cos \gamma - 1) - C_2 R \sin \gamma + D_2 R \sin \gamma$	C_2	0
0	0	0	$\cos \gamma$	$-\sin \gamma$	0	0	$D_1 \cos \gamma - C_2 \sin \gamma + D_2 \cos \gamma - D_3 \sin \gamma + D_3 \cos \gamma - E_3 \sin \gamma$	0	0
0	0	0	$\sin \gamma$	$\cos \gamma$	0	0	$D_3 \sin \gamma + C_3 \cos \gamma + D_2 \sin \gamma + D_3 \cos \gamma + D_3 \sin \gamma + E_3 \cos \gamma$	0	0
0	0	0	0	0	1	A_3	$H \sin \gamma_2$	0	0
0	0	0	0	0	0	$\cos \gamma$	$-\sin \gamma$	0	0
0	0	0	0	0	0	$\sin \gamma$	$\cos \gamma$	0	0
0	0	0	0	0	0	0	0	0	0
0	0	0	0	0	0	0	1	0	0
0	0	0	0	0	0	0	$R(\cos \gamma - 1)$	$\cos \gamma$	$-\sin \gamma$
0	0	0	0	0	0	0	$R \sin \gamma$	$\sin \gamma$	$\cos \gamma$
0	0	0	0	0	0	$R(\cos \gamma - 1)$	$-\sin \gamma$	0	1

$$\begin{aligned}
 S &= 2(1+\delta)E \\
 k &= 1.65 F^2 / R t \quad (\text{BUT} \geq 1) \\
 P &= [R^2 k + i^2(1-\zeta)]/2 \\
 Q &= (1+\delta-k)/2 \\
 \psi_1 &= \gamma + \sin \gamma \\
 \psi_2 &= \gamma - \sin \gamma \\
 A &= (i^2 \zeta \gamma + P \psi_1) R / EJ \\
 B &= (i^2 \zeta \gamma + P \psi_2) R / EJ \\
 M &= [i^2 \zeta + R^2(1+\delta)] \gamma R / EJ \\
 D &= [2R(1+\delta) \sin^2 \gamma_2] R / EJ \\
 E_1 &= (k \gamma + Q \psi_1) R / EJ \\
 F &= (k \gamma + Q \psi_2) R / EJ \\
 G &= k \gamma R / EJ \\
 H &= -2R^2 k \sin^2 \gamma_2 / EJ \\
 A_1 &= A \cos^2 \gamma_2 + 2RH \cos \gamma_2 + B \sin^2 \gamma_2 + R^2 G \\
 A_2 &= (B-A) \sin \gamma / 2 - RH \sin^2 \gamma_2 \\
 A_3 &= -H \cos \gamma_2 - RG \\
 C_1 &= M - 2RD \cos^2 \gamma_2 + R^2 E \cos^2 \gamma_2 + R^2 F \sin^2 \gamma_2 \\
 C_2 &= -D \cos^2 \gamma_2 + RE, \cos^2 \gamma_2 + RF \sin^2 \gamma_2 \\
 C_3 &= D \sin^2 \gamma_2 + R \sin \gamma (F-E_1)/2 \\
 D_1 &= -D \cos^2 \gamma_2 + RE, \cos^2 \gamma_2 + RF \sin^2 \gamma_2 \\
 D_2 &= E, \cos^2 \gamma_2 + F \sin^2 \gamma_2 \\
 D_3 &= \sin \gamma (F-E_1)/2 \\
 E_3 &= E, \sin^2 \gamma_2 + F \cos^2 \gamma_2
 \end{aligned}$$

1	0	0	0	0	0	0	0	0	0	0
0	1	0	0	0	0	0	0	0	0	0
0	0	1	0	0	0	0	0	0	0	0
0	0	0	1	0	0	0	0	0	0	0
0	0	0	0	1	0	0	0	0	0	0
0	0	0	0	0	1	0	0	0	0	0
-m\omega^2	0	0	0	0	0	0	-m\omega^2 \bar{y}	1	0	0
0	-m\omega^2	0	0	0	0	0	-m\omega^2 \bar{x}	1	0	0
0	0	-m\omega^2	m\omega^2 \bar{y}	m\omega^2 \bar{x}	0	0	0	0	1	0
0	0	0	-I_{x_0} \omega^2	0	0	0	0	0	1	0
0	0	0	0	-I_{y_0} \omega^2	0	0	0	0	0	1
0	0	0	0	0	0	-I_{z_0} \omega^2	0	0	0	0

$$\begin{aligned} r_g &= R \sin \frac{\gamma}{2} / \gamma_{1/2} \\ \bar{X} &= r_g \sin \frac{\gamma}{2} \\ \bar{Y} &= R \cdot r_g \cos \frac{\gamma}{2} \end{aligned}$$

APPENDIX C

DESCRIPTION OF PROGRAM

1. General Remarks

Program VIBREL is a double precision FORTRAN language digital computer program which is capable of performing a frequency analysis of a three-dimensional piping system. The program accomplishes this analysis by means of the transfer matrix method in which each section of pipe is characterized by a matrix known as a transfer matrix. These matrices which are functions of frequency are multiplied together in succession to form a transfer matrix which is characteristic of the entire system. To combine satisfactory accuracy with minimum computer time, a lumped mass model is used for curved sections of pipe while for straight pipe, a distributed mass approach is employed. Shear deflection and rotary inertia are optional in the model.

The solution is obtained by incrementing the frequency and forming the system transfer matrix and frequency determinant with each increment. The sign of the frequency determinant is used to control the search for and convergence to system natural frequencies. The convergence acceptability criterion may be set arbitrarily by the user depending on the accuracy desired. (See section 3.b of this appendix.) As one would expect, the greater the accuracy, the more computer time required.

The system is defined by working points which are read into the program with the intensive and extensive properties of the section of pipe preceding it. The details for this procedure are contained in Appendix E. The intensive and extensive properties may not vary along a single section of pipe but may vary from section to section. Projective complexities, including branches emanating from branches, cannot be handled directly by the program.

Three boundary conditions, fixed, free, or ball joint, may be specified by coded input for the ends of the main member and the remote ends of the branches. Alternative end conditions are discussed in section 3.b of this appendix.

2. Program Structure

a. Main Program

A simplified flow chart is shown in Fig. C.1. This discussion will pertain to that diagram.

Data which contains working points, intensive and extensive properties, and boundary condition codes is read into the program as it is required for computation. Local coordinate systems are set up at the starting point, prior to and following each directional change, and at branch junction points. From the working point locations and unit vectors describing the local coordinate systems, section lengths and coordinate rotation angles are computed.

For computational purposes piping bends are subsectioned based on the length-to-diameter ratio in the following

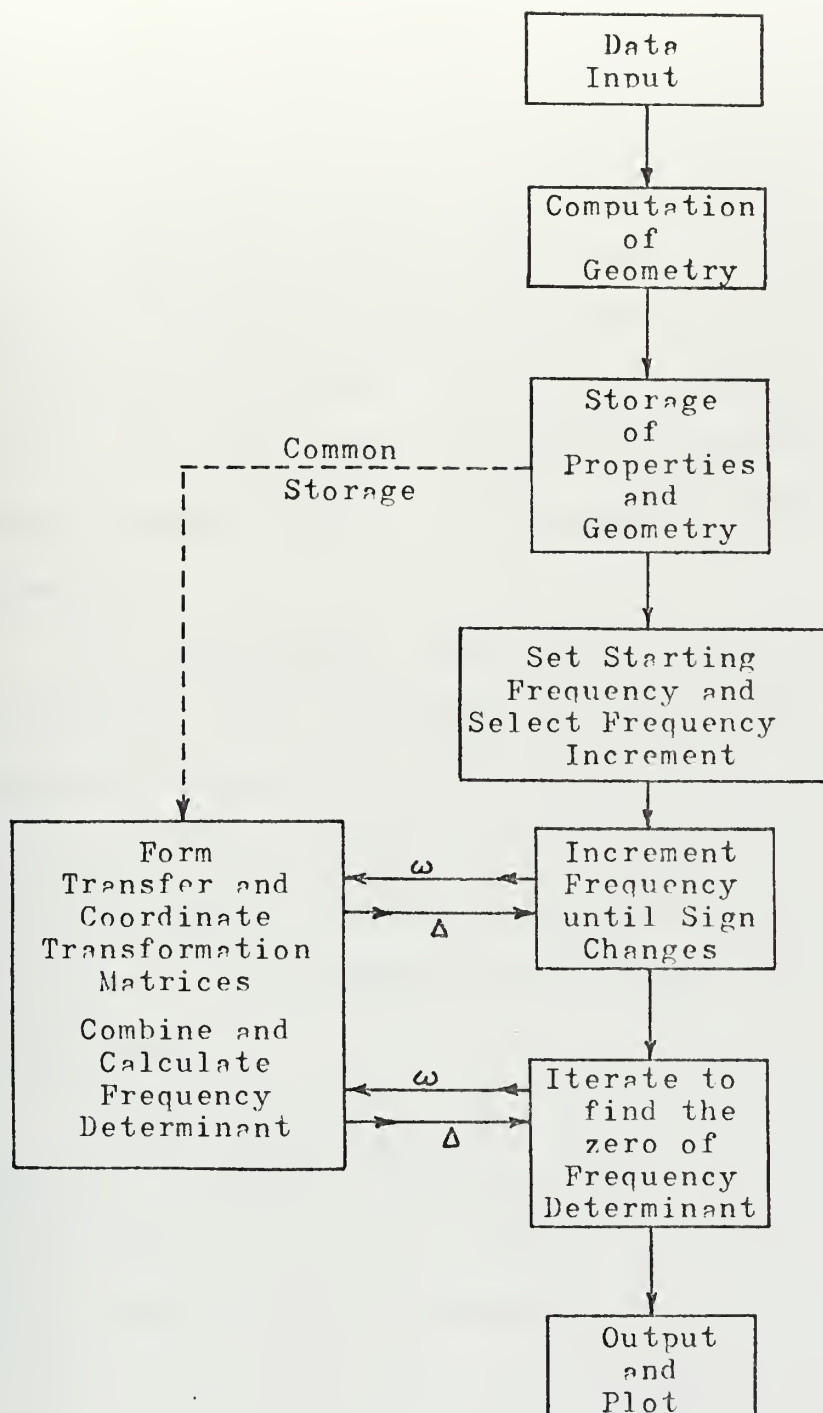


Figure C.1: Simplified Flow Diagram for Main Program

manner:

$0 < L/D \leq 1$	1 SUBSECTION
$1 < L/D \leq 3$	2 SUBSECTIONS
$3 < L/D \leq 6$	3 SUBSECTIONS

If L/D is greater than six, the number of subsections is computed as twice the number of mode frequencies sought, up to a maximum of twelve. The user may, however, choose to override the number of subsections as determined by the program through the use of the parameter MCS described in Appendix E.

As the lengths, subsections, and rotation angles are computed, they are stored, together with intensive and extensive properties, in an array. This array is passed via common storage (dotted line in Fig. C.1) to function subroutine FRDET (FRequency DETerminant) which later forms and evaluates the frequency determinant for a given value of frequency.

When the geometry for the entire system has been computed, frequency iteration begins. The starting frequency is arbitrarily set at 0.1 radians per second. To compute the starting frequency increment, the program constructs a synthetic straight pipe, with a length equal to the total length of the main member, and having outside diameter, wall thickness, and intensive properties equal in magnitude to a weighted average of the main member sections. One sixth of the fundamental frequency of this synthetic pipe

with fixed ends is taken as the starting frequency increment. The system transfer matrix and frequency determinant are formed by function subroutine FRDET for each new frequency. When a sign change is detected in the frequency determinant, an iteration process using Mueller's method of successive bisections and inverse parabolic interpolation is used for convergence to the natural frequency. The convergence acceptability criterion is taken as the starting frequency increment divided by ten thousand.

After a root has been located, the search begins anew with the starting frequency equal to the value of the natural frequency plus one radian per second. When the second mode frequency has been located, the frequency increment is changed to one tenth the difference between the previous two natural frequencies.

The search process continues until the specified number of frequencies has been found or until the significant figure capacity of the computer has been exceeded. Then the mode frequencies are printed and the pairs of frequencies and frequency determinant values used in the search are sorted and plotted.

b. Subroutines

The program utilizes sixteen subroutines in the process of computation. A brief description follows:

ANGLE	Computes the sign and the angular separation of two sets of unit vectors which have a common axis.
-------	--

COORDX	Forms an $n \times n$ (n is some multiple of 3 and ≤ 12) coordinate transformation matrix for a rotation about a local x-axis.
COORDZ	Forms an $n \times n$ coordinate transformation matrix for a rotation about a local z axis.
CURMAT	Constructs the field matrix for a curved section of pipe.
DETER	Computes the value of an $n \times n$ determinant; used to evaluate the frequency determinant.
DROOT	Performs successive bisections and inverse parabolic interpolation to locate the zeros of the frequency determinant after a sign change has been detected.
INVERT	Inverts the matrix R_1 to be used in computing the branch ₁ point matrix.
MATMUL	Multiplies two matrices together. One of the matrices must be square.
POINT	Constructs a point matrix for a curved section of pipe.
PRPLOT	Takes the array of frequencies and frequency determinant values calculated during iteration and plots them length-wise with the printer. Scaling is done automatically and the length of plot may be changed to suit the user.
SORT	Sorts the array of frequencies and frequency determinant values from iteration for plotting on the printer.
STATEM	Constructs the state matrix for the appropriate fixed, free, or ball joint boundary conditions.
STRMAT	Constructs the transfer matrix for a straight section of pipe.
UVEC	Computes a set of orthogonal unit vectors from two pipe section vectors which represent a directional change in the piping.

WSTART	Computes the value of the frequency determinant for a previously constructed synthetic pipe equal in length to the main member and having weighted average values of the properties of the main member sections. This frequency value is used in selecting the starting frequency increment.
FRDET	For a particular value of frequency, this subroutine assembles the state matrices, transfer matrices for each section of pipe, and the coordinate transformation matrices. It then multiplies them together in sequence and calculates the frequency determinant.

An interesting phenomenon occurs in the formation of the frequency determinant for a single straight pipe. Because of the symmetry of a pipe in the y and z directions, the two flexural submatrices in the straight pipe transfer matrix are identical, which causes the frequency determinant to maintain the same sign on either side of the flexural natural frequencies. This isolated case is handled in subroutine STRMAT through the use of the straight section test discriminant and appropriate exclusion of one of the flexural submatrices.

3. Remarks on Instructions for Program Use

a. General Remarks

Detailed instructions for program use are contained in the listing of Appendix E. It was intended for these instructions to be part of the program in order that any user, given only the program listing or the computer card deck, could employ the VIBREL package with no prior knowledge of transfer matrices and without perusal of this thesis.

Some clarification of the instructions for program use is justified for cases where properties or geometry change within a piping bend. First, consider the simple section of piping shown in Fig. C.2. Assuming that no change

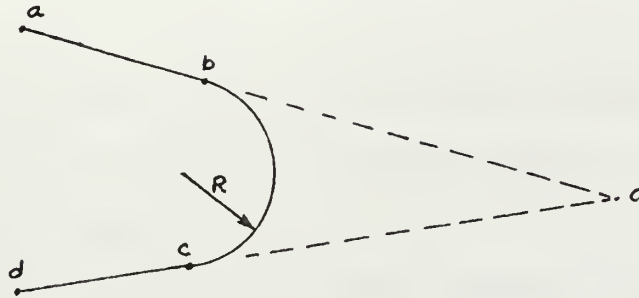


Figure C.2: Piping Bend with No Change in Properties

in cross section or properties occurs between points a and d , the only points which must be listed as working points for program input are a , o , and d . Using these points and the radius of curvature, program VIBREL will calculate lengths ab and cd as well as the arc length and angle of the bend.

Now consider the section of piping of Fig. C.3 where the cross section or properties change at point c but remain

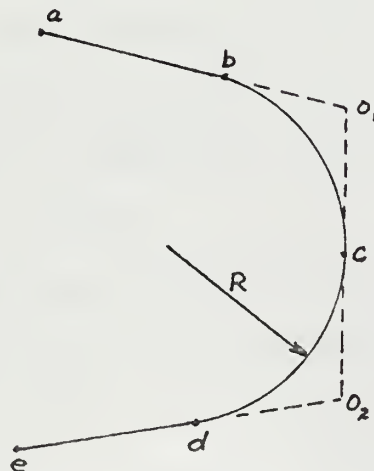


Figure C.3: Piping Bend with Change of Properties

constant between points a and c and between points c and e. In this instance points a, o_1 , o_2 , and e must be listed as working points for the program. VIBREL calculates straight lengths ab and de as well as the information required for curved lengths bc and cd. It should be obvious from this example that for the general case of two curved sections occurring sequentially that the intersection point (point c in Fig. C.3) need not be listed.

If the geometry input is such that negative lengths are computed, an error message is printed. One half inch leeway is allowed for negative lengths to take into account any errors which may occur in measurement.

b. Program Flexibilities

(1) Approximating Small Piping Accessories

Small valves, flanges, couplings and other piping accessories whose center of mass is relatively close to the centroidal axis of the pipe can be approximated using a straight section of pipe. The length, mass density, diameter and wall thickness can be artificially varied to give the mass of the item. The resulting variation in stiffness between an actual flange and artificial pipe, for example, usually will not have a significant effect on the results because of its short length.

(2) Approximating Alternative End Conditions

When the ideal boundary conditions are applied to the state vector on the end of a system, it must have six zero and six non-zero elements. Actual field conditions

may dictate that this is not the case because of flexible mountings or connection to machinery. A good approximation in these circumstances may be made by attaching one end of an artificial straight pipe to the system piping keeping its other end fixed. The mass of the artificial pipe should be small with the extensive properties and elastic modulus chosen to approximate the stiffness of the mounting.

The program can then use the ideal boundary conditions with little loss of accuracy from the real situation.

(3) Program Modifications

In some instances, a user may desire to change such quantities as frequency increment or input format to suit his particular needs. For this reason, Table C.1 lists the more important program variables and what card or cards to make these changes to.

TABLE C.1

Modification	Card Numbers	Variables Affected
Maximum number of points for print plot	2890	x(400), y(400)
Maximum number of piping sections in analysis	2910	AA(<u>100</u> ,10)
Maximum number of branches in analysis	2920	NB(50)
Input format	3210-50	
Output format (Input Data)	3390-420	
Output format (Properties and Geometry)	7750-60	
Output format (Natural Frequencies)	8740	

TABLE C.1 (Continued)

Modification	Card Number	Variables Affected
Starting frequency increment	8650	WINCR
Starting frequency	8760	W1
Convergence acceptability criterion	9120	EPS
Maximum iterations to convergence in DROOT	3670	NIT
Starting Frequency after location of natural frequency	9330	W1
Frequency increment after second mode	9320	WINCR
Number of lines of print plot	14730	RY

APPENDIX D

PROGRAM FLOW DIAGRAMS

1. General Remarks

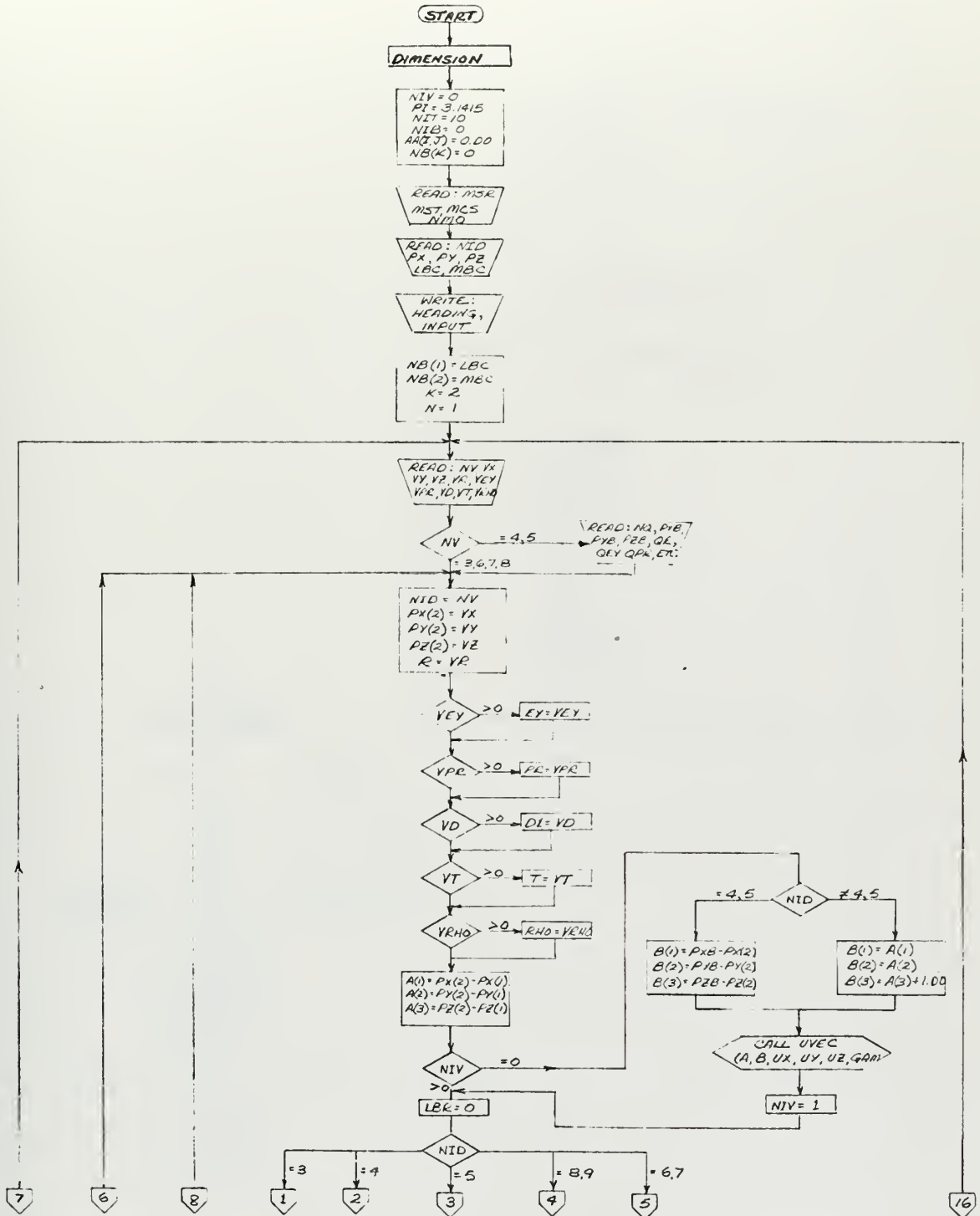
Flow diagrams are included for the main program and function subroutine FRDET. Subroutines DETER, DROOT, INVERT, and PRPLOT are standard package subroutines with minor modifications made to accommodate them to program VIBREL. Since their equivalent or an alternative routine may be substituted for them, flow diagrams are not included. Adequate comment statements are, however, contained in these subroutines to define their structure. Other subroutines for which no flow diagrams appear, incorporate, at most, two decisions in their structure.

2. Main Program Flow Diagram

Pages 69 through 76.

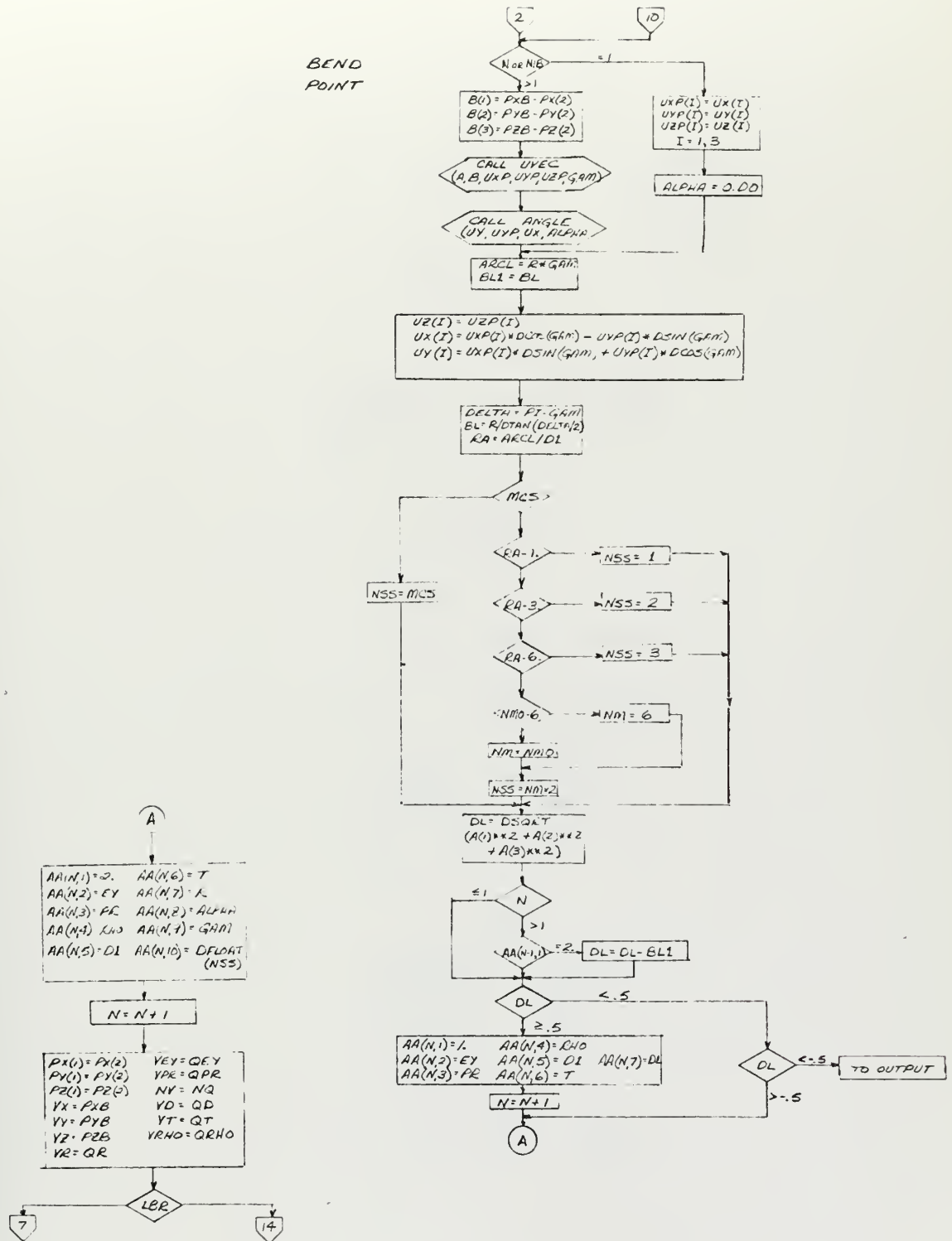
3. Function Subroutine FRDET Flow Diagram

Pages 77 through 79.

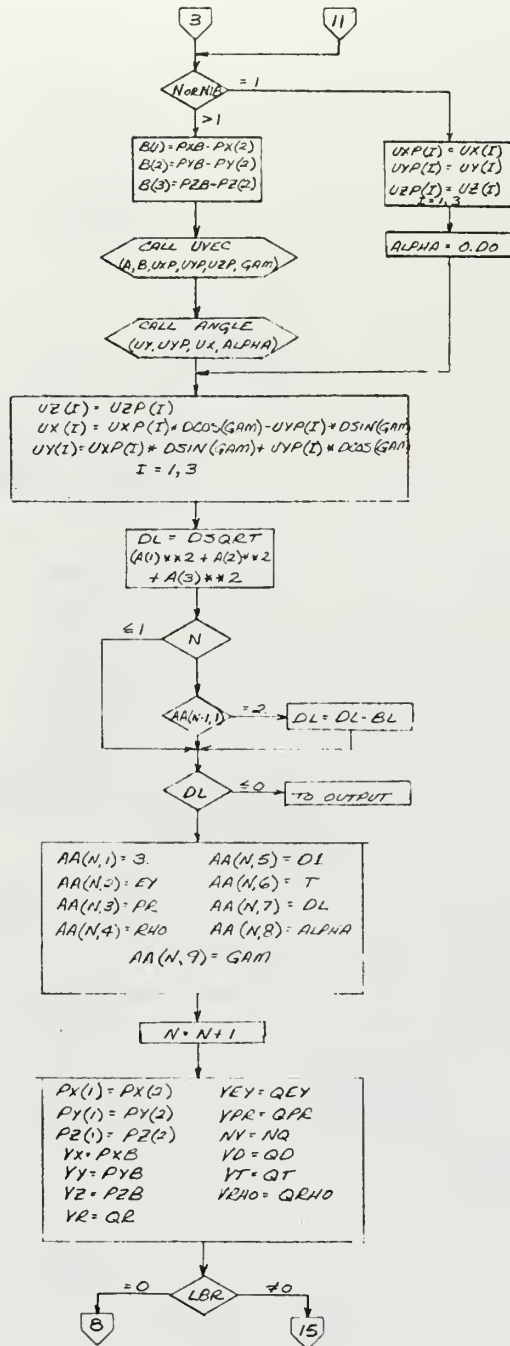




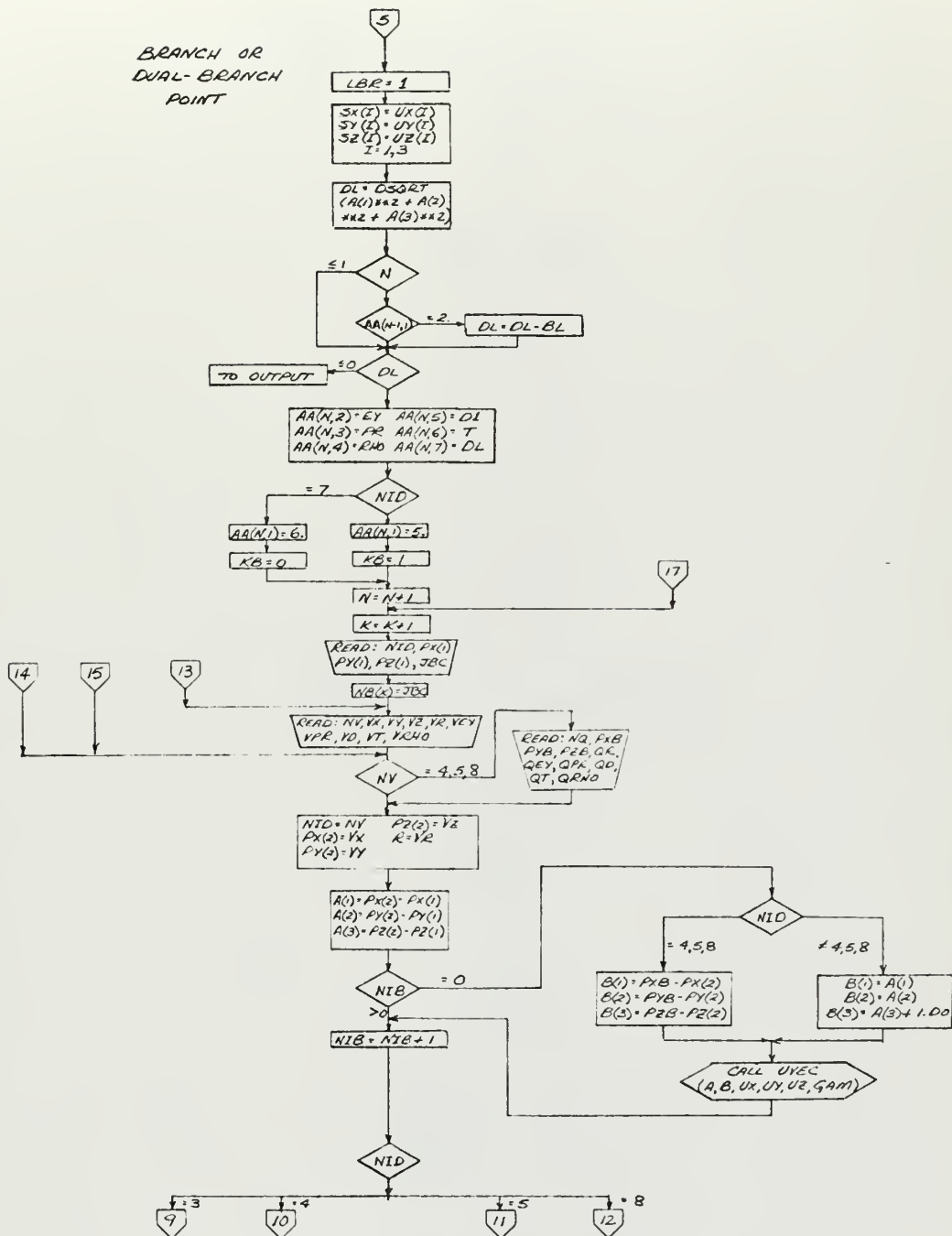
BEND
POINT



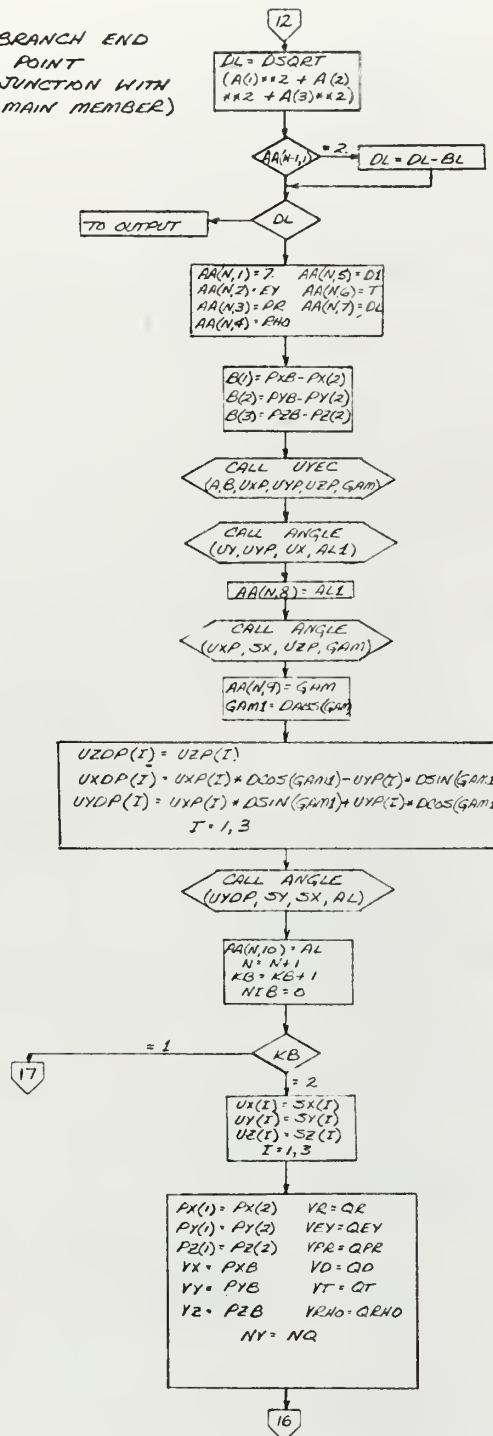
CORNER
POINT

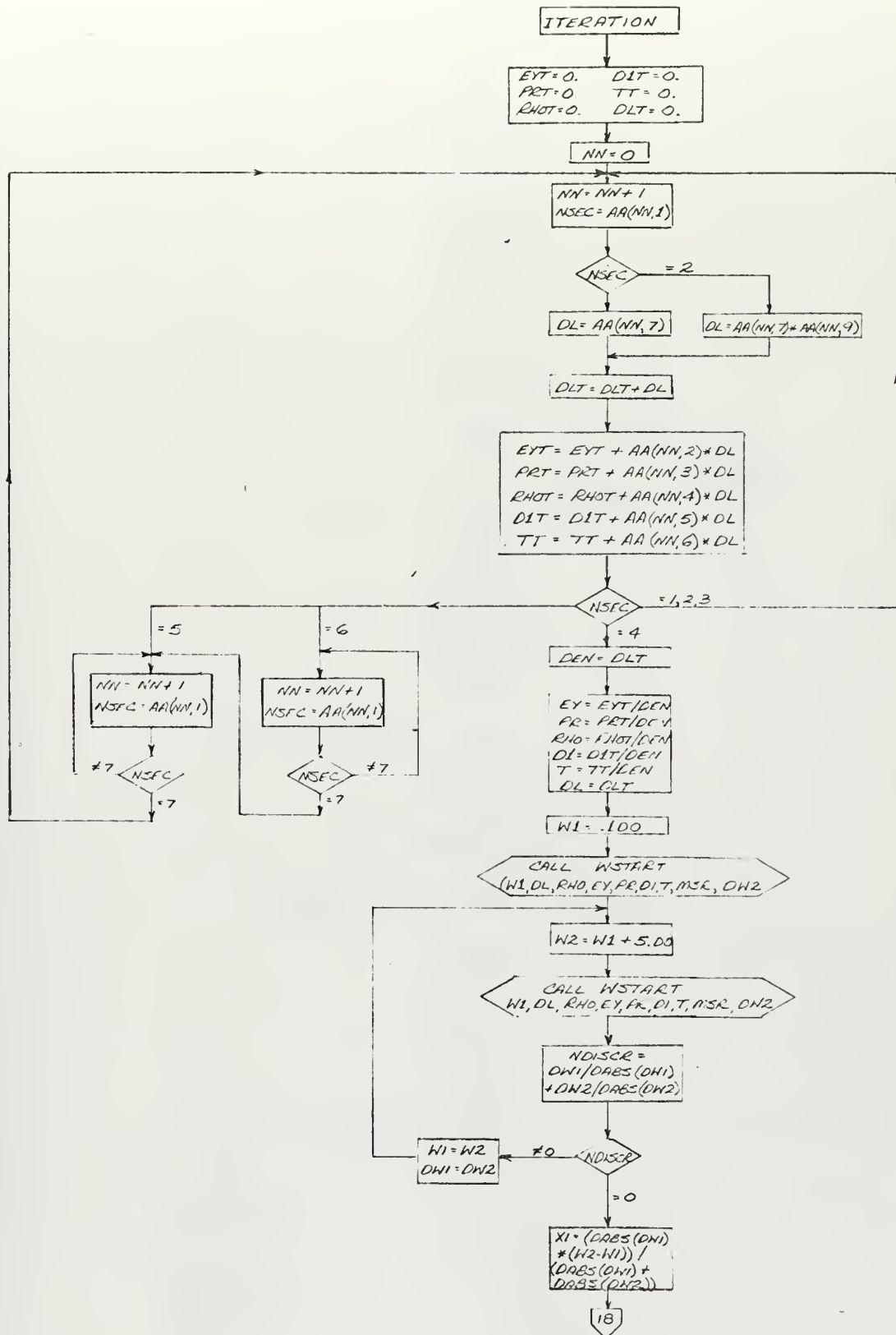


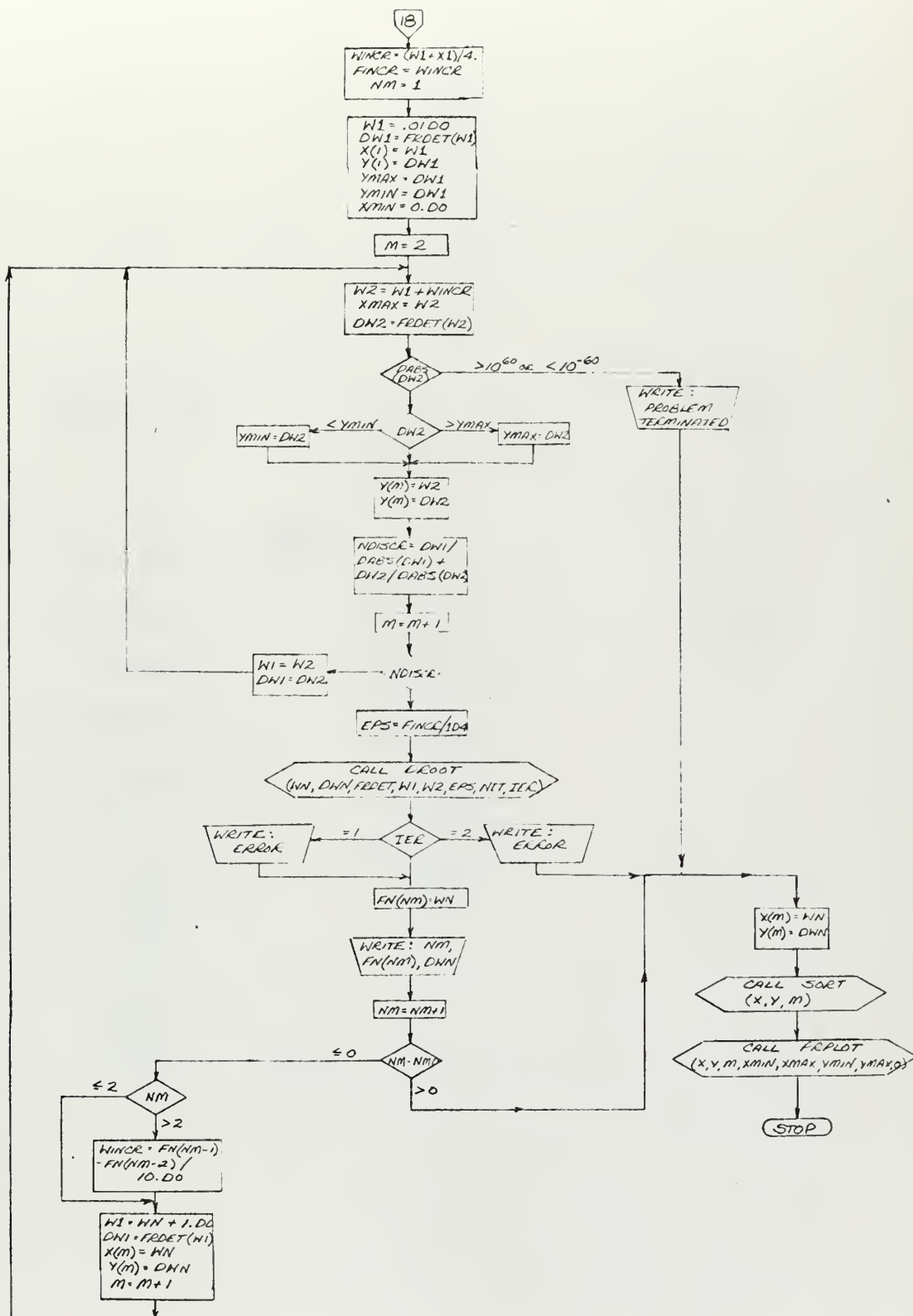
BRANCH OR
DUAL-BRANCH
POINT

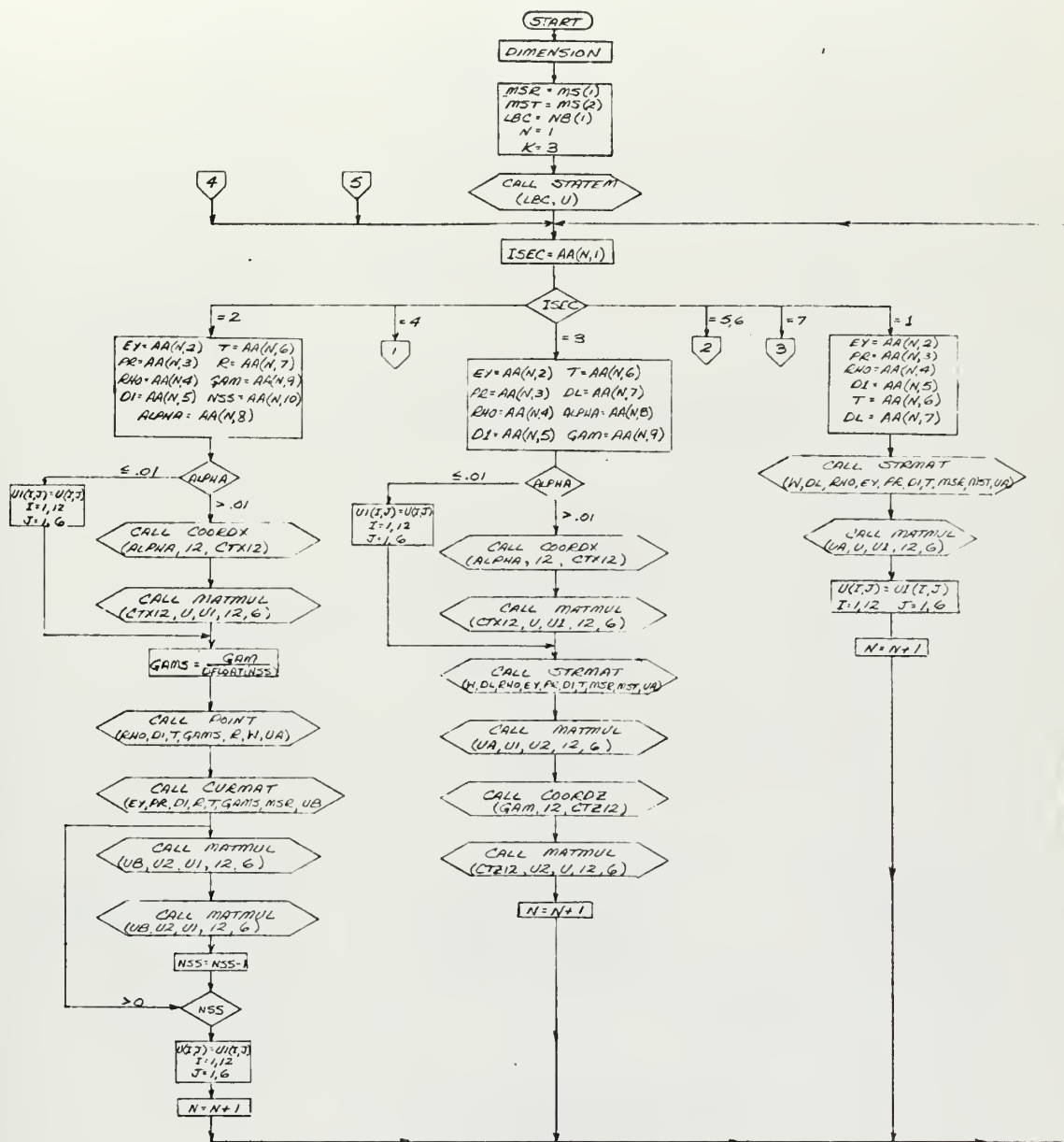


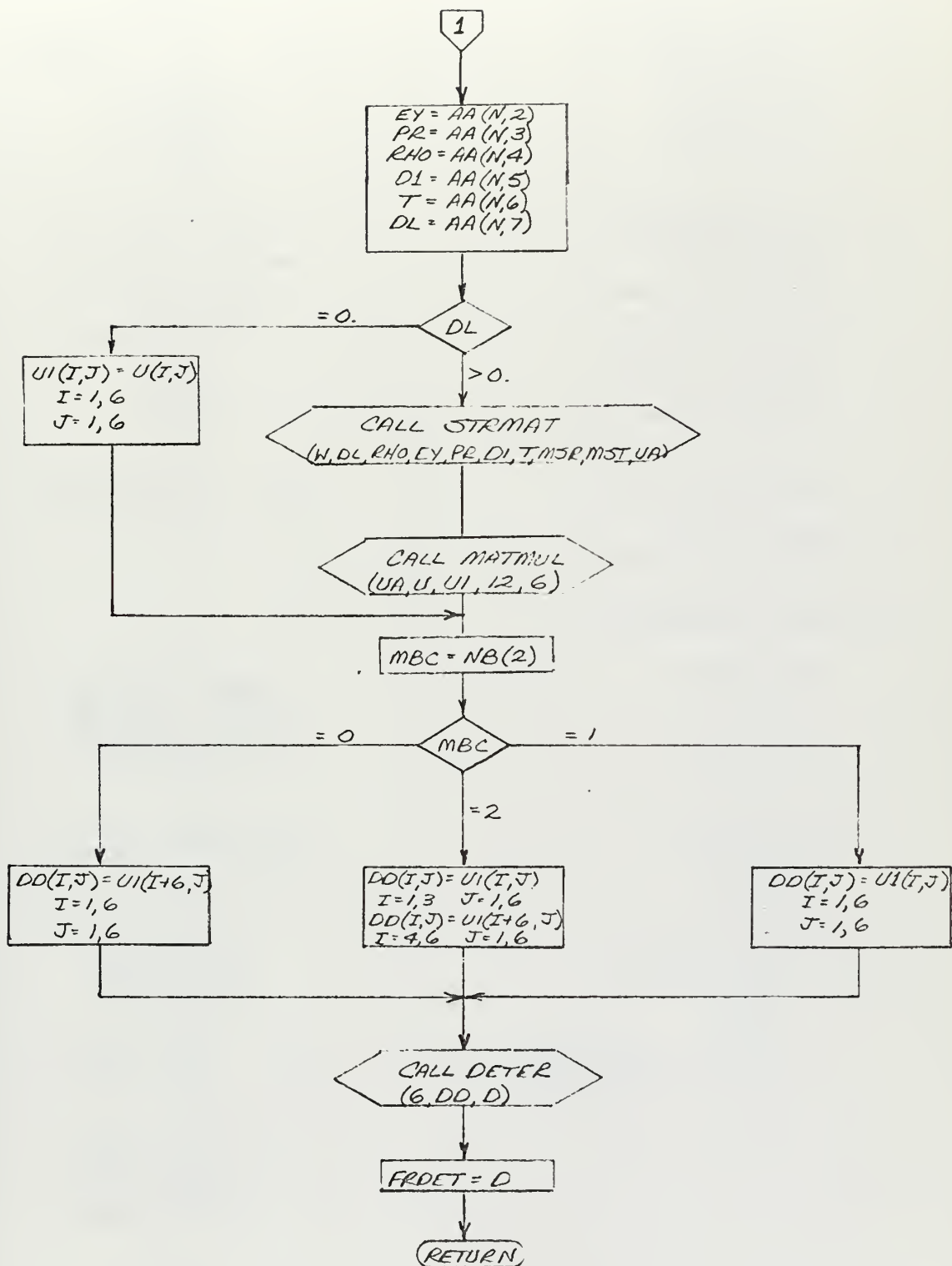
BRANCH END
POINT
(JUNCTION WITH
MAIN MEMBER)

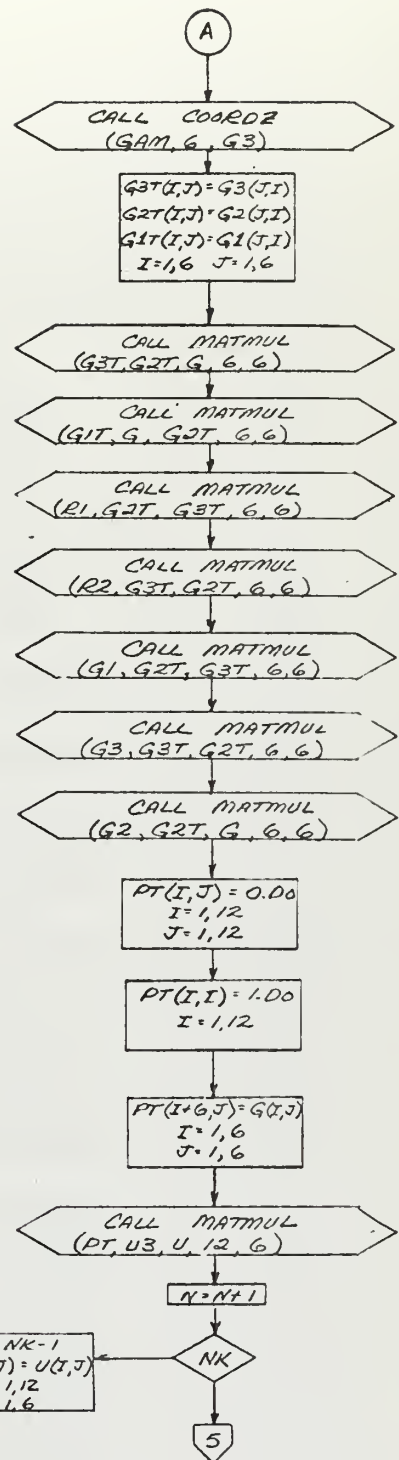
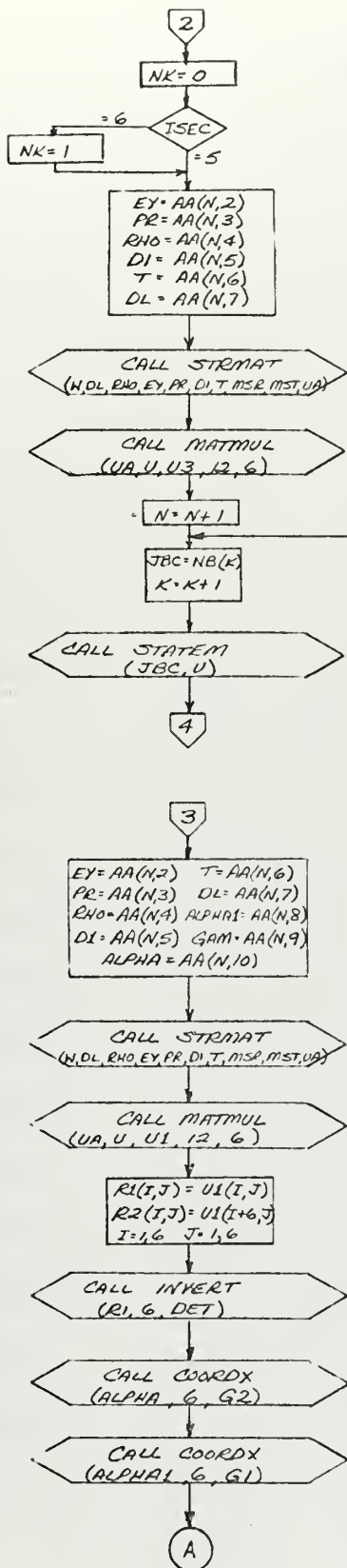












APPENDIX E

PROGRAM LISTING

1. Index

SECTION	SEQUENCE NUMBERS	PAGE
Instructions for Use	10-2810	81
Main Program	2820-9650	86
Subroutines		
ANGLE	9660-9860	101
COORDX	9870-10250	101
COORDZ	10260-10640	102
CURMAT	10650-11700	103
DETER	11710-12060	105
DROOT	12070-13200	106
INVERT	13210-13980	108
MATMUL	13990-14130	110
POINT	14140-14570	110
PRPLOT	14580-15910	111
SORT	15920-16150	114
STATEM	16160-16370	115
STRMAT	16380-17370	115
UVEC	17380-17630	117
WSTART	17640-17830	118
FRDET	17840-20090	118


```

00000010
00000020
00000030
00000040
00000050
00000060
00000070
00000080
00000090
00000100
00000110
00000120
00000130
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00000340
00000350
00000360
00000370
00000380
00000390
00000400
00000410
00000420
00000430
00000440
00000450
00000460
00000470
00000480

```

 INSTRUCTIONS FOR
 THE USE OF PROGRAM VIBREL

REFERENCE: RUDOLF, C.D., "3-D PIPING VIBRATIONS VIA TRANSFER
 MATRICES", U.S. NAVAL POSTGRADUATE SCHOOL THESIS, DEC, 1971.

A. GENERAL REMARKS

PROGRAM VIBREL MAY BE USED IN THE ANALYSIS OF A
 THREE DIMENSIONAL PIPING SYSTEM HAVING ARBITRARY CGN-
 FIGURATION. THE CAPABILITIES AND LIMITATIONS ARE:

1. THE PIPING SYSTEM MUST BE COMPOSED OF A MAIN MEMBER AND
 ANY NUMBER OF SIMPLE BRANCHES. THE MAIN MEMBER IS DESIG-
 NATED AS THE LONGEST CONTINUOUS RUN OF PIPE FROM WHICH ALL
 BRANCHES EMANATE. A MAXIMUM OF TWO BRANCHES CAN JOIN THE
 MAIN MEMBER AT ANY ONE POINT AND THERE CAN BE NO BRANCHES
 EMANATING FROM OTHER BRANCHES. THE SYSTEM CANNOT CONTAIN
 LOOPS.
2. THERE MAY BE NO CURVATURE OF THE PIPING OR ABRUPT CHANGE
 IN DIRECTION OF THE MAIN MEMBER AT A BRANCH JUNCTION
 POINT.
3. PIPING HANGERS CANNOT BE HANDLED DIRECTLY BY PROGRAM
 VIBREL. A SPRING HANGER MAY BE APPROXIMATED AS A VERY
 SMALL PIPE OF A HIGH MODULUS MATERIAL, INCLUDED IN
 THE SYSTEM AS A BRANCH EMANATING FROM THE MAIN MEMBER.
 RELATIVELY SMALL PIPING ACCESSORIES SUCH AS VALVES CAN BE
 APPROXIMATED BY VARYING THE PROPERTIES OF A SMALL STRAIGHT
 SECTION OF PIPE.
4. PIPE CROSS SECTION AND PROPERTIES MAY VARY FROM SECTION TO
 SECTION BUT MAY NOT VARY WITHIN A SECTION.
5. SHEAR DEFLECTION AND ROTARY INERTIA MAY BE INCLUDED IN
 THE ANALYSIS AT THE DISCRETION OF THE USER.
6. THE PROGRAM IS CAPABLE OF ANALYZING ONE OR MORE SYSTEMS
 DURING A SINGLE COMPUTER RUN.

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B.COORDINATE SYSTEM AND WORKING POINTS

THE PIPING SYSTEM GEOMETRY IS DETERMINED BY WORKING POINTS, SET OF THE COORDINATES OF WHICH ARE MEASURED FROM THE ORIGIN OF A SET OF MUTUALLY ORTHOGONAL X,Y, AND Z AXES. THE ORIGIN IS LOCATED ARBITRARILY AND REMAINS FIXED IN SPACE. ALL WORKING POINT COORDINATES ARE MEASURED WITH THE PIPING SYSTEM IN ITS QUIESCENT CONFIGURATION.

THERE IS PRESENTLY NO PROVISION FOR ASSOCIATING AN IDENTIFYING NUMBER OR SYMBOL WITH ANY POINTS OF PARTICULAR SIGNIFICANCE IN THE SYSTEM.

HOWEVER, THE INPUT EMPLOYS EIGHT TYPES OF WORKING POINTS AS INDICATED IN THE FOLLOWING TABLE. A NINTH TYPE IS REQUIRED FOR INTERNAL PROGRAM PURPOSES IN THE CASE OF DUAL BRANCHES, I.E., TWO BRANCHES EMANATING FROM THE SAME POINT OF THE MAIN MEMBER.

POINT TYPE CODE	DEFINITION
1	STARTING POINT(MAIN)- FIRST POINT OF THE SYSTEM MAIN MEMBER, HENCEFORTH REFERRED TO AS THE LEFT END. IT DETERMINES A SYSTEM BOUNDARY.
2	STARTING POINT(BRANCH)- FIRST POINT OF A BRANCH FARTHEST FROM THE INTERSECTION WITH THE MAIN MEMBER. IT ALSO DETERMINES A SYSTEM BOUNDARY.
3	PROPERTY VARIATION POINT-A POINT OCCURRING IN A STRAIGHT RUN OF PIPE WHERE EITHER CROSS SECTION OR PROPERTIES CHANGE. CROSS SECTION OR PROPERTIES MAY ALSO CHANGE AT ANY OTHER WORKING POINT.
4	BEND POINT-THE POINT AT WHICH THE TANGENTS TO THE TWO ENDS OF A CURVED SECTION OF PIPE INTERSECT. THE BEND POINT AND RADIUS OF CURVATURE GEOMETRICALLY DESCRIBE A CURVED SECTION OF PIPE.
5	CORNER POINT-THE POINT AT WHICH A SECTION OF PIPE CHANGES DIRECTION WITH NO APPRECIABLE RADIUS OF CURVATURE.
6	BRANCH POINT-THE POINT IN THE MAIN MEMBER FROM WHICH A BRANCH EMANATES.

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7 DUAL-BRANCH POINT-THE POINT IN THE MAIN MEMBER
  FROM WHICH TWO BRANCHES EMANATE.
8 END POINT-EITHER THE FINAL POINT OF THE MAIN
  MEMBER, HENCEFORTH CALLED THE RIGHT END, OR THE
  POINT WHERE A BRANCH JOINS THE MAIN MEMBER. THE
  BRANCH END POINT COORDINATES ARE COINCIDENTIAL
  WITH THE CORRESPONDING BRANCH POINT(6) OR DUAL-
  BRANCH POINT(7).
9 DUAL-BRANCH REPEAT- THE MAIN MEMBER WORKING
  POINT IMMEDIATELY FOLLOWING A DUAL-BRANCH POINT.
  IT IS REPEATED IN THE CARD INPUT SEQUENCE
  TO CAUSE PROPER INTERNAL PROGRAM OPERATION.

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C. SEQUENCE OF POINTS FOR PROGRAM DATA INPUT

AFTER THE STARTING POINT OF THE MAIN MEMBER(1), WORKING POINTS ARE LISTED IN SEQUENCE WHILE TRAVERSING FROM THE LEFT END TOWARD THE RIGHT END UNTIL A BRANCH OR DUAL-BRANCH POINT IS ENCOUNTERED. AFTER A (SINGLE) BRANCH POINT(6) IS LISTED, THE SEQUENCE SKIPS TO THE BRANCH STARTING POINT(2) AND CONTINUES THROUGH THE BRANCH END POINT(8) BACK ONTO THE MAIN MEMBER AND PROCEEDS AGAIN TOWARD THE RIGHT END. THE BRANCH POINT(6) AND BRANCH END POINT(8) HAVE IDENTICAL COORDINATES.

THE SEQUENCE IS THE SAME FOR A DUAL-BRANCH EXCEPT THAT AFTER THE FIRST BRANCH END POINT(8), THE DUAL-BRANCH REPEAT(9), HAVING THE COORDINATES OF THE FOLLOWING MAIN MEMBER WORKING POINT, IS LISTED. THEN THE SEQUENCE PROCEEDS WITH THE STARTING POINT OF THE SECOND BRANCH AND CONTINUES AS IN THE CASE OF A SINGLE BRANCH. THE ORDER OF LISTING THE TWO BRANCHES IS INCONSEQUENTIAL.

SINCE PROGRAM VIBREL AUTOMATICALLY DEALS WITH THE GEOMETRY OF BENDS, PROPERLY LOCATING THE POINTS OF TANGENCY, IT IS NOT NECESSARY TO LIST WORKING POINTS FOR THE ENDS OF A CURVED SECTION OF PIPE UNLESS PROPERTIES OR CROSS SECTION CHANGE AT THE END POINTS OR SOMEWHERE WITHIN THE BEND ITSELF. FOR FURTHER CLARIFICATION SEE APPENDIX C OF THE REFERENCE THESIS.

D. PIPING PROPERTIES AND DATA READ-IN FORMAT

THE PROPERTIES ASSOCIATED WITH EACH WORKING POINT ARE THOSE OF THE SECTION OF PIPE BETWEEN IT AND THE PRECEDING WORKING POINT. IF MODULUS OF ELASTICITY, POISSON'S RATIO, OUTSIDE DIAMETER, WALL THICKNESS, OR WEIGHT DENSITY ARE REPEATED FROM WORKING POINT TO WOR-

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KING POINT, A BLANK OR A ZERO MAY BE SUBSTITUTED IN THE APPROPRIATE COLUMN FOR THE REPEATED VALUES UNTIL A CHANGE IS ENCOUNTERED. FOR WORKING POINTS OTHER THAN BEND POINTS THE RADIUS OF CURVATURE IS CONSIDERED TO BE ZERO FOR PROGRAM PURPOSES, ALTHOUGH FOR STRAIGHT PIPE IT WOULD BE INFINITE.

DATA FORMAT:

1)CARD A - USED ONLY ONCE

COL 1-2	NUMBER OF PIPING SYSTEMS TO BE ANALYZED	I2
---------	---	----

2)CARD B - USED ONCE FOR EACH SYSTEM ANALYZED

COL 1	SHEAR DEFLECTION/ROTARY INERTIA DISCRIMINANT 1- S.D./R.I.: CONSIDERED 0- S.D./R.I.: NOT CONSIDERED	I1
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COL 2	STRAIGHT SECTION TEST DISCRIMINANT 1- SYSTEM COMPOSED OF ONLY ONE STRAIGHT SECTION OF PIPE 0- ANY OTHER SYSTEM	I1
-------	--	----

COL 3-4	CURVED SUBSECTION OVERRIDE IF ZERO, PROGRAM DETERMINES NUMBER OF SUBSECTIONS FOR CURVED PIPE. ANY NUMBER OTHER THAN ZERO PRESCRIBES THAT MANY SUBSECTIONS FOR EACH BEND CALCULATION	I2
---------	--	----

COL 5-6	NUMBER OF MODE FREQUENCIES DESIRED IN THE ANALYSIS	I2
---------	--	----

3)SUCCEEDING CARDS- ONE FOR EACH WORKING POINT

A)STARTING POINT(MAIN)

COL 1	POINT TYPE CODE (SEE SECTION B)	I1
COL 2-7	X COORDINATE, INCHES	F6.1
COL 8-13	Y COORDINATE, INCHES	F6.1
COL 14-19	Z COORDINATE, INCHES	F6.1

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E. PROGRAM OUTPUT

COI 47-52 WEIGHT DENSITY, LB/CUBIC FT

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[illegible][illegible]


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1(3),UZP(3),SX(3),SY(3),SZ(3),UXDP(3),UYDP(3),UZDP(3),X(400),Y(400) 00002890
2,FN(30) 00002900
COMMON/VIB1/AA(100,10) 00002910
COMMON/VIB2/NB(50) 00002920
COMMON/VIB3/MS(2) 00002930
EXTERNAL FRDET 00002940
NPROB1=1 00002950
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READ NUMBER OF DIFFERENT SYSTEMS TO BE ANALYZED

READ(5,45)NPROB

READ SHEAR DEFLECTION/ROTARY INERTIA DISCRIMINANT, SINGLE STRAIGHT
SECTION DISCRIMINANT,CURVED SUBSECTION OVERRIDE,AND NUMBER
OF MODES SOUGHT

23 READ(5,50)MSR,MST,MCS,NMO
DO 30 I=1,100
DO 30 J=1,10
30 AA(I,J)=0.000
DO 40 I=1,10
40 NB(I)=0
MS(1)=MSR
MS(2)=MST

READ STARTING POINT COORDINATES

READ(5,60)NID,PX(1),PY(1),PZ(1),LBC,MBC

WRITE TOP PORTION OF OUTPUT FORMAT AND INPUT DATA

WRITE(6,70)
FORMAT(I2)
50 FORMAT(2I1,2I2)
60 FORMAT(I1,3F6.1,2I1)
61 FORMAT(I1,3F6.1,I1)
62 FORMAT(I1,3F6.1,F5.1,E8.1,F3.2,F5.2,F6.4,F6.1)
70 1*****
2BREL,2IX,C.D.RUDOLF,10X,/,5X,/,6X,/,71X,/,6X,/,15X,PROGRAM V100003270
3*****
76 FORMAT(6X,INPUT DATA:)
77 FORMAT(/,6X,PROGRAM VIBREL COMPUTES THE NATURAL FREQUENCIES OF V100003310
1BRATION,/,9X,OF ANY RANDOMLY ARRANGED 3-DIMENSIONAL PIPING SYSTEM0003320
2M,/,6X,PIPING HANGERS AND PROJECTIVE COMPLEXITIES ARE NOT INCLUD00003330
3ED;)
78 FORMAT(/,/,14X,X,6X,Y,6X,Z,5X,RAD OF,2X,ELASTIC,3X,POIS00003350
1SONS,10X,WALL,14X,LEFT BC,4X,RIGHT BC,/,7X,ID,3X,COORD,00003360

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22X,'COORD',2X,'COORD',4X,'CURV',3X,'MODULUS',4X,'RATIO',5X,'O.D.',00003370
32X,'THICK',3X,'WT/VOL',4X,'CODE',8X,'CODE',//00003380
81 FORMAT(7X,I1,3X,F6.1,1X,F6.1,1X,F6.1,1X,I1)00003390
82 FORMAT(7X,I1,3X,F6.1,1X,F6.1,1X,F6.1,1X,I1)00003400
83 FORMAT(7X,I1,3X,F6.1,1X,F6.1,1X,F6.1,1X,I1)00003410
1 2,2X,F6.4,2X,F6.1)00003420
85 FORMAT(//,6X,*****00003430
1 *****00003440
WRITE(6,77)00003450
WRITE(6,85)00003460
WRITE(6,84)NPROB100003470
WRITE(6,85)00003480
84 FORMAT(6X,'PROBLEM #',I2)00003490
IF(MSR.EQ.0)GO TO 8700003500
WRITE(6,86)00003510
86 FORMAT(6X,'THE EFFECTS OF SHEAR DEFLECTION AND ROTARY INERTIA ARE00003520
1 CONSIDERED IN THIS PROBLEM')00003530
GO TO 8900003540
87 WRITE(6,88)00003550
88 FORMAT(6X,'THE EFFECTS OF SHEAR DEFLECTION AND ROTARY INERTIA ARE00003560
1 NOT CONSIDERED IN THIS PROBLEM')00003570
89 WRITE(6,85)00003580
WRITE(6,76)00003590
WRITE(6,78)00003600
WRITE(6,81)NID,PX(1),PY(1),PZ(1),LBC,MBC00003610
NIV=00003620
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SET MAXIMUM NUMBER OF ITERATIONS ALLOWABLE FOR BISECTION WHEN
CALCULATING THE NATURAL FREQUENCY IN SUBROUTINE DROOT

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NIT=15
NIB=0
PI=3.141592653589793
NB(1)=LBC
NB(2)=MBC
K=2
N=1

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C
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READ NEXT POINT AND SECTION PROPERTIES

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100 READ(5,62)NV,VX,VY,VZ,VR,VEY,VPR,VD,VT,VRHO
WRITE(6,83)NV,VX,VY,VZ,VR,VEY,VPR,VD,VT,VRHO
105 IF((NV.LT.4).OR.(NV.GT.5))GO TO 110

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IF BEND OR CORNER POINT READ THE FOLLOWING POINT

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READ(5,62)NQ,PXB,PYB,PZB,QR,QEY,QPR,QD,QT,QRHO
WRITE(6,83)NQ,PXB,PYB,PZB,QR,QEY,QPR,QD,QT,QRHO

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205 IF(DL-LT-0.0)GO TO 998
   IF(DL-EQ-0.0)GO TO 208
   AA(N,1)=1.00
   AA(N,2)=EY
   AA(N,3)=PR
   AA(N,4)=RHO
   AA(N,5)=DI
   AA(N,6)=T
   AA(N,7)=DL
   N=N+1
208 PX(1)=PX(2)
   PY(1)=PY(2)
   PZ(1)=PZ(2)
   IF(LBR-1)100,525,525
C$$$ COMPUTE BEND SECTION GEOMETRY
C$$$ IF((N-EQ-1).OR.(NIB.EQ-1))GO TO 302
300 B(1)=PXB-PX(2)
   B(2)=PYB-PY(2)
   B(3)=PZB-PZ(2)
   CALL UVEC(A,B,UXP,UY,UX,ALPHA)
   CALL ANGLE(UY,UY,UX,ALPHA)
   GO TO 303
302 DO 301 I=1,3
   UXP(I)=UX(I)
   UYP(I)=UY(I)
   UZP(I)=UZ(I)
   ALPHA=0.0DO
303 ARCL=R*GAM
   BL=BL
C
C REORIENT UNIT VECTORS
DO 304 I=1,3
  UZ(I)=UXP(I)*DCOS(GAM)-UYP(I)*DSIN(GAM)
  UX(I)=UXP(I)*DSIN(GAM)+UYP(I)*DCOS(GAM)
304 DELTA=PI-GAM
   BL=R/DTAN(DELT A/2.00)
   RA=ARCL/DI
C
C DETERMINE THE NUMBER OF SUBSECTIONS OF THE BEND
   IF(MCS.EQ-0)GO TO 305
   NSS=MCS
   GO TO 335
305 IF(RA-1.00)306,306,310

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306 NSS=1
GO TO 335
310 IF(RA-3.D0)311,311,315
311 NSS=2
GO TO 335
315 IF(RA-6.D0)316,316,320
316 NSS=3
GO TO 335
320 IF(NMO-6)325,325,326
325 NM=NMO
GO TO 330
326 NM=6

C      MAXIMUM NUMBER OF SUBSECTIONS IS 12
C
C      330 NSS=NM*2
C
C      COMPUTE LENGTH OF STRAIGHT SECTION PRIOR TO BEND
C
335 DL=DSQRT(A(1)**2+A(2)**2+A(3)**2)-BL
IF(N.LE.1)GO TO 337
IN=N-1
IF((AA(IN,1).GT.1.9D0).AND.(AA(IN,1).LT.2.1D0))DL=DL-BL1
337 IF(DL.LT.0.5D0)GO TO 340
AA(N,1)=1.D0
AA(N,2)=EY
AA(N,3)=PR
AA(N,4)=RHO
AA(N,5)=D1
AA(N,6)=T
AA(N,7)=DL
N=N+1
GO TO 345

C      IF LENGTH COMPUTED IS NEGATIVE PRINT AN ERROR MESSAGE
C
C
340 IF(DL.LT.-.5D0)GO TO 998
345 AA(N,1)=2.D0
AA(N,2)=EY
AA(N,3)=PR
AA(N,4)=RHO
AA(N,5)=D1
AA(N,6)=T
AA(N,7)=R
AA(N,8)=ALPHA
AA(N,9)=GAM
AA(N,10)=DFLOAT(NSS)
N=N+1

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355 PX(1)=PX(2)
    PY(1)=PY(2)
    PZ(1)=PZ(2)
    NV=NQ
    VX=PX
    VY=PY
    VZ=PZ
    VR=QR
    VEY=QEY
    VPR=QPR
    VD=QD
    VT=QT
    VRHO=QRHO
    IF(LBR-1)105,530,530
C$$$
C    COMPUTE CORNER SECTION GEOMETRY
C$$$
400 IF((N.EQ.1).OR.(NIB.EQ.1))GO TO 402
    B(1)=PX
    B(2)=PY
    B(3)=PZ
    CALL UVEC(A,B,UXP,UY,UX,ALPHA)
    GO TO 403
402 DO 401 I=1,3
    UXP(I)=UX(I)
    UYP(I)=UY(I)
    UZP(I)=UZ(I)
    ALPHA=0.0DO
C
C    REORIENT UNIT VECTORS
C
403 DO 410 I=1,3
    UZ(I)=UZP(I)
    UX(I)=UXP(I)*DCOS(GAM)-UYP(I)*DSIN(GAM)
    UY(I)=UXP(I)*DSIN(GAM)+UYP(I)*DCOS(GAM)
410 UY(I)=UXP(I)*DSIN(GAM)+UYP(I)*DCOS(GAM)
C
C    COMPUTE PRE-CORNER STRAIGHT SECTION LENGTH
C
411 DL=DSQRT(A(1)**2+A(2)**2+A(3)**2)
    IF(N.LE.1)GO TO 415
    IN=N-1
    IF((AA(IN,1).GT.1.9DO).AND.(AA(IN,1).LT.2.1DO))DL=DL-BL
C
C    IF LENGTH COMPUTED IS NEGATIVE PRINT AN ERROR MESSAGE
C
415 IF(DL.LE.0.0DO)GO TO 998
    AA(N,1)=3.0DO

```



```

IF(NID.EQ.7)GO TO 505
AA(N,1)=5.D0
KB=1
GO TO 510
505 AA(N,1)=6.D0
KB=0
510 N=N+1
520 K=K+1
C
C READ BRANCH START POINT
C
READ(5,61)NID,PX(1),PY(1),PZ(1),JBC
WRITE(6,82)NID,PX(1),PY(1),PZ(1),JBC
NB(K)=JBC
C
C READ NEXT POINT AND SECTION PROPERTIES
C
525 READ(5,62)NV,VX,VY,VZ,VR,VEY,VPR,VD,VT,VRHO
WRITE(6,83)NV,VX,VY,VZ,VR,VEY,VPR,VD,VT,VRHO
530 IF((NV.NE.4).AND.(NV.NE.5).AND.(NV.NE.8))GO TO 535
C
C IF END,CORNER,OR BEND POINT READ FOLLOWING POINT
C
READ(5,62)NQ,PXB,PYB,PZB,QR,QEY,QPR,QD,QT,QRHO
WRITE(6,83)NQ,PXB,PYB,PZB,QR,QEY,QPR,QD,QT,QRHO
535 NID=NV
PX(2)=VX
PY(2)=VY
PZ(2)=VZ
R=VR
IF(VEY.GT.1.D-9)EY=VEY
IF(VPR.GT.1.D-9)PR=VPR
IF(VD.GT.1.D-9)D1=VD
IF(VT.GT.1.D-9)T=VT
IF(VRHO.GT.1.D-9)RHO=VRHO
A(1)=PX(2)-PX(1)
A(2)=PY(2)-PY(1)
A(3)=PZ(2)-PZ(1)
560 IF(NID.GT.0)GO TO 565
IF((NID.EQ.4).OR.(NID.EQ.5).OR.(NID.EQ.8))GO TO 561
C
C COMPUTE INITIAL UNIT VECTORS
C
B(1)=A(1)
B(2)=A(2)
B(3)=A(3)+1.D0
GO TO 563
561 B(1)=PXB-PX(2)

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C      B(2)=PYB-PY(2)
C      B(3)=PZB-PZ(2)
563 CALL UVEC(A,B,UX,UY,UZ,GAM)
565 NIB=NIB+1
C
C      COMPUTE APPROPRIATE SECTION GEOMETRY USING MAIN MEMBER ROUTINES
C
566 GO TO(999,999,200,300,400),NID
C
C      COMPUTE BRANCH JUNCTION SECTION GEOMETRY
C
570 DL=DSQRT(A(1)**2+A(2)**2+A(3)**2)
    IN=N-1
    IF((AA(IN,1).GT.1.9D0).AND.(AA(IN,1).LT.2.1D0))DL=DL-BL
571 IF(DL.LE.0.D0)GO TO 998
    AA(N,1)=7.D0
    AA(N,2)=EY
    AA(N,3)=PR
    AA(N,4)=RHO
    AA(N,5)=DI
    AA(N,6)=T
    AA(N,7)=DL
    B(1)=PXB-PX(2)
    B(2)=PYB-PY(2)
    B(3)=PZB-PZ(2)
    CALL UVEC(A,B,UXP,UYP,UZP,GAM)
C
C      COMPUTE INITIAL X-AXIS ROTATION ANGLE OF BRANCH UNIT VECTORS
C
    CALL ANGLE(UY,UYP,UX,AL1)
    AA(N,8)=AL1
C
C      COMPUTE Z-AXIS ROTATION TO ALIGN X COMPONENTS OF MAIN AND BRANCH
C      UNIT VECTORS
C
    CALL ANGLE(UXP,SX,UZP,GAM)
    AA(N,9)=GAM
    GAM1=DABS(GAM)
    DO 575 I=1,3
    UZDP(I)=UZP(I)
    UXDP(I)=UXP(I)*DCOS(GAM1)-UYP(I)*DSIN(GAM1)
    UYDP(I)=UXP(I)*DCOS(GAM1)+UYP(I)*DSIN(GAM1)
575
C      COMPUTE FINAL X-AXIS ROTATION TO ALIGN Y AND Z COMPONENTS OF MAIN
C      AND BRANCH UNIT VECTORS
C
    CALL ANGLE(UYDP,SY,SX,AL)
    AA(N,10)=AL

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00006990
00007000
00007010
00007020
00007030
00007040
00007050
00007060
00007070
00007080
00007090
00007100
00007110
00007120
00007130
00007140
00007150
00007160
00007170
00007180
00007190
00007200

```



```

C
C
C
N=N+1
KB=KB+1
NIB=0
IF DUAL-BRANCH REPEAT BRANCH GEOMETRY ROUTINES
IF(KB.EQ.1)GO TO 520
DO 580 I=1,3
UX(I)=SX(I)
UY(I)=SY(I)
UZ(I)=SZ(I)
PX(I)=PX(2)
PY(I)=PY(2)
PZ(I)=PZ(2)
VX=PX8
VY=PY8
VZ=PZ8
VR=QR
VEY=QEY
VPR=QPR
VD=QD
VT=QT
VRHO=QRHO
NV=NQ
GO TO 105
C$$$ COMPUTE END SECTION GEOMETRY
C$$$ DL=DSQRT(A(1)**2+A(2)**2+A(3)**2)
600 IF(N.LE.1)GO TO 604
IN=N-1
IF((AA(IN,1).GT.1.9D0).AND.(AA(IN,1).LT.2.1D0))DL=DL-BL
604 IF(DL.LT.-.5D0)GO TO 998
IF((DL.LT.0.1D0).AND.(DL.GE.-.5D0))DL=0.D0
AA(N,1)=4.D0
AA(N,2)=EY
AA(N,3)=PR
AA(N,4)=RHO
AA(N,5)=DI
AA(N,6)=T
AA(N,7)=DL
605 FORMAT(6X,'PROPERTIES AND GEOMETRY:',//)
C
C
C
PRINT ARRAY OF PROPERTIES AND GEOMETRY
610 WRITE(6,85)
WRITE(6,605)
WRITE(6,620)

```



```

620 FORMAT(6X,'SECT',8X,'ELASTIC',4X,'POISSONS',19X,'WALL',5X,'LENGTH',00007690
10R',24X,'NR OF SUBSEC',/,7X,'NR',3X,'ID',4X,'MODULUS',6X,'RATIO',300007700
2X,'WT/VOL',12X,'O.D.',3X,'THICK',4X,'RAD OF CURV',4X,'ALPHA',3X00007710
3,'GAMMA',4X,'OR ALPHA',/)
DO 630 I=1,N
630 WRITE(6,635)I,((AA(I,J),J=1,10)
635 FORMAT(7X,I2,3X,F3.1,2X,E9.3,5X,F3.2,6X,F6.1,2X,F5.2,3X,F6.4,4X,
1F8.3,5X,F6.3,4X,F6.3,5X,F6.3)
WRITE(6,85)
636 FORMAT(6X,'BOUNDARY CONDITION CODES:',/)
637 FORMAT(6X,'LEFT B.C. CODE=',1X,I1)
638 FORMAT(6X,'RIGHT B.C. CODE=',I1)
WRITE(6,636)
WRITE(6,637)NB(1)
WRITE(6,638)NB(2)
IF(K.LT.3)GO TO 1000
DO 640 I=3,K
JI=I-2
640 WRITE(6,642)JI,NB(I)
642 FORMAT(6X,'BRANCH #',I2,1X,'B.C. CODE=',1X,I1)
GO TO 1000

C THE FOLLOWING ARE THE ERROR MESSAGES
C
C
998 WRITE(6,995)N
GO TO 1001
999 WRITE(6,996)N
GO TO 1001
995 FORMAT(/,6X,'GEOMETRY ERROR-RECOMPUTE WORKING POINTS IN SECTION',
1I4)
996 FORMAT(/,6X,'POINT IDENTIFICATION NUMBER HAD A STARTING POINT COD
1E AT SECTION',I4)
1000 CONTINUE

C
C
C// THE SECOND PORTION OF THE MAIN PROGRAM COMPUTES SUCCESSIVE
C// VALUES OF THE FREQUENCY DETERMINANT AND ITERATES UNTIL A
C// NATURAL FREQUENCY OF THE PIPING SYSTEM IS LOCATED
C//
EYT=0.00
PRT=0.00
RHOT=0.00
DIT=0.00
TT=0.00
DLT=0.000
NN=0

```



```

C   CONSTRUCT SYNTHETIC STRAIGHT PIPE EQUAL IN LENGTH TO THE MAIN
C   MEMBER AND HAVING THE AVERAGE VALUES OF THE PROPERTIES OF THE
C   MAIN MEMBER SECTIONS
C
700  NN=NN+1
      NSEC=AA(NN,1)
      IF(NSEC.EQ.2)GO TO 705
      DL=AA(NN,7)
      GO TO 706
705  DL=AA(NN,7)*AA(NN,9)
706  DLT=DLT+DL
      EYT=EYT+AA(NN,2)*DL
      PRT=PRT+AA(NN,3)*DL
      RHOT=RHOT+AA(NN,4)*DL
      DIT=DIT+AA(NN,5)*DL
      TT=TT+AA(NN,6)*DL
707  GO TO(700,700,700,714,711,713),NSEC
711  NN=NN+1
      NSEC=AA(NN,1)
      IF(NSEC.EQ.7)GO TO 700
      GO TO 711
713  NN=NN+1
      NSEC=AA(NN,1)
      IF(NSEC.EQ.7)GO TO 711
      GO TO 713
714  DEN=DLT
      EY=EYT/DEN
      PR=PRT/DEN
      RHO=RHOT/DEN
      DL=DIT/DEN
      T=TT/DEN
      DL=DLT
C   FIND THE FUNDAMENTAL FREQUENCY OF THE SYNTHETIC STRAIGHT PIPE
C
7140 W1=.1D0
      CALL WSTART(W1,DL,RHO,EY,PR,D1,T,MSR,1,DW1)
715  W2=W1+.5D0
      CALL WSTART(W2,DL,RHO,EY,PR,D1,T,MSR,1,DW2)
      NDISCR=DW1/DABS(DW1)+DW2/DABS(DW2)
716  IF(NDISCR.EQ.0)GO TO 717
      W1=W2
      DW1=DW2
      GO TO 715
717  X1=(DABS(DW1)*((W2-W1))/((DABS(DW1)+DABS(DW2))))
C   THE STARTING FREQUENCY INCREMENT IS 1/6 OF THE 1ST SYNTHETIC
C   FREQUENCY
C
00008170
00008180
00008190
00008200
00008210
00008220
00008230
00008240
00008250
00008260
00008270
00008280
00008290
00008300
00008310
00008320
00008330
00008340
00008350
00008360
00008370
00008380
00008390
00008400
00008410
00008420
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00008470
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00008580
00008590
00008600
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00008620
00008630
00008640

```



```

C      WINCR=(W1+X1)/6.D0
      FINCR=WINCR
      NM=1
      WRITE(6,85)
      WRITE(6,712)
712  FORMAT(6X,'PIPING SYSTEM NATURAL FREQUENCIES:',//)
719  FORMAT(6X,'MODE',3X,'FREQUENCY(RAD/SEC)',3X,'FREQUENCY DETERMINANT',//)
720  FORMAT(7X,12,6X,F13.7,8X,E11.5)
      WRITE(6,719)
      W1=.10D0
      DW1=FRDET(W1)
718  X(1)=W1
      Y(1)=DW1
      YMAX=DW1
      YMIN=DW1
      XMIN=0.D0
      M=2
C      INCREMENT FREQUENCY UNTIL THE SIGN OF THE FREQUENCY DETERMINANT
      CHANGES
721  W2=W1+WINCR
      XMAX=W2
      DW2=FRDET(W2)
      DW2CHK=DABS(DW2)
      IF(DW2CHK.LT.1.D-60)GO TO 756
      IF(DW2CHK.GT.1.D+60)GO TO 756
      IF(DW2.LT.YMIN)YMIN=DW2
      IF(DW2.GT.YMAX)YMAX=DW2
722  X(M)=W2
      Y(M)=DW2
C      CHECK TO SEE IF THE SIGN OF THE FREQUENCY DETERMINANT HAS CHANGED
C
C      NDISCR=DW1/DABS(DW1)+DW2/DABS(DW2)
      M=M+1
      IF(NDISCR.EQ.0)GO TO 724
      IF(M.EQ.1000)GO TO 751
      W1=W2
      DW1=DW2
      GO TO 721
C      THE SOLUTION ACCEPTABILITY CRITERION IS EQUAL TO THE INITIAL
      FREQUENCY INCREMENT DIVIDED BY TEN THOUSAND
724  EPS=FINCR/1.D4

```



```

C      ONCE A ROOT HAS BEEN BRACKETED BY INCREMENTING FREQUENCY UNTIL
C      THE SIGN OF THE FREQUENCY DETERMINANT CHANGES, ITERATION IS
C      ACCOMPLISHED BY MUELLER'S METHOD UNTIL THE ROOT IS LOCATED
C
C      CALL DROOT(WN,DWN,FRDET,W1,W2,EPS,NIT,IER)
C      IF(IER-1)726,725,750
C      725 WRITE(6,727)
C      727 FORMAT('+',50X,'ITERATION OF THE NEXT FREQUENCY VALUE DID NOT CONV
C      726 FN(NM)=WN
C      WRITE(6,720)NM, FN(NM), DWN
C      NM=NM+1
C      IF(NM=NM0)730,730,735
C      730 IF(NM.LE.2)GO TO 733
C
C      AFTER THE SECOND MODE FREQUENCY IS FOUND CHANGE THE FREQUENCY
C      INCREMENT TO 1/10 THE DIFFERENCE BETWEEN THE LAST TWO MODES
C
C      WINCR=(FN(NM-1)-FN(NM-2))/10.D0
C      733 W1=WN+1.D0
C      DW1=FRDET(W1)
C      X(M)=WN
C      Y(M)=DWN
C      M=M+1
C      GO TO 721
C      735 X(M)=WN
C      Y(M)=DWN
C
C      SORT THE ARRAY OF VALUES OF FREQUENCIES AND FREQUENCY DETERMINANT
C      FOR PLOTTING ON THE PRINTER
C
C      740 CALL SORT(X,Y,M)
C      742 FORMAT('1',6X,'GRAPH OF THE VALUE OF FREQUENCY DETERMINANT VS. FRE
C      1 QUENCY',///)
C      WRITE(6,742)
C
C      PLOT THE ARRAY OF VALUES OF FREQUENCY DETERMINANT VS. FREQUENCY
C
C      CALL PRPLOT(X,Y,M,XMIN,XMAX,YMIN,YMAX,0)
C      GO TO 751
C      750 WRITE(6,755)
C      755 FORMAT('///,6X,'PROBLEM TERMINATED,DW1/DW2 HAD SAME SIGN GOING INTO
C      1 DROOT',///)
C      GO TO 740
C      756 WRITE(6,757)
C      757 FORMAT('///,6X,'PROBLEM TERMINATED, SIGNIFICANT FIGURE LIMITATION 00009580
C      1F THE COMPUTER WAS EXCEEDED',///)
C
00009130
00009140
00009150
00009160
00009170
00009180
00009190
00009200
00009210
00009220
00009230
00009240
00009250
00009260
00009270
00009280
00009290
00009300
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00009330
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00009360
00009370
00009380
00009390
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00009570
00009580
00009590
00009600

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00009610
00009620
00009630
00009640
00009650

```

```

GO TO 740
751 NPROB1=NPROB1+1
    IF(NPROB-NPROB1)1001,23,23
1001 STOP
    END

```

```

C*****
C***** SUBROUTINE ANGLE(UY,UYP,UX,ALPHA) *****
C***** SUBROUTINE ANGLE OF ROTATION *****
C***** COMPUTES THE ANGLE OF TWO SETS OF *****
C***** ABOUT A COMMON AXIS *****
C***** UNIT VECTORS *****
C***** *****
C***** UY IS INITIAL UNIT VECTOR *****
C***** UYP IS FINAL UNIT VECTOR AFTER ROTATION THRU ALPHA *****
C***** UX IS UNIT VECTOR USED TO CHECK THE SIGN OF ALPHA *****
C***** IMPLICIT REAL*8(A-H,O-Z) *****
C***** DIMENSION UY(3),UYP(3),UX(3),C(3) *****
C***** DOTPR=(UY(1)*UYP(1)+UY(2)*UYP(2)+UY(3)*UYP(3)) *****
C***** ALPHA=DARCOS(DOTPR) *****
C***** C(1)=(UY(2)*UYP(3)-UYP(2)*UY(3)) *****
C***** C(2)=(UY(3)*UYP(1)-UYP(3)*UY(1)) *****
C***** C(3)=(UY(1)*UYP(2)-UYP(1)*UY(2)) *****
C***** TRIPR=(C(1)*UX(1)+C(2)*UX(2)+C(3)*UX(3)) *****
C***** IF(TRIPR.LT.0.0D0)ALPHA=-ALPHA *****
C***** RETURN *****
C***** END *****

```

```

00009660
00009670
00009680
00009690
00009700
00009710
00009720
00009730
00009740
00009750
00009760
00009770
00009780
00009790
00009800
00009810
00009820
00009830
00009840
00009850
00009860

```

```

C*****
C***** SUBROUTINE COORDX(AL,N,CTX) *****
C***** SUBROUTINE COORDX *****
C***** SUBROUTINE COORDINATE TRANSFORMATION *****
C***** COMPUTES COORDINATE TRANSFORMATION *****
C***** MATRIX FOR ROTATION ABOUT THE *****
C***** X-AXIS *****
C***** *****
C***** IMPLICIT REAL*8(A-H,O-Z) *****
C***** DIMENSION CX(12,12),CTX(N,N) *****
C***** DO 10 I=1,12 *****
C***** DO 10 J=1,12 *****
C***** DO 10 J=1,12 *****
C***** CX(I,J)=0.0D0 *****
C***** A=DCOS(AL) *****
C***** B=DSIN(AL) *****
C***** CX(1,1)=1.D0 *****
C***** CX(2,2)=A *****
C***** CX(2,3)=B *****
C***** CX(3,2)=-B *****

```

```

00009870
00009880
00009890
00009900
00009910
00009920
00009930
00009940
00009950
00009960
00009970
00009980
00009990
00010000
00010010
00010020
00010030
00010040

```



```

CX(3,3)=A
CX(4,4)=1.D0
CX(5,5)=A
CX(6,6)=B
CX(7,7)=A
CX(8,8)=A
CX(9,9)=B
CX(10,10)=1.D0
CX(11,11)=A
CX(12,12)=B
CX(13,13)=A
DO 20 I=1,N
DO 20 J=1,N
CX(I,J)=CX(I,J)
20 CONTINUE
RETURN
END

```

```

C** SUBROUTINE COORDZ(GAM,N,CTZ)
C** SUBROUTINE COORDZ
C** COMPUTES COORDINATE TRANSFORMATION
C** MATRIX FOR ROTATION ABOUT THE
C** Z-AXIS
C**
C** IMPLICIT REAL*8(A-H,O-Z)
C** DIMENSION CZ(12,12),CTZ(N,N)
C** DO 10 I=1,12
C** DO 10 J=1,12
C** CZ(I,J)=0.0D0
C** 10 A=DCOS(GAM)
C** B=DSIN(GAM)
C** CZ(1,1)=A
C** CZ(1,2)=B
C** CZ(2,1)=-B
C** CZ(2,2)=A
C** CZ(3,3)=1.0D0
C** CZ(4,4)=A
C** CZ(4,5)=B
C** CZ(5,4)=-B
C** CZ(5,5)=A
C** CZ(6,6)=1.0D0
C** CZ(7,7)=A

```


00010510
00010520
00010530
00010540
00010550
00010560
00010570
00010580
00010590
00010600
00010610
00010620
00010630
00010640

00010650
00010660
00010670
00010680
00010690
00010700
00010710
00010720
00010730
00010740
00010750
00010760
00010770
00010780
00010790
00010800
00010810
00010820
00010830
00010840
00010850
00010860
00010870
00010880
00010890
00010900
00010910
00010920
00010930
00010940
00010950
00010960

```

CZ(7,8)=B
CZ(8,7)=-B
CZ(8,8)=A
CZ(9,9)=1.D0
CZ(10,10)=A
CZ(10,11)=B
CZ(11,10)=-B
CZ(11,11)=A
CZ(12,12)=1.D0
DO 20 I=1,N
DO 20 J=1,N
20 CTZ(I,J)=CZ(I,J)
RETURN
END

SUBROUTINE CURMAT(EY,PR,DO,R,T,GAM,MSR,UCV)
C*****
C***** SUBROUTINE CURMAT
C***** CONSTRUCTS A FIELD MATRIX FOR A
C***** CURVED SECTION OF PIPE
C*****
C***** IMPLICIT REAL*8(A-H,O-Z)
C***** DIMENSION UCV(12,12)
C***** PI=3.141592653589793
C***** DI=DO-2.D0*T
C***** RM=(DO-T)/2.D0
C***** RGYRS=(DO**2+DI**2)/16.D0
C***** DJ=RGYRS*PI*(DO**2-DI**2)/4.D0
C***** V=1.D0-2.D0*T/DO
C***** IF(MSR.EQ.0)GO TO 10
C***** SDF=((7.D0+14.D0*PR+8.D0*PR**2)+(2.D0*V**2)/(1.D0+V**2)**2)
C***** 1*((10.D0+20.D0*PR+8.D0*PR**2))/(6.D0*(1.D0+PR)**2)
C***** GO TO 20
C***** SDF=0.D0
C***** XL=2.D0*(1.D0+PR)*SDF
C***** BFF=(1.65D0*RM**2)/(R*T)
C***** IF(BFF.LT.1.D0)BFF=1.D0
C***** C=DCOS(GAM/2.D0)
C***** S=DSIN(GAM/2.D0)
C***** CO=DCOS(GAM)
C***** SO=DSIN(GAM)
C***** CC=C**2
C***** SS=S**2
C***** CO2=CO**2
C***** SO2=SO**2
C***** P=(R**2*BFF+RGYRS*(1.D0-XL))/2.D0
C***** Q=(1.D0+PR-BFF)/2.D0

```



```

PS1=GAM+S0
PS2=GAM-S0
ZZ=R/(EY*DJ)
A=(RGYRS*XL*GAM+P*PS1)*ZZ
B=(RGYRS*XL*GAM+P*PS2)*ZZ
XM=(RGYRS*XL+R**2*(1.D0+PR))*GAM*ZZ
D=(2.D0*R*(1.D0+PR)*S)*ZZ
E=(BFF*GAM+Q*PS1)*ZZ
F=(BFF*GAM+Q*PS2)*ZZ
G=BFF*GAM*ZZ
H=-2.D0*R*BFF*S*ZZ
A1=A*CC+2.D0*R*C*H+SS*B+R**2*G
A2=-A*C*S+B*C*S-R*S*H
A3=-A*C*H-R*B*G
B2=A*SS+B*CC
C1=XM-2.D0*R*C*D+R**2*CC*E+R**2*SS*F
C2=-C*D+R*CC*E+R*SS*F
C3=S*D+R*S0*(F-E)/2.D0
D1=-C*D+R*CC*E+R*SS*F
D2=CC*E+SS*F
D3=S0*(F-E)/2.D0
E3=SS*E+CC*F
COM1=C0-1.D0
DO 30 I=1,12
DO 30 J=1,12
DO 30 J=0,OD0
UCV(I,J)=C0
UCV(I,1)=-S0
UCV(1,2)=R*COM1
UCV(1,6)=A1*C0+A3*R*(COM1)-A2*S0
UCV(1,7)=A1*C0+A3*R*(COM1)-A2*S0
UCV(1,8)=A2*C0+S*H*R*(COM1)-S0*S*H
UCV(1,12)=A3*C0+G*R*(COM1)-S0*S*H
UCV(2,1)=S0
UCV(2,2)=C0
UCV(2,6)=R*S0
UCV(2,7)=A1*S0+A3*R*S0+A2*CO
UCV(2,8)=A2*S0+S*H*R*S0+A2*CO
UCV(2,12)=A3*S0+G*R*S0+CO*S*H
UCV(3,3)=1.D0
UCV(3,4)=UCV(1,6)
UCV(3,5)=-R*S0
UCV(3,9)=C1+D1*R*(COM1)-C3*R*S0
UCV(3,10)=C2+D2*R*(COM1)-D3*R*S0
UCV(3,11)=C3+D3*R*(COM1)-E3*R*S0
UCV(4,4)=C0
UCV(4,5)=-S0
UCV(4,9)=C0*D1-S0*C3
UCV(4,10)=C0*D2-S0*D3

```

30

00010970
00010980
00010990
00011000
00011010
00011020
00011030
00011040
00011050
00011060
00011070
00011080
00011090
00011100
00011110
00011120
00011130
00011140
00011150
00011160
00011170
00011180
00011190
00011200
00011210
00011220
00011230
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00011370
00011380
00011390
00011400
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00011420
00011430
00011440


```

UCV(4,11)=C0*D3-S0*E3
UCV(5,4)=S0
UCV(5,5)=C0
UCV(5,9)=S0*D1+C0*C3
UCV(5,10)=S0*D2+C0*D3
UCV(5,11)=S0*D3+C0*E3
UCV(6,6)=1.D0
UCV(6,7)=A3
UCV(6,8)=S*H
UCV(6,12)=G
UCV(7,7)=C0
UCV(7,8)=-S0
UCV(8,7)=S0
UCV(8,8)=C0
UCV(9,9)=1.D0
UCV(10,9)=UCV(1,6)
UCV(10,10)=C0
UCV(10,11)=-S0
UCV(11,9)=R*S0
UCV(11,10)=S0
UCV(11,11)=C0
UCV(12,7)=UCV(1,6)
UCV(12,8)=-R*S0
UCV(12,12)=1.D0
50 RETURN
END

```

```

00011450
00011460
00011470
00011480
00011490
00011500
00011510
00011520
00011530
00011540
00011550
00011560
00011570
00011580
00011590
00011600
00011610
00011620
00011630
00011640
00011650
00011660
00011670
00011680
00011690
00011700

```

```

C***** SUBROUTINE DETER(N,A,D) *****
C***** SUBROUTINE DETER *****
C***** COMPUTES THE VALUE OF THE *****
C***** FREQUENCY DETERMINANT *****
C***** IMPLICIT REAL*8 (A-H), REAL*8 (O-Z) *****
C***** DIMENSION A(N,N) *****
10 DD=1.D0
10 DO 34 L=1,N
   KP=0
   Z=0.0
   DO 12 K=L,N
     IF(Z-DABS(A(K,L)))11,12,12
     Z=DABS(A(K,L))
   KP=K
11 CONTINUE
12 IF(L-KP)13,20,20
13 DO 14 J=L,N
   Z=A(L,J)

```

```

00011710
00011720
00011730
00011740
00011750
00011760
00011770
00011780
00011790
00011800
00011810
00011820
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00011840
00011850
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00011870
00011880
00011890
00011900

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00011910
00011920
00011930
00011940
00011950
00011960
00011970
00011980
00011990
00012000
00012010
00012020
00012030
00012040
00012050
00012060

```

```

14 A(L,J)=A(KP,J)
   A(KP,J)=Z
   DD=-DD
20 IF(L-N)31,40,40
31 LP1=L+1
   DO 34 K=LP1,N
   IF(A(K,L))32,34,32
32 RATIO=A(K,L)/A(L,L)
   DO 33 J=LP1,N
33 A(K,J)=A(K,J)-RATIO*A(L,J)
34 CONTINUE
40 DO 41 K=1,N
41 DD=DD*A(K,K)
   D=DD
   RETURN
   END

```

```

C***** SUBROUTINE DROOT(X,F,FCT,XLI,XRI,EPS,IEND,IER)*****
C***** SUBROUTINE DROOT
C***** ITERATES MUELLER'S METHOD TO
C***** FIND THE VALUE OF THE NATURAL
C***** FREQUENCY
C*****
C*****
C*****
C*****

```

```

00012070
00012080
00012090
00012100
00012110
00012120
00012130
00012140
00012150
00012160
00012170
00012180
00012190
00012200
00012210
00012220
00012230
00012240
00012250
00012260
00012270
00012280
00012290
00012300
00012310
00012320
00012330
00012340
00012350
00012360

```

```

DOUBLE PRECISION X,F,FCT,XLI,XRI,XL,XR,FL,FR,TOL,TOLF,A,DX,XM,FM
DOUBLE PRECISION DSIGN,DABS

```

```

C PREPARE ITERATION

```

```

IER=0
XL=XLI
XR=XRI
X=XL
TOL=X
F=FCT(TOL)
IF(F)1,16,1
1 X=XR
  TOL=X
  F=FCT(TOL)
  IF(F)2,16,2
2 FR=F
  IF(DSIGN(1.00,FL)+DSIGN(1.00,FR))25,3,25

```

```

C BASIC ASSUMPTION FL*FR LESS THAN 0 IS SATISFIED.
C GENERATE TOLERANCE FOR FUNCTION VALUES.

```

```

00012370
00012380
00012390
00012400
00012410
00012420
00012430
00012440
00012450
00012460
00012470
00012480
00012490
00012500
00012510
00012520
00012530
00012540
00012550
00012560
00012570
00012580
00012590
00012600
00012610
00012620
00012630
00012640
00012650
00012660

```



```

3 I=0
  TOLF=100.*EPS
C
C
C
4
  START ITERATION LOOP
  I=I+1
C
C
  START BISECTION LOOP
  DO 13 K=1,IEND
  X=.5D0*(XL+XR)
  TOL=X
  F=FACT(TOL)
  IF(F)5,16,5
5 IF(DSIGN(1.D0,F)+DSIGN(1.D0,FR))7,6,7
C
C
  INTERCHANGE XL AND XR IN ORDER TO GET THE SAME SIGN IN F AND FR
6 TOL=XL
  XL=XR
  XR=TOL
  TOL=FL
  FL=FR
  FR=TOL
7 TOL=F-FL
  A=F*TOL
  A=A+A
  IF(A-FR*(FR-FL))8,9,9
8 IF(I-IEND)17,17,9
9 XR=X
  FR=F
C
C
  TEST ON SATISFACTORY ACCURACY IN BISECTION LOOP
  TOL=EPS
  A=DABS(XR)
  IF(A-1.D0)11,11,10
10 TOL=TOL*A
11 IF(DABS(XR-XL)-TOL)12,12,13
12 IF(DABS(FR-FL)-TOLF)14,14,13
13 CONTINUE
  END OF BISECTION LOOP
C
C
C
C
  NO CONVERGENCE AFTER IEND ITERATION STEPS FOLLOWED BY IEND
  SUCCESSIVE STEPS OF BISECTION OR STEADILY INCREASING FUNCTION
  VALUES AT RIGHT BOUNDS. ERROR RETURN.
  IER=1
14 IF(DABS(FR)-DABS(FL))16,16,15
15 X=XL
  F=FL
16 RETURN

```

```

00012370
00012380
00012390
00012400
00012410
00012420
00012430
00012440
00012450
00012460
00012470
00012480
00012490
00012500
00012510
00012520
00012530
00012540
00012550
00012560
00012570
00012580
00012590
00012600
00012610
00012620
00012630
00012640
00012650
00012660
00012670
00012680
00012690
00012700
00012710
00012720
00012730
00012740
00012750
00012760
00012770
00012780
00012790
00012800
00012810
00012820
00012830
00012840

```



```

C      COMPUTATION OF ITERATED X-VALUE BY INVERSE PARABOLIC INTERPOLATION
C      00012850
C      00012860
C      00012870
C      00012880
C      00012890
C      00012900
C      00012910
C      00012920
C      00012930
C      00012940
C      00012950
C      00012960
C      00012970
C      00012980
C      00012990
C      00013000
C      00013010
C      00013020
C      00013030
C      00013040
C      00013050
C      00013060
C      00013070
C      00013080
C      00013090
C      00013100
C      00013110
C      00013120
C      00013130
C      00013140
C      00013150
C      00013160
C      00013170
C      00013180
C      00013190
C      00013200

C      00013210
C      00013220
C      00013230
C      00013240
C      00013250
C      00013260
C      00013270
C      00013280
C      00013290
C      00013300

C      COMPUTATION OF ITERATED X-VALUE BY INVERSE PARABOLIC INTERPOLATION
C      17 A=FR-F
C      DX=(X-XL)*FL*(1.DO+F*(A-TOL)/(A*(FR-FL)))/TOL
C      XM=X
C      FM=F
C      X=XL-DX
C      TOL=X
C      F=FC(TOL)
C      IF(F)18,16,18

C      TEST ON SATISFACTORY ACCURACY IN ITERATION LOOP
C      18 TOL=EPS
C      A=DABS(X)
C      IF(A-1.DO)20,20,19
C      19 TOL=TOL*A
C      20 IF(DABS(DX)-TOL)21,21,22
C      21 IF(DABS(F)-TOLF)16,16,22

C      PREPARATION OF NEXT BISECTION LOOP
C      22 IF(DSIGN(1.DO,F)+DSIGN(1.DO,FL))24,23,24
C      23 XR=X
C      FR=F
C      GO TO 4
C      24 XL=X
C      FL=F
C      XR=XM
C      FR=FM
C      GO TO 4
C      END OF ITERATION LOOP

C      ERROR RETURN IN CASE OF WRONG INPUT DATA
C      25 IER=2
C      RETURN
C      END

C      SUBROUTINE INVERT(A,N,D)
C      *****
C      SUBROUTINE INVERT
C      *****
C      INVERTS THE MATRIX FOR
C      *****
C      USE IN COMPUTING THE BRANCH
C      *****
C      POINT MATRIX
C      *****
C      *****
C      IMPLICIT REAL*8(A-H,O-Z)
C      *****
C      DIMENSION A(N,N),L(9),M(9)
C      *****
C      SEARCH FOR LARGEST ELEMENT
C      *****

```


00013310
00013320
00013330
00013340
00013350
00013360
00013370
00013380
00013390
00013400
00013410
00013420
00013430
00013440
00013450
00013460
00013470
00013480
00013490
00013500
00013510
00013520
00013530
00013540
00013550
00013560
00013570
00013580
00013590
00013600
00013610
00013620
00013630
00013640
00013650
00013660
00013670
00013680
00013690
00013700
00013710
00013720
00013730
00013740
00013750
00013760
00013770
00013780

```

D=1.0      K=1,N
DO 80 K=1,N
  L(K)=K
  M(K)=K
  BIGA=A(K,K)
DO 20 I=K,N
DO 20 J=K,N
10 IF(DABS(BIGA)-DABS(A(I,J))) 15,20,20
15 BIGA=A(I,J)
  M(K)=J
20 CONTINUE
  INTERCHANGE ROWS
  J=L(K)
  IF(L(K)-K) 35,35,25
25 DO 30 I=1,N
  HOLD=-A(K,I)
  A(K,I)=A(J,I)
  A(J,I)=HOLD
30 A(J,I)=HOLD
  INTERCHANGE COLUMNS
  I=M(K)
  IF(M(K)-K) 45,45,38
38 DO 40 J=1,N
  HOLD=-A(J,K)
  A(J,K)=A(J,I)
  A(J,I)=HOLD
40 A(J,I)=HOLD
  DIVIDE COLUMN BY MINUS PIVOT (VALUE OF PIVOT ELEMENT IS
  CONTAINED IN BIGA)
45 DO 55 I=1,N
  IF(I-K) 50,55,50
50 A(I,K)=A(I,K)/(-A(K,K))
55 CONTINUE
  REDUCE MATRIX
DO 65 I=1,N
DO 65 J=1,N
  IF(I-K) 60,65,60
  IF(J-K) 62,65,62
60 A(I,J)=A(I,K)*A(K,J)+A(I,J)
62 CONTINUE
  DIVIDE ROW BY PIVOT
DO 75 J=1,N
  IF(J-K) 70,75,70
70 A(K,J)=A(K,J)/A(K,K)
75 CONTINUE
  PRODUCT OF PIVOTS
D=D*A(K,K)
  REPLACE PIVOT BY RECIPROCAL
  A(K,K)=1.0D0/A(K,K)

```



```

00013790
00013800
00013810
00013820
00013830
00013840
00013850
00013860
00013870
00013880
00013890
00013900
00013910
00013920
00013930
00013940
00013950
00013960
00013970
00013980

```

```

C      80  CONTINUE
        FINAL ROW AND COLUMN INTERCHANGE
      K=N
100  K=(K-1)
      IF(K) 150,150,105
105  I=L(K)
      IF(I-K) 120,120,108
108  DO 110 J=1,N
      HOLD=A(J,K)
      A(J,K)=-A(J,I)
      A(J,I)=HOLD
110  J=M(K)
120  IF(J-K) 100,100,125
125  DO 130 I=1,N
      HOLD=A(K,I)
      A(K,I)=-A(J,I)
130  A(J,I)=HOLD
      GO TO 100
150  RETURN
      END

```

```

00013990
00014000
00014010
00014020
00014030
00014040
00014050
00014060
00014070
00014080
00014090
00014100
00014110
00014120
00014130

```

```

C*****
C***** SUBROUTINE MATMUL(A,B,C,L,M)
C*****
C***** SUBROUTINE MATMUL
C***** MULTIPLIES TWO MATRICES ONE OF
C***** WHICH IS SQUARE
C*****
C***** IMPLICIT REAL*8(A-H,O-Z)
C***** DIMENSION A(L,L),B(L,M),C(L,M)
C***** DO 10 J=1,M
C***** DO 10 I=1,L
C***** C(I,J)=0.0D0
C***** DO 10 K=1,L
C***** C(I,J)=C(I,J)+A(I,K)*B(K,J)
10  RETURN
      END

```

```

00014140
00014150
00014160
00014170
00014180
00014190
00014200
00014210
00014220

```

```

C*****
C***** SUBROUTINE POINT(RHO,D1,T,GAM,R,W,UPT)
C*****
C***** SUBROUTINE POINT
C***** CONSTRUCTS A POINT MATRIX FOR A
C***** CURVED PIPE
C*****
C***** IMPLICIT REAL*8(A-H,O-Z)
C***** DIMENSION UPT(12,12)
C***** GO=32.174

```



```

RHOM=RHO/GO
PI=3.141592653589793
G=GAM
G1=GAM/2.D0
G2=2.D0*GAM
D2=D1-(2.D0*T)
RBAR={D1+D2}/4.D0
AREA=PI*(D1**2-D2**2)/4.D0
RG=RD SIN(G1)/G1
XBAR=RG*DSIN(G1)
YBAR=RG*DCOS(G1)
DM=(RHOM*R*GAM*AREA)/20736.D0
A1=(RHOM*AREA*R*RBAR**2)/41472.D0
A2=(2.D0*RHOM*AREA*R*RBAR**3)/20736.D0
DIX=A1*(1.5D0*G+DSIN(G2)/4.D0)+A2*(.75D0*G-DSIN(G)+DSIN(G2)/8.D0)
DIY=A1*(1.5D0*G-DSIN(G2)/4.D0)+A2*(.25D0*G-DSIN(G2)/8.D0)
DIZ=A1*G+A2*(G-DSIN(G))
DO 10 I=1,12
DO 10 J=1,12
10 UPT(I,J)=0.000
DO 20 I=1,12
20 UPT(I,1)=1.D0
W2=W**2
UPT(7,1)=-DM*W2
UPT(7,6)=-DM*W2*YBAR
UPT(8,2)=UPT(7,1)
UPT(9,3)=UPT(7,1)
UPT(8,6)=-DM*W2*XBAR
UPT(9,4)=-UPT(7,6)
UPT(9,5)=-UPT(8,6)
UPT(10,4)=-DIX*W2
UPT(11,5)=-DIY*W2
UPT(12,6)=-DIZ*W2
RETURN
END

```

```

SUBROUTINE PRPLOT(X,Y,N,XMIN,XMAX,YMIN,YMAX,MODCUR)
C*****
C***** SUBROUTINE PRPLOT *****
C***** PLOTS A DOUBLE PRECISION FUNCTION *****
C***** LENGTHWISE ON TWO PAGES OF OUTPUT *****
C*****
C***** DIMENSION GRID(161, 81),YSCALE(5),XSCALE(17)
C*****
C***** DIMENSION Y(1),X(1)
C***** DIMENSION*2 GRID,BLANK,DASH,DOT,SLINE,YCHAR(4)/1H:,1H+,1H*,1HX/
C***** DATA DOT,BLANK,DASH,SLINE,Z4B40,Z4040,Z6D40,Z4F40/
C***** NDATA=N*2

```



```

C      KKZ=2
C      RY IS THE NUMBER OF LENGTHWISE LINES OF OUTPUT PLOT DESIRED
C
C      RY = 160.
C      IRYP1 = IFIX(RY) +1
C      IXSC = IRYP1/10 + 1
C      IERR=0
C
C      GRID IS THE MATRIX USED TO PLOT THE POINTS
C      CHECKING Y AND X POINTS AND PLOTTING THOSE OUT OF RANGE
C      AT THE MARGIN
C
C      DO 30 I=1,NDATA,KKZ
C      IF(Y(I)-YMAX) 205,205,220
C      Y (I)=YMAX
C      IERR=IERR+1
C      GO TO 210
C      IF(Y(I)-YMIN)203,210,210
C      Y (I)=YMIN
C      IERR=IERR+1
C      IF(X(I)-XMAX)215,215,212
C      X (I)=XMAX
C      IERR=IERR+1
C      GO TO 30
C      IF(X(I)-XMIN)217, 30,30
C      X (I)=XMIN
C      IERR=IERR+1
C      30 CONTINUE
C
C      PLOTTING Y AND X AXIS , IF NECESSARY
C
C      YRANGE=YMAX-YMIN
C      XRANGE=XMAX-XMIN
C      IF (XRANGE.NE.0.) GO TO 298
C      XMIN=YMIN
C      XMAX=YMAX
C      XRANGE=YRANGE
C      GO TO 299
C      IF (YRANGE.NE.0.) GO TO 299
C      YMIN=XMIN
C      YMAX=XMAX
C      YRANGE=XRANGE
C
C      BLANKING OUT MATRIX-(GRID)
C
00014690
00014700
00014710
00014720
00014730
00014740
00014750
00014760
00014770
00014780
00014790
00014800
00014810
00014820
00014830
00014840
00014850
00014860
00014870
00014880
00014890
00014900
00014910
00014920
00014930
00014940
00014950
00014960
00014970
00014980
00014990
00015000
00015010
00015020
00015030
00015040
00015050
00015060
00015070
00015080
00015090
00015100
00015110
00015120
00015130
00015140
00015150
00015160

```



```

299 DO 300 I=1,IRYP1
301 DO 301 JJ=1,81
300 GRID(I,JJ)=BLANK
CONTINUE
XTEST=XMAX*XMIN
YTEST=YMAX*YMIN
IF(YTEST)1,222,222
222 IF(XTEST)333,444,444
1 IXAXIS=80.*(YMIN)/YRANGE+1.5
40 DO 40 I=1,IRYP1
GRID(I,IXAXIS)=SLINE
GOTO 222
333 IYAXIS=RY *XMAX/XRANGE+1.5
DO 60 I=1,81
60 GRID(IYAXIS,I)=DASH

C
C
C PLACING POINTS IN THEIR PROPER GRID POSITIONS
444 IF(MODCUR.EQ.0.OR.MODCUR.EQ.1)JSET=0
JSET=JSET+1
IF(JSET.GT.4) JSET=1
DO 700 I=1,NDATA,KKZ
IPTV=RY *(X(I)-XMIN)/XRANGE+1.5
IPTX=80.*(Y(I)-YMIN)/YRANGE+1.5
IF(IPTY.GT.IRYP1.OR.IPTX.GT.81) GO TO 70
IF(IPTY.LE.0.OR.IPTX.LE.0)GO TO 70
GRID(IPTY,IPTX) = YCHAR(JSET)
GO TO 700
70 IERR=IERR+1
700 CONTINUE

C
C
C COMPUTE PROPER SCALE NUMBERS
IF(MODCUR.EQ.0.OR.MODCUR.EQ.1) GO TO 8000
IF(MODCUR.EQ.2) RETURN
GO TO 922
8000 YINCR=YRANGE/4.
XINCR=XRANGE/FLOAT(IXSC-1)
YSCALE(1)=YMAX
XSCALE(1)=XMIN
DO 80 I=2,5
80 YSCALE(I)=YSCALE(I-1)-YINCR
DO 81 I=2,IXSC
81 XSCALE(I)=XSCALE(I-1)+XINCR
C
C
C OUTPUT SECTION WITH GRAPH
IF(MODCUR.EQ.0.OR.MODCUR.EQ.3)GO TO 922

```



```

RETURN
17 FORMAT(12X,
1P, E10.3,4(10X,E10.3)/15X,2H**,8(10H+*****),3H+**)
922 WRITE(6,17) YSCALE(5),YSCALE(4),YSCALE(3),YSCALE(2),YSCALE(1)
II=1
I=0
DO 101 IK=1,IRYP1
IF(I)91,91,92
91 WRITE(6,18) XSCALE(II),(GRID(IK,IY),IY=1,81),XSCALE(II)
18 FORMAT(3X,1P
E10.3,2X,1H+,1X,81A1,1X,1H+,2X,E10.3)
II=II+1
GOTO 102
92 WRITE(6,19) (GRID(IK,IY),IY=1,81)
19 FORMAT(15X,1H*,1X,81A1,1X,1H*)
102 I=I+1
IF(I-10)101,103,103
103 I=0
101 CONTINUE
22 WRITE(6,22) YSCALE(5),YSCALE(4),YSCALE(3),YSCALE(2),YSCALE(1)
1P
FORMAT(15X,2H**,8(10H+*****),3H+**/
12X,E10.3,4(10X,E10.3))
1 IF(IERR) 1000,1000,1001
1001 WRITE(6,20) IERR
20 FORMAT(10X 'NUMBER OF POINTS OUT OF RANGE =' I4)
1000 RETURN
END

```

00015650
00015660
00015670
00015680
00015690
00015700
00015710
00015720
00015730
00015740
00015750
00015760
00015770
00015780
00015790
00015800
00015810
00015820
00015830
00015840
00015850
00015860
00015870
00015880
00015890
00015900
00015910

```

SUBROUTINE SORT(X,Y,M)
*****
***** SUBROUTINE SORT
***** SORTS THE ARRAY OF FREQUENCIES
***** AND VALUES OF THE FREQUENCY DE-
***** TERMINANT FOR PLOTTING ON THE PRINTER
*****
***** IMPLICIT REAL*8(A-H,O-Z)
***** DIMENSION X(M),Y(M)
***** MCONST=M-1
10 DO 30 I=1,MCONST
IF(X(I+1)-X(I))20,30,30
20 A=X(I)
B=Y(I)
X(I)=X(I+1)
Y(I)=Y(I+1)
X(I+1)=A
Y(I+1)=B
30 CONTINUE

```

00015920
00015930
00015940
00015950
00015960
00015970
00015980
00015990
00016000
00016010
00016020
00016030
00016040
00016050
00016060
00016070
00016080
00016090
00016100

00016110
00016120
00016130
00016140
00016150

```

MCONST=MCONST-1
IF(MCONST.EQ.1)GO TO 40
GO TO 10
40 RETURN
END

```

00016160
00016170
00016180
00016190
00016200
00016210
00016220
00016230
00016240
00016250
00016260
00016270
00016280
00016290
00016300
00016310
00016320
00016330
00016340
00016350
00016360
00016370

```

C*****
C***** SUBROUTINE STATEM(L,S)
C***** SUBROUTINE STATEM
C***** SUBSTRUCTS THE STATE MATRIX FOR
C***** FIXED, FREE, OR PINNED B.C.'S
C***** IMPLICIT REAL*8(A-H,O-Z)
C***** DIMENSION S(12,6)
C***** DO 10 J=1,6
C***** DO 10 I=1,12
C***** IF(I,J)=0.0D0
C***** IF(I-1)14,20,22
C***** 10 S(I,J)=1.0D0
C***** DO 15 I=1,6
C***** S(I,I)=1.0D0
C***** GO TO 26
C***** DO 21 I=1,6
C***** S(I+6,I)=1.0D0
C***** GO TO 26
C***** DO 22 I=1,6
C***** S(I+3,I)=1.0D0
C***** 26 RETURN
C***** END

```

00016380
00016390
00016400
00016410
00016420
00016430
00016440
00016450
00016460
00016470
00016480
00016490
00016500
00016510
00016520
00016530
00016540

```

C***** SUBROUTINE STRMAT(W,DL,RHO,EY,PR,D1,T,MSR,MSI,UST)
C***** SUBROUTINE STRMAT
C***** SUBSTRUCTS A TRANSFER MATRIX FOR
C***** A STRAIGHT SECTION OF PIPE
C***** IMPLICIT REAL*8(A-H,O-Z)
C***** DIMENSION UST(12,12)
C***** GO=32.174D0
C***** PI=3.141592653589793
C***** G=EY/(2.0D0*(1.0D0+PR))
C***** D2=D1-(2.0D0*T)
C***** RX=DSQRT((D1**2+D2**2)/8.0D0)
C***** DJ=(PI*(D1**4-D2**4))/64.0D0
C***** DJT=2.0D0*DJ
C***** AREA=(PI*(D1**2-D2**2))/4.0D0

```



```

UU=(RHO*AREA)/(20736.D0*G0)
GAM=DL*W*DSQRT(UU/(AREA*G))
AL=DL*W*DSQRT(UU/(AREA*EY))
A=(DL**2)/(EY*DJ)
B4=(UU*(W**2)*(DL**4))/(EY*DJ)
IF(MSR.EQ.0) GO TO 10
V=1.D0-2.D0*T/D1
SDF=((7.D0+14.D0*PR+8.D0*PR**2)+((2.D0*V**2)/(1.D0+V**2)**2)
1 *{10.D0+20.D0*PR+8.D0*PR**2))/(6.D0*(1.D0+PR)**2)
SIG=(UU*(W**2)*(DL**2))/(SDF*G*AREA)
TAU=((DL**2)*UU*(RY**2)*(W**2))/(EY*DJ)
GO TO 20
10 SIG=0.0D0
TAU=0.0D0
20 XL2=DSQRT(DSQRT(B4+.25D0*((SIG-TAU)**2))- .5D0*(SIG+TAU))
CL=1.0D0/(XL1**2+XL2**2)
CO=CL*(XL2**2*DCOSH(XL1)+XL1**2*DCOS(XL2))
C1=CL*((XL2**2/XL1)*DSINH(XL1)+(XL1**2/XL2)*DSIN(XL2))
C2=CL*(DCOSH(XL1)-DCOS(XL2))
C3=CL*((1.0D0/XL1)*DSINH(XL1)-(1.0D0/XL2)*DSIN(XL2))
DO 30 I=1,12
DO 30 J=1,12
UST(I,J)=0.0D0
UST(I,1)=DCOS(AL)
UST(1,1)=DSIN(AL)/(EY*AREA*AL)
UST(1,7)=DL*SIG*C2)
UST(2,2)=CO*(SIG+TAU)*C3)
UST(2,6)=DL*(C1-(SIG+TAU)*C3)
UST(2,8)=-(A*DL)/B4)*(-SIG*C1+(B4+SIG**2)*C3)
UST(2,12)=A*C2
UST(3,3)=UST(2,2)
UST(3,5)=-UST(2,6)
UST(3,9)=UST(2,8)
UST(3,11)=-UST(2,12)
UST(4,4)=DCOS(GAM)
UST(4,10)=(DL*DSIN(GAM))/(G*DJT*GAM)
UST(5,3)=-B4*C3)/DL
UST(5,5)=CO-(TAU*C2)
UST(5,9)=UST(2,12)
UST(5,11)=(A/DL)*(C1-TAU*C3)
UST(6,2)=-UST(5,3)
UST(6,6)=UST(5,5)
UST(6,8)=-A*C2
UST(6,12)=UST(5,11)
UST(7,1)=-UU*DL*W**2*DSIN(AL))/AL
UST(7,7)=UST(1,1)
UST(7,9)=UST(1,1)
UST(8,2)=-B4*C3)/(A*DL)
UST(8,6)=-B4*C2)/A

```



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00017190
00017200
00017210
00017220
00017230
00017240
00017250
00017260
00017270
00017280
00017290
00017300
00017310
00017320
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00017340
00017350
00017360
00017370

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      UST(8,8)=UST(2,2)
      UST(8,12)=UST(5,3)
      UST(9,3)=UST(8,2)
      UST(9,5)=-UST(8,6)
      UST(9,9)=UST(2,2)
      UST(9,11)=UST(6,2)
      UST(10,4)=-DL*UU*RX**2**W**2*DSIN(GAM))/GAM
      UST(10,10)=DCOS(GAM)
      UST(11,3)=UST(8,6)
      UST(11,5)=(DL/A)*(-TAU*C1+(B4+TAU**2)*C3)
      UST(11,9)=UST(2,6)
      UST(11,11)=UST(5,5)
      UST(12,2)=UST(9,5)
      UST(12,6)=UST(11,5)
      UST(12,8)=-UST(11,9)
      UST(12,12)=UST(11,11)
      IF(MST.EQ.0)GO TO 41
      UST(3,3)=0.D0
      UST(3,5)=1.D0
      UST(3,9)=0.D0
      UST(3,11)=1.D0
      UST(5,3)=1.D0
      UST(5,5)=0.D0
      UST(5,9)=1.D0
      UST(5,11)=0.D0
      UST(9,3)=0.D0
      UST(9,5)=1.D0
      UST(9,9)=0.D0
      UST(9,11)=1.D0
      UST(11,3)=1.D0
      UST(11,5)=0.D0
      UST(11,9)=1.D0
      UST(11,11)=0.D0
      RETURN
41 END

```

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00017380
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```

C***** SUBROUTINE UVEC(A,B,UX,UY,UZ,GAM)
C***** SUBROUTINE UVEC
C***** COMPUTES A SET OF ORTHOGONAL UNIT
C***** VECTORS FROM TWO PIPE SECTION VECTORS
C***** IMPLICIT REAL*8(A-H,O-Z)
C***** DIMENSION A(3),B(3),C(3),UX(3),UY(3),UZ(3)
C***** AABS=DSQRT(A(1)**2+A(2)**2+A(3)**2)
C***** DO 10 N=1,3
C***** 10 UX(N)=A(N)/AABS
C*****
C***** SUBROUTINE UVEC(A,B,UX,UY,UZ,GAM)
C***** SUBROUTINE UVEC
C***** COMPUTES A SET OF ORTHOGONAL UNIT
C***** VECTORS FROM TWO PIPE SECTION VECTORS
C***** IMPLICIT REAL*8(A-H,O-Z)
C***** DIMENSION A(3),B(3),C(3),UX(3),UY(3),UZ(3)
C***** AABS=DSQRT(A(1)**2+A(2)**2+A(3)**2)
C***** DO 10 N=1,3
C***** 10 UX(N)=A(N)/AABS
C*****

```



```

C(1)=(B(2)*A(3)-A(2)*B(3))
C(2)=(B(3)*A(1)-A(3)*B(1))
C(3)=(B(1)*A(2)-A(1)*B(2))
CABS=DSQRT(C(1)**2+C(2)**2+C(3)**2)
DO 20 N=1,3
  UZ(N)=C(N)/CABS
  UY(1)=(UZ(2)*UX(3)-UX(2)*UZ(3))
  UY(2)=(UZ(3)*UX(1)-UX(3)*UZ(1))
  UY(3)=(UZ(1)*UX(2)-UX(1)*UZ(2))
  BABS=DSQRT(B(1)**2+B(2)**2+B(3)**2)
  BOOT A=B(1)*A(1)+B(2)*A(2)+B(3)*A(3)
  GAM=DARCOS(BOOT A/(BABS*AABS))
RETURN
END

```

```

C *** FUNCTION FRDET(W) ***
C *** FUNCTION FRDET ***
C *** COMPUTES THE TRANSFER MATRIX FOR ***
C *** EACH SECTION OF PIPE AND EACH ***
C *** COORDINATE TRANSFORMATION, MUL- ***
C *** TIPLIES THEM TOGETHER AND COMPUTES ***
C *** THE FREQUENCY DETERMINANT ***
C *** IMPLICIT REAL*8(A-H,O-Z) ***

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00017840
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204 I=1,12
204 J=1,12
204 U(I,J)=U(I,J)
204 GO TO 210
206 CALL COORDX(ALPHA,12,CTX12)
206 CALL MATMUL(CTX12,U,U,12,6)
210 GAMS=GAM/DFLOAT(NSS)
212 CALL POINT(RHO,D1,T,GAMS,R,W,UA)
220 CALL CURMAT(EY,PR,D1,R,T,GAMS,MSR,UB)
220 CALL MATMUL(UA,U1,U2,12,6)
220 CALL MATMUL(UB,U2,U1,12,6)
223 NSS=NSS-1
223 IF(NSS.GT.0)GO TO 220
223 DO 230 I=1,12
223 DO 230 J=1,6
230 U(I,J)=U1(I,J)
230 N=N+1
230 GO TO 100

C      FORM TRANSFER MATRICES FOR CORNER SECTIONS AND MULTIPLY TIMES
C      SYSTEM TRANSFER MATRIX
300 EY=AA(N,2)
300 PR=AA(N,3)
300 RHO=AA(N,4)
300 D1=AA(N,5)
300 T=AA(N,6)
300 DL=AA(N,7)
300 ALPHA=AA(N,8)
300 GAM=AA(N,9)
300 IF(ALPHA.GT.0.01D0)GO TO 306
304 I=1,12
304 J=1,12
304 U(I,J)=U(I,J)
304 GO TO 310
306 CALL COORDX(ALPHA,12,CTX12)
306 CALL MATMUL(CTX12,U,U,12,6)
310 CALL STRMAT(W,DL,RHO,EY,PR,D1,T,MSR,MST,UA)
310 CALL MATMUL(UA,U1,U2,12,6)
310 CALL COORDZ(GAM,12,CTZ12)
310 CALL MATMUL(CTZ12,U2,U,12,6)
310 N=N+1
310 GO TO 100

C      FORM TRANSFER MATRIX FOR THE STRAIGHT SECTION PRECEEDING A
C      BRANCH AND MULTIPLY TIMES THE SYSTEM TRANSFER MATRIX
400 NK=0

```



```

IF(I SEC.EQ.5)GO TO 410
NK=1
410 EY=AA(N,2)
PR=AA(N,3)
RHO=AA(N,4)
D1=AA(N,5)
T=AA(N,6)
DL=AA(N,7)
CALL STRMAT(W,DL,RHO,EY,PR,D1,T,MSR,MST,UA)
CALL MATMUL(UA,U,U3,12,6)
N=N+1
420 JBC=NB(K)
K=K+1

C
C SET UP INITIAL STATE MATRIX FOR THE END OF THE BRANCH
C
C CALL STATEM(JBC,U)
C
C USE MAIN MEMBER ROUTINES TO COMPUTE THE BRANCH TRANSFER MATRIX
C
C
GO TO 100
430 EY=AA(N,2)
PR=AA(N,3)
RHO=AA(N,4)
D1=AA(N,5)
T=AA(N,6)
DL=AA(N,7)
ALPHA1=AA(N,8)
GAM=AA(N,9)
ALPHA=AA(N,10)
CALL STRMAT(W,DL,RHO,EY,PR,D1,T,MSR,MST,UA)
CALL MATMUL(UA,U,U1,12,6)

C
C COMPUTE THE BRANCH POINT MATRIX AND MULTIPLY TIMES THE SYSTEM
C TRANSFER MATRIX
C
DO 440 I=1,6
DO 440 J=1,6
R1(I,J)=U1(I,J)
R2(I,J)=U1(I+6,J) DET)
440 CALL INVERT(R1,6,6)
442 CALL COORDX(ALPHA,6,G2)
CALL COORDX(ALPHA1,6,G1)
CALL COORDZ(GAM,6,G3)
DO 450 I=1,6
DO 450 J=1,6
G3T(I,J)=G3(J,I)
G2T(I,J)=G2(J,I)

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450  G1T(I,J)=G1(J,I)
      CALL MATMUL(G3T,G2T,G,6,6)
      CALL MATMUL(G1T,G,G2T,6,6)
      CALL MATMUL(R1,G2T,G3T,6,6)
      CALL MATMUL(R2,G3T,G2T,6,6)
      CALL MATMUL(G1,G2T,G3T,6,6)
      CALL MATMUL(G3,G3T,G2T,6,6)
457  CALL MATMUL(G2,G2T,G,6,6)
      DO 460 I=1,12
      DO 460 J=1,12
      PT(I,J)=0.DO
460  DO 465 I=1,12
465  PT(I,I)=1.DO
      DO 470 I=1,6
      DO 470 J=1,6
470  PT(I+6,J)=G(I,J)
471  CALL MATMUL(PT,U3,U,12,6)
C
C   CHECK TO SEE IF JUNCTION CONTAINS A DUAL-BRANCH
C
      N=N+1
      IF(NK.EQ.0)GO TO 100
      NK=NK-1
      DO 480 I=1,12
      DO 480 J=1,6
480  U3(I,J)=U(I,J)
      GO TO 420
C
C   FORM TRANSFER MATRIX FOR END SECTION(IF IF LENGTH IS NONZERO) AND
C   MULTIPLY TIMES SYSTEM TRANSFER MATRIX
C
500  EY=AA(N,2)
      PR=AA(N,3)
      RHO=AA(N,4)
      D1=AA(N,5)
      T=AA(N,6)
      DL=AA(N,7)
      W1=W
      IF(DL.GT.0.DO)GO TO 510
      DO 505 I=1,6
      DO 505 J=1,6
505  U1(I,J)=U(I,J)
506  GO TO 520
510  CALL STRMAT(W,DL,RHO,EY,PR,D1,T,MSR,MST,UA)
      CALL MATMUL(UA,U,12,6)
C
C   SET UP FREQUENCY DETERMINANT ACCORDING TO RIGHT END BOUNDARY CON-
C   DITION
C

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00020000
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00020090

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C      520  MBC=NB(2)
      522  IF(MBC-1)522,530,536
      525  DO 525 I=1,6
      525  DO 525 J=1,6
      525  DD(I,J)=UI(I+6,J)
      530  GO TO 540
      530  DO 535 I=1,6
      535  DO 535 J=1,6
      535  DD(I,J)=UI(I,J)
      536  GO TO 540
      536  DO 537 I=1,3
      537  DO 537 J=1,6
      537  DD(I,J)=UI(I,J)
      538  DO 538 I=4,6
      538  DO 538 J=1,6
      538  DD(I,J)=UI(I+6,J)

C      C      COMPUTE FREQUENCY DETERMINANT
C
C      540  CALL DETER(6,DD,D)
      560  FRDET=D
      560  RETURN
      560  END

```


APPENDIX F
PROGRAM ACCURACY AND INTEGRITY ANALYSIS

1. General Remarks

This appendix tabulates by mode frequency (radians/second) the results of the accuracy and integrity analyses of various piping systems from simple to complex.

The comparison frequency values for the simple straight section systems were obtained by solution of the differential equations for longitudinal, torsional and flexural vibrations with application of the appropriate boundary conditions. These solutions were evaluated on the digital computer with accuracy to at least as many places as listed in the tabulation. Evaluation of the comparison solution and VIBREL solution were carried out using a value of 32.174 FT/SEC^2 for the gravitational constant and a value of π to 15 significant figures.

Comparison values for the curved pipe analysis were obtained from three sources: Fink [5], Kim [6], and Ojalvo [8]. Those extracted directly from [5] and [6] were compiled using a lumped mass model and digital computation. From graphs and formulas (based on the theory of clamped ring segments) appearing in [8], the other natural frequency values for the curved pipe were calculated.

Several comparison frequencies for a single branch system were obtained from [6].

Shear deflection and rotary inertia were neglected except where noted.

2. SYSTEMS ANALYSIS

a. STRAIGHT PIPE WITH VARIOUS END CONDITIONS



END CONDITIONS:
FIXED-FIXED

SYSTEM PARAMETERS:	
DIAMETER (IN)	2.0
WALL THICKNESS (IN)	.125
LENGTH (IN)	200.
MODULUS OF ELASTICITY (PSI)	30 x 10 ⁶
POISSON'S RATIO	.25
WEIGHT DENSITY (LB/FT ³)	490.

COMPARISON RESULTS:

MODE	ANALYTICAL	VIBREL	% DIFFERENCE
1ST FLEXURAL	75.10461420	75.10461590	.00000
2ND FLEXURAL	207.02876110	207.02876110	.00000
3RD FLEXURAL	405.85914940	405.85914210	.00000
4TH FLEXURAL	670.90578980	670.90577600	.00000
5TH FLEXUPAL	1002.21750380	1002.21750410	.00000
6TH FLEXURAL	1399.79137910	1399.79140510	.00000
7TH FLEXURAL	1863.62757630	1863.62799710	.00002
1ST TORSIONAL	2007.8325054	2007.8275305	.00025
8TH FLEXURAL	2393.7260868	2393.7157896	.00043
9TH FLEXURAL	2990.0869112	2990.3653337	.00931
1ST LONGITUDINAL	3174.6619386	3174.6560265	.00019

DUAL ANALYSIS RESULTS:

MODE	STARTING AT LEFT END	STARTING AT RIGHT END	% DIFFERENCE
1	75.1046142	75.1046142	.00000
2	207.0287611	207.0287611	.00000
3	405.8591494	405.8591494	.00000
4	670.9057898	670.9057898	.00000
5	1002.2175038	1002.2175038	.00000
6	1399.7913791	1399.7913791	.00000
7	1863.6275763	1863.6275763	.00000



END CONDITIONS:
FIXED-FREE

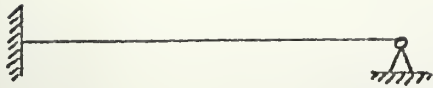
SYSTEM PARAMETERS:
SAME AS (1)

COMPARISON RESULTS:

MODE	ANALYTICAL	VIBREL	% DIFFERENCE
1	11.8028657	11.8028696	.00003
2	73.9673211	73.9673092	.00002
3	207.1106480	207.1106397	.00000
4	405.8541957	405.8541953	.00000
5	670.9060651	670.9060651	.00000
6	1399.7913799	1399.7913885	.00000

DUAL ANALYSIS RESULTS:

MODE	STARTING AT LEFT END	STARTING AT RIGHT END	% DIFFERENCE
1	11.8028696	11.8028696	.00000
2	73.9673092	73.9673092	.00000
3	207.1106397	207.1106397	.00000
4	405.8541953	405.8541953	.00000
5	670.9060651	670.9060651	.00000
6	1399.7913885	1399.7913885	.00000



SYSTEM PARAMETERS:

SAME AS (1)

END CONDITIONS:

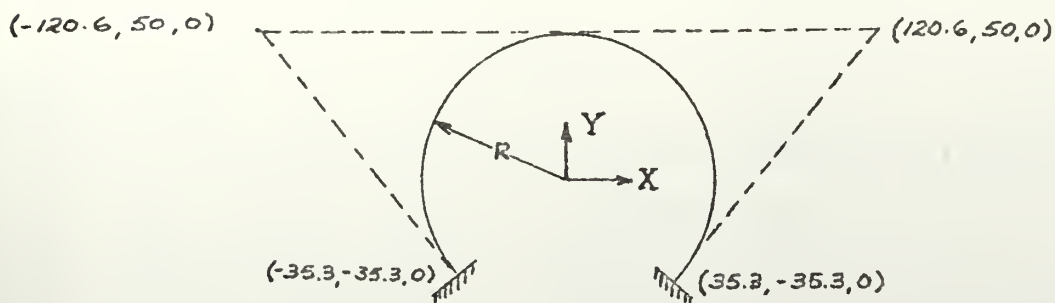
FIXED-PINNED

COMPARISON RESULTS:

MODE	ANALYTICAL	VIBREL	% DIFFERENCE
1	51.7571905	51.7571773	.00003
2	167.7264474	167.7264180	.00002
3	349.9478448	349.9478460	.00000
4	598.4315201	598.4315226	.00000
5	913.1775124	913.17753170	.00000
6	1003.9162520	1003.9164796	.00002

DUAL ANALYSIS RESULTS:

MODE	STARTING AT LEFT END	STARTING AT RIGHT END	% DIFFERENCE
1	51.7571773	51.7571773	.00000
2	167.7264180	167.7264180	.00000
3	349.9478460	349.9478460	.00000
4	598.4315226	598.4315226	.00000
5	913.17753170	913.17753170	.00000
6	1003.9164796	1003.9164796	.00000



END CONDITIONS: FIXED-FIXED

SYSTEM PARAMETERS:

PIPE DIAMETER (IN)	2.0
WALL THICKNESS (IN)	.125
INCLUDED ANGLE OF ARC (DEG)	270
MODULUS OF ELASTICITY (PSI)	30×10^6
POISSON'S RATIO	.25
WEIGHT DENSITY (LB/FT ³)	556

COMPARISON RESULTS

a. MODE 1

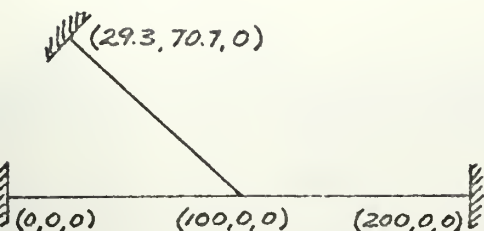
SOURCE	FREQUENCY	% VARIATION WITH VIBREL VALUE
VIBREL	37.2234058	
OJALVO [8]	38.5930983	3.68
FINK [5]	38.8773520	4.44
KIM [6]	35.3639740	-5.00

b. MODE 2

SOURCE	FREQUENCY	% VARIATION WITH VIBREL VALUE
VIBREL	72.1914615	
OJALVO	69.9962862	-3.04
FINK	73.3297307	1.58
KIM	66.6993490	-7.61

c. BRANCHED SYSTEMS

(1) SINGLE BRANCH SYSTEM



SYSTEM PARAMETERS (ALL 3 SFCTIONS):

DIAMETER (IN)	2.0
WALL THICKNESS (IN)	.125
LENGTH (IN)	100.
MODULUS OF ELASTICITY (PSI)	30 x 10 ⁶
POISSON'S RATIO	.25
WEIGHT DENSITY (LB/FT ³)	556.

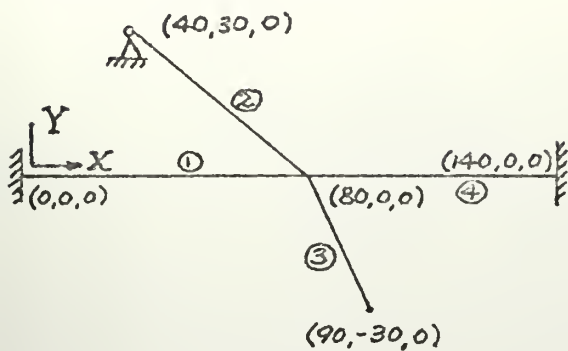
COMPARISON RESULTS:

MODE	KIM [6]	VIBREL	% DIFFERENCE
2	194.25470558	194.2888499	.01758
5	278.61078241	278.6390155	.01013
8	628.59556355	628.9417782	.05508

DUAL ANALYSIS RESULTS:

MODE	STARTING AT LEFT END	STARTING AT RIGHT END	% DIFFERENCE
1	67.6456627	67.6456627	.00000
2	194.2888474	194.2888474	.00000
3	198.0817333	198.0817333	.00000
4	242.0773169	242.0773169	.00000
5	278.6390155	278.6390155	.00000
6	281.9864461	281.9864461	.00000
7	370.7949928	370.7949928	.00000
8	628.9417782	628.9417782	.00000
9	634.7478785	634.7478786	.00000
10	689.7373111	689.7373111	.00000

(2) DUAL BRANCH SYSTEM



END CONDITIONS:

MAIN MEMBER - FIXED-FIXED
BRANCH #1 - PINNED
BRANCH #2 - FREE

SYSTEM PARAMETERS:

PARAMETER	SECTION 1	SECTION 2	SECTION 3	SECTION 4
DIAMETER (IN)	2.0	2.0	2.0	2.0
WALL THICKNESS (IN)	.125	.125	.125	.125
LENGTH (IN)	80.0	50.0	31.6	60.0
MOD. OF ELAST. (PSI)	30×10^6	30×10^6	30×10^6	30×10^6
POISSON'S RATIO	.30	.30	.30	.30
WT. DENSITY (LB/FT ³)	490.0	490.0	490.0	490.0

DUAL ANALYSIS RESULTS*:

MODE	STARTING AT LEFT END	STARTING AT RIGHT END	% DIFFERENCE
1	110.5015347	110.5015347	.00000
2	276.6928874	276.6928874	.00000
3	314.3597336	314.3597336	.00000
4	417.0897739	417.0897739	.00000
5	463.5288123	463.5288123	.00000
6	675.6526889	675.6526889	.00000

* SHEAR DEFLECTION AND ROTARY INERTIA ARE CONSIDERED IN THIS ANALYSIS.

† COORDINATES SHOWN IN DIAGRAM ARE IN INCHES.

d. COMPLEX SYSTEM

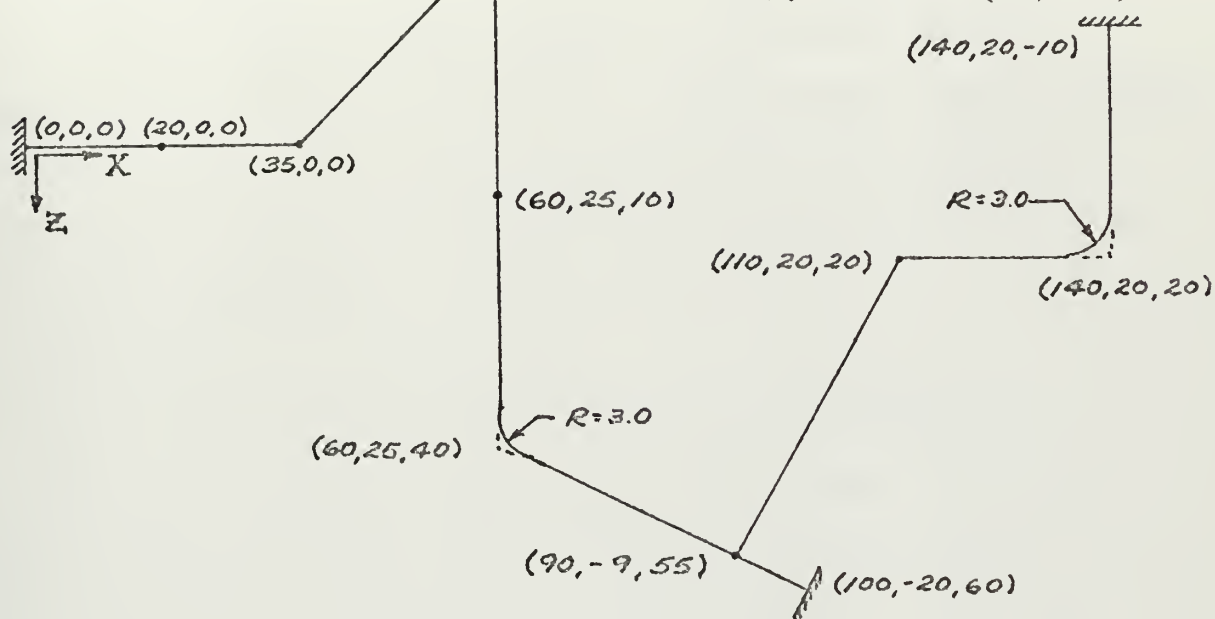
END CONDITIONS:

MAIN MEMBER - FIXED-FIXED
BRANCH - FIXED

(60,25,-25)

SYSTEM PARAMETERS
(ALL SECTIONS):

DIAMETER (IN) 2.25
WALL THICKNESS (IN) .035
ELASTIC MODULUS (PSI) 30×10^6
POISSON'S RATIO .30
WT. DENSITY (LB/FT³) 491.5



DUAL ANALYSIS RESULTS:

MODE	STARTING AT LEFT END	STARTING AT RIGHT END	% DIFFERENCE
1	73.0350449	73.7545503	.985
2	102.7404907	102.9266533	.181
3	142.5065200	146.3168530	2.674
4	201.7233189	201.3564078	.182
5	239.1786608	239.1641100	.006
6	327.8210841	321.1763486	2.027
7	331.8165320	337.3328726	1.662
8	364.5803428	364.1833110	.109
9	428.5622381	429.1841008	.145
10	520.1734071	521.2514969	.207
11	542.4026997	544.3290021	.355
12	565.0762600	564.3679439	.125
13	574.0701684	571.0709491	.522
14	648.8240390	651.2831922	.379
15	754.8640271	759.4399716	.606
16	1052.0436077	1052.3129250	.026
17	1210.9202599	1210.7321375	.016
18	1250.1288524	1247.3298693	.224

APPENDIX G

SAMPLE PROBLEM

1. Piping System

The piping used in this example is identical to the complex system shown in Appendix F. It includes several corners and bends and has one branch junction point. The boundary conditions are fixed for the two ends of the main member and the remote branch end.

2. Data Cards

Shown in Fig. G.1 is the arrangement of data cards as they would appear following the program deck. Note that Card #1 specifies one system to be analyzed, and Card #2 asks for four mode frequencies in the analysis. Since the curved subsection override is not utilized, the program will go ahead and determine the number of curved pipe subdivisions according to the length-to-diameter ratio. Shear deflection and rotary inertia are not included in this problem analysis.


```

*****
*                                *
*          PROGRAM  VIBREL      * C.O.RUOOLF
*                                *
*****

```

PROGRAM VIBREL COMPUTES THE NATURAL FREQUENCIES OF VIBRATION
OF ANY RANDOMLY ARRANGED 3-DIMENSIONAL PIPING SYSTEM
PIPING HANGERS AND PROJECTIVE COMPLEXITIES ARE NOT INCLUDED

PROBLEM # 1

THE EFFECTS OF SHEAR DEFLECTION AND ROTARY INERTIA ARE NOT CONSIDERED IN THIS PROBLEM

INPUT DATA:

IO	X COORD	Y COORD	Z COORD	RAD OF CURV	ELASTIC MODULUS	POISSONS RATIO	O.O.	WALL THICK	WT/VOL	LEFT BC CODE	RIGHT BC CODE
1	0.0	0.0	0.0								
3	20.0	0.0	0.0	0.0	0.3000	0.30	2.25	0.0350	491.5	1	1
5	35.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0		
5	60.0	25.0	-25.0	0.0	0.0	0.0	0.0	0.0	0.0		
3	60.0	25.0	10.0	0.0	0.0	0.0	0.0	0.0	0.0		
4	60.0	25.0	40.0	3.0	0.0	0.0	0.0	0.0	0.0		
6	90.0	-9.0	55.0	0.0	0.0	0.0	0.0	0.0	0.0		
2	140.0	20.0	-10.0							1	
4	140.0	20.0	20.0	3.0	0.0	0.0	0.0	0.0	0.0		
5	110.0	20.0	20.0	0.0	0.0	0.0	0.0	0.0	0.0		
B	90.0	-9.0	55.0	0.0	0.0	0.0	0.0	0.0	0.0		
B	100.0	-20.0	60.0	0.0	0.0	0.0	0.0	0.0	0.0		

PROPERTIES AND GEOMETRY:

SECT NR	IO	ELASTIC MODULUS	POISSONS RATIO	WT/VOL	O.O.	WALL THICK	LENGTH OR RAD OF CURV	ALPHA1	GAMMA	NR OF SUBSEC OR ALPHA
1	1.0	0.3000	0.30	491.5	2.25	0.0350	20.000	0.0	0.0	0.0
2	3.0	0.3000	0.30	491.5	2.25	0.0350	15.000	-2.356	0.955	0.0
3	3.0	0.3000	0.30	491.5	2.25	0.0350	43.301	2.094	2.186	0.0
4	1.0	0.3000	0.30	491.5	2.25	0.0350	35.000	0.0	0.0	0.0
5	1.0	0.3000	0.30	491.5	2.25	0.0350	27.833	0.0	0.0	0.0
6	2.0	0.3000	0.30	491.5	2.25	0.0350	3.000	1.508	1.251	2.000
7	5.0	0.3000	0.30	491.5	2.25	0.0350	45.592	0.0	0.0	0.0
B	1.0	0.3000	0.30	491.5	2.25	0.0350	27.000	0.0	0.0	0.0
9	2.0	0.3000	0.30	491.5	2.25	0.0350	3.000	0.0	1.571	2.000
10	3.0	0.3000	0.30	491.5	2.25	0.0350	27.000	2.450	1.156	0.0
11	7.0	0.3000	0.30	491.5	2.25	0.0350	49.659	-0.372	-1.177	-0.847
12	4.0	0.3000	0.30	491.5	2.25	0.0350	15.684	0.0	0.0	0.0

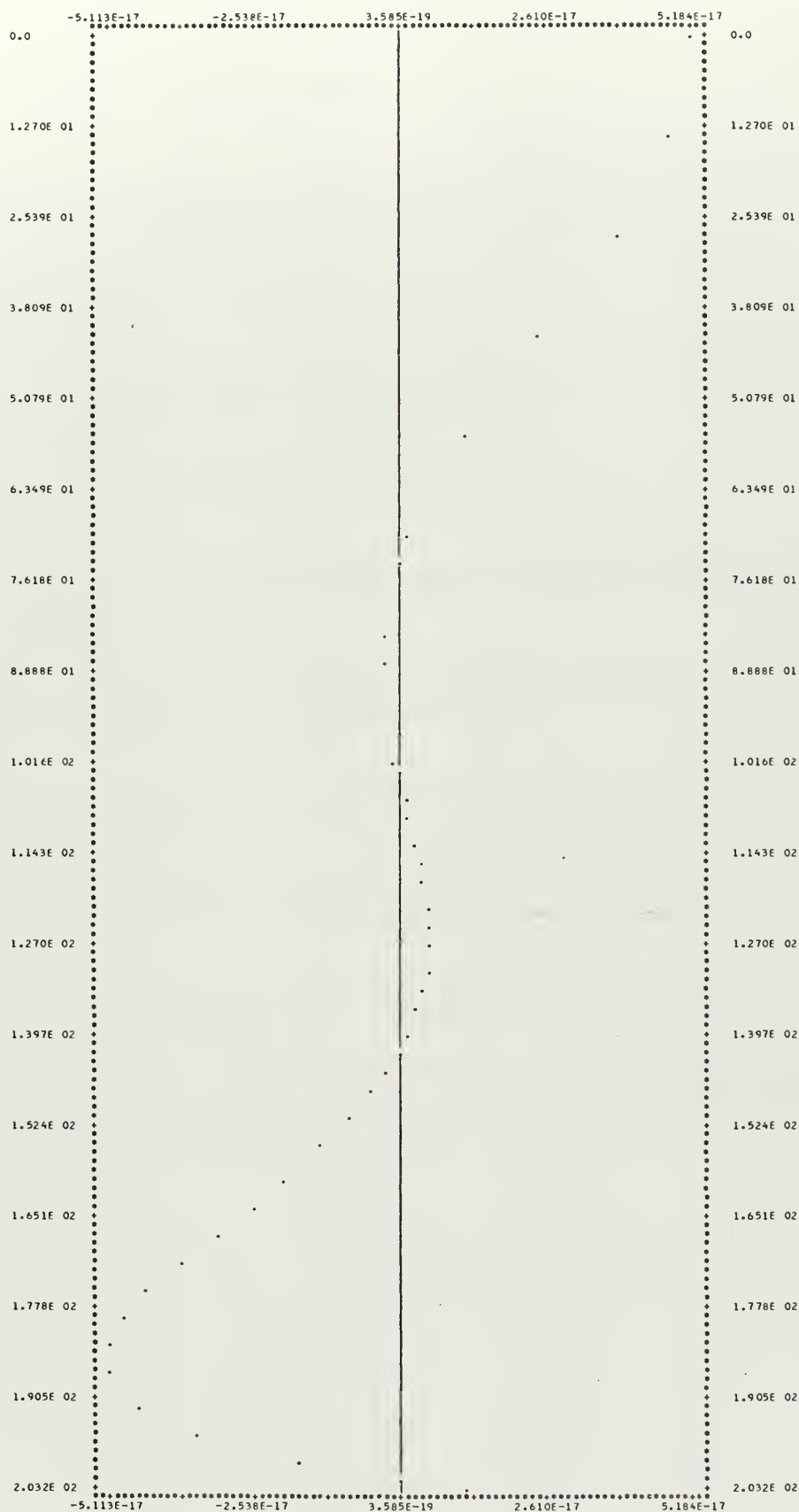
BOUNDARY CONDITION CODES:

LEFT B.C. CODE= 1
RIGHT B.C. CODE= 1
BRANCH # 1 B.C. CODE= 1

PIPING SYSTEM NATURAL FREQUENCIES:

MODE	FREQUENCY(RAO/SEC)	FREQUENCY DETERMINANT
1	73.0350449	0.122780-22
2	102.7404907	-1.369030-21
3	142.5065200	0.859130-21
4	201.7233189	0.524240-22

GRAPH OF THE VALUE OF FREQUENCY DETERMINANT VS. FREQUENCY



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13. ABSTRACT

This thesis develops the theory for and presents a digital computer program capable of determining the natural frequencies of a three-dimensional piping system having arbitrary configuration. The analysis uses the method of transfer matrices. Piping hangers, loops, and complex branches (branches emanating from branches) have not been included in the analysis. A distributed mass model is used for straight pipe sections, while mass is lumped for curved sections. Inclusion of shear deflection and rotary inertia is optional.

Several piping configurations are analyzed using the program; the results are compared with analytical solutions or values from the literature to demonstrate the accuracy and integrity of the program.

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Piping						
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